# SOME ISSUES AND REMARKS ON LAGRANGIAN STOCHASTIC MODELS FOR SINGLE-PHASE TURBULENCE AND FIBRE SUSPENSIONS

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Jean-Pierre Minier On Lagrangian stochastic approaches for fibre suspensions

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### Plan

#### General guidelines: Modelling approach and actual models

- Choice of a modelling approach for single-phase reactive flows
- Choice of a modelling approach for polydispersed two-phase flows

### 2 The framework of present Lagrangian two-phase flow models

- The (one-particle) PDF formalism: from SDEs to Mean fields
- An reminder on the physical contents

### Consistency issues in the fluid limit case

- Correspondence with Reynolds-stress modelling
  - The spurious issue of spurious drifts
  - Various forms of the Langevin equations
- Consistency issues in the hybrid "Euler/Lagrange" formulation

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Choice of a modelling approach for single-phase reactive flows Choice of a modelling approach for polydispersed two-phase flows

## Do all roads lead to Rome?

- In turbulence, the issue of "modelling" is to reduce the number of degree of freedom and to come up with a "relevant" *contracted statistical description* (for single-phase and two-phase flow turbulence)
- From microscopic to macroscopic descriptions
  - (i) Microscopic level: DNS for the carrier fluid and "exact" fiber tracking

#### (ii) PDF closures (Lagrangian stochastic models)

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(iii) Macroscopic or continuum modelling: two-fluid approaches

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 (iv) Macroscopic or continuum modelling: single-fluid approaches with modified rheological properties

General guidelines: Modelling approach and actual models

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## Along the PDF road



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## Distinguishing the frame from the painting

- It is important to distinguish between the modelling framework and actual modelling proposals
- The choice of a modelling framework is an essential step, resulting from the **physical analysis** of *the situations to be addressed* (what flows are we trying to simulate?) and from **the choice** of *a set of precise statistical quantities* (what specific details are we trying to capture?)
- The "game" is then to tailor the "best model" within a chosen framework
- For example, is it relevant to criticise  $k \epsilon$  models because they cannot predict the fluid kinetic energy turbulence spectrum?
- Or is it relevant to criticise present (one-particle) Langevin models for two-phase flows because they contain no-length information?
- A double hierarchy and a double choice in the PDF world one or N-fiber PDF?/What variables for each fiber (fiber state-vector)?

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## The issue of the closure of the chemical source term (1)

- In turbulent reactive flows, one may look for reduced information on some one-point moments on scalars φ = (T, c<sub>1</sub>, c<sub>2</sub>,..., c<sub>α</sub>)
- For instance, the mean scalar equation has the form

$$\frac{\partial \langle \phi_l \rangle}{\partial t} + \langle U_k \rangle \frac{\partial \langle \phi_l \rangle}{\partial x_k} = -\frac{\partial \langle u_k \phi_l' \rangle}{\partial x_k} + \langle S_l(\phi) \rangle$$

- The moments equations involve typical terms such as the mean chemical source term  $\langle S_l(\phi) \rangle$  and the scalar turbulent flux  $\langle u_k \phi'_l \rangle$
- These terms cannot be easily closed when only moments are known

$$\langle S(\phi) \rangle \stackrel{?}{=} \mathcal{F}(\langle \phi \rangle, \dots, \langle (\phi)^m \rangle)$$

• However, these terms are closed if the one-point pdf is known

$$\langle S(\phi) 
angle(t, \mathbf{x}) = \int S(\psi) \, p(t, \mathbf{x}; \psi) \, d\psi$$
  
 $\langle U_k \phi 
angle = \int V_k \, \psi \, p(t, \mathbf{x}; V_k, \psi) \, dV_k \, d\psi$ 

## The issue of the closure of the chemical source term (2)

- if we wish to be able to treat the chemical source term,  $\langle S_l(\phi) \rangle$ , and turbulent fluxes (convective effects),  $\langle u_k \phi_l' \rangle$ , without approximation, then, at least, the one-point pdf  $p(t, \mathbf{x}; \mathbf{V}, \psi)$  must be known
- But do we need more detailed information such as the two-point pdf, *p*(*t*, **x**<sub>1</sub>, **x**<sub>2</sub>; **V**<sub>1</sub>, ψ<sub>1</sub>, **V**<sub>2</sub>, ψ<sub>2</sub>) ?
- For turbulent reactive flows, **if we choose to describe only** one-point moments, then one-point pdfs are necessary but.. nothing more
- In that case, one-point pdfs closures appear as the best "tailor choice" with respect to **this given objective**
- Once this framework is chosen, it is important to know what is inside

$$\langle \boldsymbol{c}^{m} \rangle(t, \mathbf{x}) = \int \varphi^{m} \underbrace{\boldsymbol{p}(t, \mathbf{x}; \varphi)}_{known} d\varphi \quad \forall m$$

• and what is outside, such as the concentration spatial correlation

$$\langle c(t,\mathbf{x}_1)c(t,\mathbf{x}_2)\rangle = \int \varphi_1 \varphi_2 \underbrace{p(t,\mathbf{x}_1,\mathbf{x}_2;\varphi_1,\varphi_2)}_{unknown} d\varphi_1 d\varphi_2$$

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## Dispersed two-phase Flow Modelling: point particles

• "exact equations" : Navier-Stokes eqs. and forces on a particle

$$\begin{cases} \frac{d\mathbf{x}_{\rho}}{dt} = \mathbf{U}_{\rho} \\ \frac{d\mathbf{U}_{\rho}}{dt} = \frac{\mathbf{U}_{s} - \mathbf{U}_{\rho}}{\tau_{\rho}} + \frac{\rho_{\rho} - \rho_{f}}{\rho_{\rho}} \mathbf{g} + \frac{\rho_{f}}{\rho_{\rho}} \left(\frac{d\mathbf{U}_{s}}{dt}\right) + \frac{1}{2} \frac{\rho_{f}}{\rho_{\rho}} \left(\frac{d\mathbf{U}_{s}}{dt} - \frac{d\mathbf{U}_{\rho}}{dt}\right) \\ \tau_{\rho} = \frac{\rho_{\rho}}{\rho_{f}} \frac{4d}{3C_{D}|\mathbf{U}_{s} - \mathbf{U}_{\rho}|} \quad C_{D} = \frac{24}{Re} [1 + 0.15Re^{0.687}] \quad Re = \frac{d|\mathbf{U}_{s} - \mathbf{U}_{\rho}|}{\nu}$$

• case of 'heavy' particles  $\rho_p \gg \rho_f$ 

$$\left\{ egin{array}{l} \displaystyle rac{d \mathbf{x}_{
ho}}{dt} = \mathbf{U}_{
ho} \ \displaystyle rac{d \mathbf{U}_{
ho}}{dt} = rac{\mathbf{U}_{s} - \mathbf{U}_{
ho}}{ au_{
ho}} + \mathbf{g} \end{array} 
ight.$$

• key word: turbulence  $\Rightarrow$  statistical approach

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## But how "exactly" should we describe each fiber?

• The rigid-fiber approach

$$rac{d \mathbf{U}_{
ho}}{dt} \sim \mathbf{K} \left( \mathbf{U}_{s} - \mathbf{U}_{
ho} 
ight) \ rac{d \omega_{
ho}}{dt} \sim \mathcal{F} ( \omega_{
ho}, S^{f}_{ij}, \Omega^{f}_{ij} )$$



- $\Rightarrow$  need of a local (one-point) fluid information on  $U_i$  and  $\partial U_i / \partial x_j$
- The chain-rod approach for flexible fibers: N-connected rods



 $\Rightarrow$  need of a spatial (multi-point) fluid information on  $U_i$  and  $\partial U_i / \partial x_j$ 

• A new and intermediate description with a reduced state-vector?

define a few variables:  $I, \theta, \ldots$ 



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### Modelling constraints

- limitations of the macroscopic road
  - detailed information is needed
  - it is too difficult to close directly at the macroscopic level through constitutive relations between macroscopic variables.
- need of different physical descriptions
  - the important physical phenomena are treated without approximation,
  - enough information is available to handle correctly issues of complex physics (combustion, polydispersed particles),
  - the resulting numerical model is tractable for non-homogeneous flows,
  - the model can be coupled to other approaches, either more fundamental or applied descriptions.

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## Let's go stochastic

• (one) central issue: turbulent dispersion

$$\frac{d\mathbf{U}_{p}}{dt} = \frac{\mathbf{U}_{s} - \mathbf{U}_{p}}{\tau_{p}} + \mathbf{g}$$

 $\mathbf{U}_{s}(t) = \mathbf{U}[t, \mathbf{x}_{p}(t)]$  is the *instantaneous* fluid velocity 'seen' by the particle along its trajectory as it moves across the flow.

 limited available information on U<sub>s</sub> (ex : first two moments provided by the turbulence model)

#### $\Rightarrow$ stochastic modelling

- compromise is sought between physical accuracy, information content and tractability to (or possible generalisation to) general flows
   ⇒ one-particle pdf
- key issue: proper choice of the variables retained in the state vector and those that are eliminated from the PDF description

The (one-particle) PDF formalism: from SDEs to Mean fields An reminder on the physical contents

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## PDF approach to dispersed two-phase flow

 The fluid 'seen' is included in the state vector Z = (x<sub>p</sub>, U<sub>p</sub>, U<sub>s</sub>) which is modelled by a general diffusion process

$$\begin{cases} d\mathbf{x}_{p} = \mathbf{U}_{p} dt \\ d\mathbf{U}_{p} = \mathbf{A}_{p}(t, \mathbf{Z}), dt \qquad (= \frac{\mathbf{U}_{s} - \mathbf{U}_{p}}{\tau_{p}} dt + \mathbf{g} dt) \\ d\mathbf{U}_{s} = \mathbf{A}_{s} (t, \mathbf{Z}, \langle \mathcal{F}[\mathbf{Z}] \rangle) dt + \mathbf{B}_{s} (t, \mathbf{Z}, \langle \mathcal{G}[\mathbf{Z}] \rangle) d\mathbf{W} \end{cases}$$

- particle tracking approach = PDF approach
   Monte Carlo simulation of the pdf p<sub>L</sub>(t; y<sub>ρ</sub>, V<sub>ρ</sub>, V<sub>s</sub>)
- hierarchy of descriptions (marginal pdfs)

$$p_L(t; \mathbf{y}_{\rho}, \mathbf{V}_{\rho}) = \int p_L(t; \mathbf{y}_{\rho}, \mathbf{V}_{\rho}, \mathbf{V}_{s}) d\mathbf{V}_s$$
$$p_L(t; \mathbf{y}_{\rho}) = \int \int p_L(t; \mathbf{y}_{\rho}, \mathbf{V}_{\rho}, \mathbf{V}_s) d\mathbf{V}_s d\mathbf{V}_{\rho}$$

• The PDF formalism for the fluid case is retrieved by taking  $au_
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# The PDF formalism (1)

• Lagrangian model for a large number N of stochastic particles

$$\begin{cases} d\mathbf{x}_{\rho} = \mathbf{U}_{\rho} dt, \\ d\mathbf{U}_{\rho} = \mathbf{A}_{\rho}(t, \mathbf{Z}), dt \\ d\mathbf{U}_{s} = \mathbf{A}_{s} (t, \mathbf{Z}, \langle \mathcal{F}[\mathbf{Z}] \rangle) dt + \mathbf{B}_{s} (t, \mathbf{Z}, \langle \mathcal{G}[\mathbf{Z}] \rangle) d\mathbf{W} \end{cases}$$

• Discrete Lagrangian and Eulerian Mass Density Functions (MDFs)

$$\begin{aligned} F_{\rho}^{L}(t; \mathbf{y}, \mathbf{V}_{\rho}, \mathbf{V}_{s}) &= M_{\rho} \, \rho_{L}(t; \mathbf{y}, \mathbf{V}_{\rho}, \mathbf{V}_{s}) \\ F_{\rho}^{E}(t, \mathbf{x}; \mathbf{V}_{\rho}, \mathbf{V}_{s}) &= F_{\rho}^{L}(t; \mathbf{y} = \mathbf{x}, \mathbf{V}_{\rho}, \mathbf{V}_{s}) \\ &= \int F_{\rho}^{L}(t; \mathbf{y}, \mathbf{V}_{\rho}, \mathbf{V}_{s}) \delta(\mathbf{y} - \mathbf{x}) \, d\mathbf{y} \end{aligned}$$

• The general definition of an average quantity is

$$\alpha_{\rho}(t,\mathbf{x})\,\rho_{\rho}\langle H_{\rho}\rangle(t,\mathbf{x}) = \int H(\mathbf{V}_{\rho},\psi_{\rho})F_{\rho}^{E}(t,\mathbf{x};\mathbf{V}_{\rho},\mathbf{V}_{s})\,d\mathbf{V}_{\rho}\,\mathbf{V}_{s}$$

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# The PDF formalism (2)

• Discrete Lagrangian and Eulerian MDFs

$$\begin{aligned} F_{\rho,N}^{L}(t;\mathbf{y},\mathbf{V}_{\rho},\mathbf{V}_{s}) &= \sum_{i=1}^{N} m_{\rho}^{(i)} \delta(\mathbf{y}-\mathbf{x}_{\rho}^{(i)}) \otimes \delta(\mathbf{V}_{\rho}-\mathbf{U}_{\rho}^{(i)}) \otimes \delta(\mathbf{V}_{s}-\mathbf{U}_{s}^{(i)}) \\ F_{\rho,N}^{E}(t,\mathbf{x};\mathbf{V}_{\rho},\mathbf{V}_{s}) &= F_{\rho,N}^{L}(t;\mathbf{y}=\mathbf{x},\mathbf{V}_{\rho},\mathbf{V}_{s}) \end{aligned}$$

• In the discrete formulation, in a given cell with  $N_x^p$  particles

$$\langle H_{\rho} \rangle \simeq H_{\rho,N} = rac{\sum_{i=1}^{N_x^{\rho}} m_{\rho}^i H(\mathbf{U}_{\rho}^i(t),\mathbf{U}_{s}^i(t))}{\sum_{i=1}^{N_x^{\rho}} m_{\rho}^i}.$$

The Eulerian MDF satisfies the same Fokker-Planck as p<sub>L</sub>

$$\begin{aligned} \frac{\partial F_{\rho}^{E}}{\partial t} + V_{\rho,i} \frac{\partial F_{\rho}^{E}}{\partial x_{i}} &= -\frac{\partial}{\partial V_{\rho,i}} (A_{\rho,i} F_{\rho}^{E}) \\ &- \frac{\partial}{\partial V_{s,i}} (A_{s,i} F_{\rho}^{E}) + \frac{1}{2} \frac{\partial^{2}}{\partial V_{s,i} \partial V_{s,j}} \left( (B_{s} B_{s}^{T})_{ij} F_{\rho}^{E} \right). \end{aligned}$$

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## **Resulting mean field equations**

Mean continuity and mean momentum equations

$$\begin{aligned} &\frac{\partial}{\partial t}(\alpha_{p}\rho_{p}) + \frac{\partial}{\partial x_{i}}(\alpha_{p}\rho_{p}\langle U_{p,i}\rangle) = \mathbf{0}.\\ &\alpha_{p}\rho_{p}\frac{d}{dt}\langle U_{p,i}\rangle = -\frac{\partial}{\partial x_{j}}(\alpha_{p}\rho_{p}\langle u_{p,i}u_{p,j}\rangle) + \alpha_{p}\rho_{p}\langle A_{p,i}\rangle. \end{aligned}$$

• The (rather intricate) second-order equations

$$\begin{split} \alpha_{p}\rho_{p}\frac{d}{dt}\langle u_{p,i}u_{p,j}\rangle &= -\frac{\partial}{\partial x_{k}}(\alpha_{p}\rho_{p}\langle u_{p,i}u_{p,j}u_{p,k}\rangle) - \alpha_{p}\rho_{p}\langle u_{p,i}u_{p,k}\rangle\frac{\partial\langle U_{p,j}\rangle}{\partial x_{k}} \\ &- \alpha_{p}\rho_{p}\langle u_{p,j}u_{p,k}\rangle\frac{\partial\langle U_{p,i}\rangle}{\partial x_{k}} + \alpha_{p}\rho_{p}\langle A_{p,i}v_{p,j} + A_{p,j}v_{p,i}\rangle, \\ \alpha_{p}\rho_{p}\frac{d}{dt}\langle u_{s,i}u_{p,j}\rangle &= -\frac{\partial}{\partial x_{k}}(\alpha_{p}\rho_{p}\langle u_{s,i}u_{p,j}u_{p,k}\rangle) - \alpha_{p}\rho_{p}\langle u_{s,i}u_{p,k}\rangle\frac{\partial\langle U_{p,j}\rangle}{\partial x_{k}} \\ &- \alpha_{p}\rho_{p}\langle u_{p,j}u_{p,k}\rangle\frac{\partial\langle U_{s,i}\rangle}{\partial x_{k}} + \alpha_{p}\rho_{p}\langle A_{s,i}v_{p,j}\rangle + \alpha_{p}\rho_{p}\langle A_{p,j}v_{s,i}\rangle, \end{split}$$

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# Lagrangian stochastic modelling (fluid case): key physical notions

- $\bullet~\textbf{Z}=(\textbf{x},\textbf{U})$  is treated as a diffusion process
- At  $Re \gg 1$ , the acceleration **a** is seen as a fast-variable and eliminated
- Kolmogorov theory yields a coarse-grained description  $au_\eta \ll dt \ll T$
- decomposition of a fluid particle acceleration

$$\mathbf{a} = -\frac{1}{\rho} \frac{\partial \langle \boldsymbol{P} \rangle}{\partial \mathbf{x}} + \underline{\mathbf{G}} \left( \mathbf{U} - \langle \mathbf{U} \rangle \right) + \boldsymbol{\gamma}.$$

- Kolmogorov theory at  $Re \gg 1$ :  $\langle (\gamma)^2 \rangle \sim \langle a^2 \rangle \Rightarrow \langle (\gamma)^2 \rangle \simeq \frac{\langle \epsilon \rangle}{\tau_n}$
- fast-variable elimination:  $\langle (\gamma)^2 \rangle \times \tau_\eta \xrightarrow{R_{\theta} \to \infty} D \simeq \langle \epsilon \rangle$
- slaving principle:  $\gamma$  is a function of the local value of the slow mode  ${\bf x}$

$$\gamma \, dt pprox \sqrt{C_0 \, \langle \epsilon 
angle ({f x})} \, d{f W}$$

This slaving principle has direct relevance to the mean-field equations

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# Lagrangian stochastic modelling (fluid case): the resulting form

one-point description (at present)

$$\begin{cases} dx_i^+ = U_i^+ dt \\ dU_i^+ = \left( -\frac{1}{\rho} \frac{\partial P}{\partial x_i} + \nu \Delta U_i \right)^+ dt \\ d\phi_l^+ = (\Gamma \Delta \phi_l)^+ dt + S_l(\phi^+) dt \\ \psi \end{cases}$$

$$\begin{cases} dx_i = U_i \, dt \\ dU_i = -\frac{1}{\rho} \frac{\partial \langle P \rangle}{\partial x_i} \, dt + D_i \, dt + \sqrt{C_0 \langle \epsilon \rangle} dW_i \\ d\phi_l = A_{\phi_l} \, dt + S_l(\phi) \, dt \end{cases} \\ D_i = G_{ij} \left(U_j - \langle U_j \rangle\right) = -\left(\frac{1}{2} + \frac{3}{4}C_0\right) \frac{\langle \epsilon \rangle}{k} (U_i - \langle U_i \rangle) + G_{ij}^a \left(U_j - \langle U_j \rangle\right) \\ A_{\phi_l} = -\frac{\phi_l - \langle \phi_l \rangle}{\tau_{\phi}} \quad \tau_{\phi} = \frac{2k}{C_{\phi} \langle \epsilon \rangle} \end{cases}$$

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## Mean velocity Equations

mass conservation and mean momentum equation (high-Reynolds case)

$$\begin{cases} \frac{\partial \langle U_i \rangle}{\partial x_i} = 0\\ \frac{\partial \langle U_i \rangle}{\partial t} + \langle U_j \rangle \frac{\partial \langle U_i \rangle}{\partial x_j} + \frac{\partial \langle u_i u_j \rangle}{\partial x_j} = -\frac{1}{\rho} \frac{\partial \langle P \rangle}{\partial x_i} \end{cases}$$

second-order equation (high-Reynolds case)

$$\frac{\partial \langle u_i u_j \rangle}{\partial t} + \langle U_k \rangle \frac{\partial \langle u_i u_j \rangle}{\partial x_k} + \frac{\partial \langle u_i u_j u_k \rangle}{\partial x_k} = - \langle u_i u_k \rangle \frac{\partial \langle U_j \rangle}{\partial x_k} - \langle u_j u_k \rangle \frac{\partial \langle U_i \rangle}{\partial x_k} \langle u_i D_j \rangle + \langle u_j D_i \rangle + C_0 \langle \epsilon \rangle \delta_{ij}$$

$$\frac{\partial \langle u_i u_j \rangle}{\partial t} + \langle U_k \rangle \frac{\partial \langle u_i u_j \rangle}{\partial x_k} = -\frac{\partial \langle u_i u_j u_k \rangle}{\partial x_k} - \langle u_i u_k \rangle \frac{\partial \langle U_j \rangle}{\partial x_k} - \langle u_j u_k \rangle \frac{\partial \langle U_i \rangle}{\partial x_k} + G^a_{ik} \langle u_j u_k \rangle + G^a_{jk} \langle u_i u_k \rangle - (1 + \frac{3}{2}C_0) \frac{\langle \epsilon \rangle}{k} \left( \langle u_i u_j \rangle - \frac{2}{3} k \delta_{ij} \right) - \frac{2}{3} \langle \epsilon \rangle \delta_{ij}$$

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### Mean scalar Equations

mean scalar equation

$$\frac{\partial \langle \phi_l \rangle}{\partial t} + \langle U_k \rangle \frac{\partial \langle \phi_l \rangle}{\partial x_k} = -\frac{\partial \langle u_k \phi_l' \rangle}{\partial x_k} + \langle S_l(\phi) \rangle$$

second-order scalar transport equation

$$\begin{aligned} \frac{\partial \langle u_{i}\phi_{i}^{'}\rangle}{\partial t} + \langle u_{k}\rangle \frac{\partial \langle u_{i}\phi_{i}^{'}\rangle}{\partial x_{k}} + \frac{\partial \langle u_{i}u_{k}\phi_{i}^{'}\rangle}{\partial x_{k}} &= -\langle u_{i}u_{k}\rangle \frac{\partial \langle \phi_{l}\rangle}{\partial x_{k}} - \langle u_{k}\phi_{l}^{'}\rangle \frac{\partial \langle U_{i}\rangle}{\partial x_{k}} \\ &+ \langle u_{i}S_{l}(\phi)\rangle - \left(G_{ik} - \frac{1}{2}C_{\phi}\frac{\langle \epsilon\rangle}{k}\delta_{ik}\right) \langle u_{k}\phi_{l}^{'}\rangle\end{aligned}$$

- All convective and source terms are in closed form
- No *instantaneous* field but **mean fields**  $\langle \mathbf{U} \rangle (t, \mathbf{x}), \langle u_i u_j \rangle (t, \mathbf{x}), \langle \phi \rangle (t, \mathbf{x}).$
- Closures with white-noise terms in the particle stochastic equations correspond to local closures (in space) in the mean-field equations

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## White-noise terms and local mean-field closures (1)

• The white-noise term in the particle stochastic equation

$$dU_{i} = -\frac{1}{\rho} \frac{\partial \langle P \rangle}{\partial x_{i}} dt + D_{i} dt + \underbrace{\sqrt{C_{0} \langle \epsilon(\mathbf{x}) \rangle} dW_{i}}_{last term}$$

#### which has a zero correlation timescale correspond to

$$\frac{\partial \langle u_{i} u_{j} \rangle}{\partial t} + \langle U_{k} \rangle \frac{\partial \langle u_{i} u_{j} \rangle}{\partial x_{k}} + \frac{\partial \langle u_{i} u_{j} u_{k} \rangle}{\partial x_{k}} = -\langle u_{i} u_{k} \rangle \frac{\partial \langle U_{j} \rangle}{\partial x_{k}} - \langle u_{j} u_{k} \rangle \frac{\partial \langle U_{i} \rangle}{\partial x_{k}}$$
$$\langle u_{i} D_{j} \rangle + \langle u_{j} D_{i} \rangle + \underbrace{C_{0} \langle \epsilon \rangle (\mathbf{x})}_{local \ term} \delta_{ij}$$

 The memory-less term in the particle velocity equation induce a "transport-less" or local term in the corresponding equation (which appears always as the level of the work performed by these "forces")

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### White-noise terms and local mean-field closures (2)

If the fast-acceleration terms is now regularised (with a "coloured noise")

$$dU_i = -rac{1}{
ho}rac{\partial\langle P
angle}{\partial x_i} dt + D_i dt + \gamma_i dt$$

which is a defined process with a non-zero integral timescale, then

$$\frac{\partial \langle u_{i}u_{j} \rangle}{\partial t} + \langle U_{k} \rangle \frac{\partial \langle u_{i}u_{j} \rangle}{\partial x_{k}} + \frac{\partial \langle u_{i}u_{j}u_{k} \rangle}{\partial x_{k}} = -\langle u_{i}u_{k} \rangle \frac{\partial \langle U_{j} \rangle}{\partial x_{k}} - \langle u_{j}u_{k} \rangle \frac{\partial \langle U_{i} \rangle}{\partial x_{k}}$$
$$\langle u_{i}D_{j} \rangle + \langle u_{j}D_{i} \rangle + \underbrace{\langle u_{i}\gamma_{j} \rangle + \langle u_{j}\gamma_{i} \rangle}_{non-local term}$$

 For example, if γ is also a Langevin model dγ<sub>i</sub> = - γ<sub>i</sub>/τ dt + K<sub>i</sub> dW<sub>i</sub> then (u<sub>i</sub>γ<sub>i</sub>) is the solution of a **transport equation**: non-locality (τ > 0)

$$\begin{cases} \frac{\partial \langle u_i \gamma_i \rangle}{\partial t} + \langle U_k \rangle \frac{\partial \langle u_i \gamma_i \rangle}{\partial x_k} + \frac{\partial \langle u_k u_i \gamma_i \rangle}{\partial x_k} = \langle D_i \gamma_i \rangle - \frac{\langle u_i \gamma_i \rangle}{\tau} + \langle (\gamma_i)^2 \rangle \\ \frac{\partial \langle (\gamma_i)^2 \rangle}{\partial t} + \langle U_k \rangle \frac{\partial \langle (\gamma_i)^2 \rangle}{\partial x_k} + \frac{\partial \langle u_k (\gamma_i)^2 \rangle}{\partial x_k} = -2 \frac{\langle (\gamma_i)^2 \rangle}{\tau} + \underbrace{\mathcal{K}_i^2}_{\text{local term}} \end{cases}$$

The (one-particle) PDF formalism: from SDEs to Mean fields An reminder on the physical contents

## Acceleration and the derivative of the velocity auto-correlation (1)

• Lagrangian velocity auto-correlation  $R_L(t,s) = \frac{\langle U(t)U(t+s) \rangle}{\sqrt{\langle U^2(t) \rangle \langle U^2(t+s) \rangle}}$ 



In stationary homogeneous turbulence, the Lagrangian velocity auto-correlation,  $R_L(s)$  depends only on the time lag, *s*, and is an even  $C^2$ -function. Thus, at the origin,  $R'_L(s = 0) = 0$ 

• With a Langevin model for particle velocity written simply as

$$dU(t) = -\frac{U(t)}{T}\,dt + K\,dW_t$$

then the auto-correlation is an exponential function

$$R(s) = \exp\left(-\frac{s}{T}\right) \Longrightarrow R'(s=0) = -\frac{1}{T} \neq 0$$

• The form of *R*<sub>L</sub> is a consequence of a Langevin model but **not** its input

# Acceleration and the derivative of the velocity auto-correlation (2)

- Is there a defect of Lagrangian models? Is it even a result or a surprise?
- The derivative of the velocity auto-correlation is governed by acceleration

$$R_{L}^{'}(s=0) = \langle U(t)\left(\frac{dU(t)}{dt}\right) \rangle$$

- Yet, the *basic starting idea* behind the Langevin model was to *skip over details of the acceleration* and describe it as a *fast-variable process*
- The non-zero slope is not a result but reflects the modelling choice
- If acceleration is important, it should be treated as a non-infinite process

$$\begin{cases} dU(t) = -\frac{U(t)}{T} dt + \gamma dt \\ d\gamma = \frac{\gamma}{\tau} dt + K_a dW \end{cases} \Rightarrow \begin{cases} R_L(s) = \frac{1}{1 - \tau/T} \left[ e^{-s/\tau} - \frac{\tau}{T} e^{-s/\tau} \right] \\ \lim_{\tau \to 0} R'_L(s = 0) \neq \left( \lim_{\tau \to 0} R_L \right)' (s = 0) \end{cases}$$

• Auto-correlations result from the choice of a stochastic model

• Numerically, Stratonovich calculus can be used (cf. Lectures 4-5)

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## Kolmogorov hypothesis for the fluid seen

Kolmogorov theory in the inertial range

$$\langle (d\mathbf{U}_f)^2 \rangle = C_0 \langle \epsilon \rangle dt$$

Langevin equation for fluid particle velocities

$$dU_{f,i} = -\frac{1}{\rho_f} \frac{\partial \langle P \rangle}{\partial x_i} dt - \frac{U_{f,i} - \langle U_{f,i} \rangle}{T_L} dt + \sqrt{C_0 \langle \epsilon \rangle} dW_i$$

• construction in two steps : step 1 (particle inertia) and step 2 (CTE)

- present models neglect step 1  $\rightarrow$  statistics of the fluid seen (**U**<sub>s</sub>) differ from the statistics of fluid particle velocities due to mean drifts
- Fluid seen  $d\mathbf{U}_s = \delta \mathbf{u}[dt, \langle \mathbf{U}_r \rangle dt] \Rightarrow$  space and time issue
- Langevin models (diffusion process for the velocity of the fluid seen)

$$dU_{s,i} = A_{s,i}dt + B_{s,ij}dW_j$$

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## Langevin model for turbulent dispersion

• One general model

$$dU_{s,i} = -\frac{1}{\rho_{f}} \frac{\partial \langle P \rangle}{\partial x_{i}} dt + (\langle U_{\rho,j} \rangle - \langle U_{f,j} \rangle) \frac{\partial \langle U_{f,i} \rangle}{\partial x_{j}} dt$$
$$- \frac{(U_{s,i} - \langle U_{f,i} \rangle)}{T_{L,i}^{*}} dt + \sqrt{\langle \epsilon \rangle \left( C_{0} b_{i} \tilde{k} / k + \frac{2}{3} (b_{i} \tilde{k} / k - 1) \right)} dW_{i}$$
$$\tilde{k} = \frac{3}{2} \frac{\sum_{i=1}^{3} b_{i} \langle u_{f,i}^{2} \rangle}{\sum_{i=1}^{3} b_{i}} b_{i} = \frac{T_{L}}{T_{L,i}^{*}} \quad 1/T_{L} = \left( \frac{1}{2} + \frac{3}{4} C_{0} \right) \frac{\langle \epsilon \rangle}{k}$$
$$T_{L,1}^{*} = \frac{T_{L}}{\sqrt{1 + \beta^{2} \frac{|\langle \mathbf{U}_{f} \rangle|^{2}}{2k/3}}} \quad T_{L,2}^{*} = T_{L,3}^{*} = \frac{T_{L}}{\sqrt{1 + 4\beta^{2} \frac{|\langle \mathbf{U}_{f} \rangle|^{2}}{2k/3}}$$

• When  $\tau_{\rho} \rightarrow$  0, the Langevin model for fluid particles is retrieved

$$dU_{s,i} = -\frac{1}{\rho_f} \frac{\partial \langle P \rangle}{\partial x_i} dt - \frac{U_{s,i} - \langle U_{f,i} \rangle}{T_L} dt + \sqrt{C_0 \langle \epsilon \rangle} dW_i$$

Correspondence with Reynolds-stress modelling Consistency issues in the hybrid "Euler/Lagrange" formulation

### Plan

General guidelines: Modelling approach and actual models

- Choice of a modelling approach for single-phase reactive flows
- Choice of a modelling approach for polydispersed two-phase flows

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### Consistency issues in the fluid limit case

- Correspondence with Reynolds-stress modelling
  - The spurious issue of spurious drifts
  - Various forms of the Langevin equations
- Consistency issues in the hybrid "Euler/Lagrange" formulation

## A chosen list of specifications

- In two-phase flow modelling, stochastic (or pdf) models are used to predict averaged equations and constitutive relations
- It is important to check that the fluid limit is "well reproduced"
- An important issue to decide: what makes a given (fluid) particle stochastic model acceptable?
- Among other possible criteria, at least two appear compulsory:
  - the stochastic model should be free of spurious drifts and respect the mean-continuity equation
  - All convective terms in the mean Navier-Stokes equation AND in the second-order must be exactly reproduced (as well as dissipative terms)
- These two items means that, at least, the first two moment equations should be physically and well reproduced
- Other criteria include, for example, Gaussian pdfs in homogeneous cases and the ability to predict non-Gaussianity in inhomogeneous flows

## The issue of spurious drifts

- The issue of "spurious drifts" refers to the possibility of having (fluid) particles which tend to accumulate in some areas
- Concentration build-ups amounts to a false accumulation of mass (in incompressible flows): the mean continuity equation is not respected !
- Uniform particle concentration (no spurious drift) and respect of the mean continuity equation is equivalent (cf. Pope)

$$\left(\frac{\partial}{\partial t} + \langle U \rangle_k \frac{\partial}{\partial x_k}\right) \ln p(t, \mathbf{x}) = -\frac{\partial \langle U_k \rangle}{\partial x_k}, \quad \langle U \rangle_k (t, \mathbf{x}) = \langle U_k \, | \, \mathbf{x}(t) = \mathbf{x} \rangle$$

Any stochastic model formulated with instantaneous velocity as

$$\begin{cases} dx_{f,i} = U_{f,i} dt \\ dU_{f,i} = -\frac{1}{\rho} \frac{\partial \langle P \rangle}{\partial x_i} dt + StM(\mathbf{x}_f, \mathbf{U}_f) \end{cases}$$

where  $\langle P \rangle$  is such that the mean-continuity equation is satisfied and with  $\langle StM(\mathbf{x}_f, \mathbf{U}_f) | \mathbf{x}(t) = \mathbf{x} \rangle = 0$  is free of spurious drifts

## Instantaneous or fluctuating velocities? (1)

- Any stochastic model can be expressed in terms of the instantaneous velocity, U(t), or of the fluctuating velocitiy u(t) = U(t) − ⟨U⟩(t, x<sub>f</sub>(t))
- This is not an issue of stochastic modelling but a change of variables
- Indeed, any stochastic model formulated in terms of U as

$$\begin{cases} d\mathbf{x}_{f,i} = U_{f,i} \, dt \\ dU_{f,i} = \underbrace{-\frac{1}{\rho} \frac{\partial \langle \boldsymbol{P} \rangle}{\partial x_i}}_{no \, spurious \, drift} dt + \underbrace{StM(\mathbf{x}_f, \mathbf{U}_f)}_{model} \end{cases}$$

• is equivalent to a stochastic model in terms of u formulated as

$$\begin{cases} d\mathbf{x}_{f,i} = (\langle \mathbf{U} \rangle_{f,i} + u_{f,i}) \ dt \\ du_{f,i} = \underbrace{\frac{\partial \langle u_i u_k \rangle}{\partial x_k} \ dt}_{(a) \ no \ spurious \ drift} - \underbrace{u_k \frac{\partial \langle U_{f,i} \rangle}{\partial x_k} \ dt}_{(b) \ production \ term} + \underbrace{StM(\mathbf{x}_f, \mathbf{u}_f)}_{model} \end{cases}$$

# Instantaneous or fluctuating velocities? (2)

- The first two terms on the rhs of the equation for **u** are **essential** and **compulsory**: they **must be respected** by any acceptable model
- The term (*a*) is necessary to ensure that no spurious drifts occur: *consistency with the mean Navier-Stokes equation*
- The term (*b*) is compulsory in order to treat correctly **convective terms** and, in particular to obtain the correct form of the second-order equations: *consistency with Reynolds-stress modelling*

$$\underbrace{\frac{\partial \langle u_{i} u_{j} \rangle}{\partial t} + \langle U_{k} \rangle \frac{\partial \langle u_{i} u_{j} \rangle}{\partial x_{k}} + \frac{\partial \langle u_{i} u_{j} u_{k} \rangle}{\partial x_{k}}}_{\langle d(u_{i} u_{j}) \rangle} = \underbrace{-\langle u_{i} u_{k} \rangle \frac{\partial \langle U_{j} \rangle}{\partial x_{k}} - \langle u_{j} u_{k} \rangle \frac{\partial \langle U_{i} \rangle}{\partial x_{k}}}_{correct production term}}_{\langle u_{i} StM_{i} \rangle + \langle u_{i} StM_{i} \rangle}$$

- A stochastic model for **U** requires 3 gradients (for  $\nabla \langle P \rangle$ )
- A stochastic model for **u** requires 27 gradients (for  $\nabla \langle U \rangle$  and  $\nabla \langle u_i u_j \rangle$ )

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## Consistency issues in the Moment/PDF approach (1)

In classical hybrid Moment/PDF formulations (Eulerian/Lagrangian), evaluations of fluid and particle statistics may seem well separated ⇒ fluid mean fields are solved by a Moment Approach based on the choice of a turbulence model (*k* − *ε* or RSM): ⟨*P*⟩, *k*, ⟨*ε*⟩, ⟨**U**⟩<sub>*f*</sub>, ⟨*u*<sub>*f*,*i*</sub>*u*<sub>*f*,*j*</sub>⟩ ⇒ particle statistics are solved by the Lagrangian approach based on the choice of a stochastic model for **Z** = (**x**<sub>*p*</sub>, **U**<sub>*p*</sub>, **U**<sub>*s*</sub>, …)

$$\begin{cases} d\mathbf{x}_{\rho} = \mathbf{U}_{\rho} dt \\ d\mathbf{U}_{\rho} = \frac{\mathbf{U}_{s} - \mathbf{U}_{\rho}}{\tau_{\rho}} dt + \mathbf{g} dt \\ d\mathbf{U}_{s} = \mathbf{\Pi}(\mathbf{Z}, \langle \mathbf{H}_{\rho} \rangle, \langle \mathbf{U}_{f} \rangle^{E}, \langle \mathbf{H}_{f} \rangle) dt \\ - \frac{\mathbf{U}_{s} - \langle \mathbf{U}_{f} \rangle^{E}}{T_{L}^{*}} dt + \mathbf{B}(\mathbf{Z}, \langle \mathbf{H}_{\rho} \rangle, \langle \mathbf{U}_{f} \rangle^{E}, \langle \mathbf{H}_{f} \rangle) d\mathbf{W} \end{cases}$$

 $\Rightarrow$  Monte Carlo estimations give particle mean values:  $\langle \mathbf{U}_{p} \rangle, \langle u_{p,i} u_{p,j} \rangle$ .

## Consistency issues in the Moment/PDF approach (2)

• When  $\tau_{p} \rightarrow 0$ , the particle stochastic model reverts to the fluid one

$$\begin{cases} d\mathbf{x}_{f} = \mathbf{U}_{f} dt \\ d\mathbf{U}_{f} = -\frac{1}{\rho_{f}} \frac{\partial \langle \boldsymbol{P} \rangle}{\partial \mathbf{x}} dt - \frac{\mathbf{U}_{f} - \langle \mathbf{U}_{f} \rangle^{\mathcal{E}}}{T_{L}} dt + \sqrt{C_{0} \langle \epsilon \rangle} d\mathbf{W} \end{cases}$$

- The Lagrangian solver yield mean fields:  $\langle \mathbf{U}_f \rangle^L, \langle u_{f,i} u_{f,j} \rangle^L$
- We are now dealing with duplicate mean fields!
- The consistency issue requires:  $\langle \mathbf{U}_f \rangle^L = \langle \mathbf{U}_f \rangle^E$ ,  $\langle u_{f,i} u_{f,j} \rangle^L = \langle u_{f,i} u_{f,j} \rangle^E$ .
- the fluid Langevin model is consistent with a Rotta RSM

$$\frac{\partial \langle u_i u_j \rangle}{\partial t} + \langle U_k \rangle \frac{\partial \langle u_i u_j \rangle}{\partial x_k} + \frac{\partial \langle u_i u_j u_k \rangle}{\partial x_k} = -\langle u_i u_k \rangle \frac{\partial \langle U_j \rangle}{\partial x_k} - \langle u_j u_k \rangle \frac{\partial \langle U_i \rangle}{\partial x_k} \\ - (1 + \frac{3}{2}C_0) \frac{\langle \epsilon \rangle}{k} \left( \langle u_i u_j \rangle - \frac{2}{3}k \delta_{ij} \right) - \frac{2}{3} \delta_{ij} \langle \epsilon \rangle$$

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## Consistency issues in the Moment/PDF approach (3)

- <u>Issue</u>: What happens when two different, and possible inconsistent, turbulence models are used in the Fluid Description (Moment or Eulerian phase) and in the Particle Description (PDF or Lagrangian phase)?
- "A better (U<sub>f</sub>)<sup>E</sup> fed into the Langevin model gives better outcomes...".
   Is that really so when consistency issues are disregarded?
- this question is best assessed in the fluid-limit case by considering various (U<sub>f</sub>)<sup>E</sup> taken as input into the Langevin model

$$\begin{cases} d\mathbf{x}_{f} = \mathbf{U}_{f} dt \\ d\mathbf{U}_{f} = -\frac{1}{\rho_{f}} \frac{\partial \langle \boldsymbol{P} \rangle}{\partial \mathbf{x}} dt - \frac{\mathbf{U}_{f} - \langle \mathbf{U}_{f} \rangle^{E}}{T_{L}} dt + \sqrt{C_{0} \langle \epsilon \rangle} d\mathbf{W} \end{cases}$$

• Potential discrepancies are also present for non-zero inertia particles.

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### Numerical assessments (1)

 Numerical tests were performed in a channel with various (U<sub>f</sub>)<sup>E</sup> ("A note on the consistency of hybrid Eulerian/Lagrangian approach to multiphase flows" S. Chibbaro and J.-P. Minier, IJMF, 37, 293-297, 2011)



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## Numerical assessments (2)



