

LARGE-EDDY SIMULATIONS AND RELATED TECHNIQUES

UGO PIOMELLI

*Department of Mechanical and Materials Engineering
Queen's University
Kingston, Ontario
CANADA*

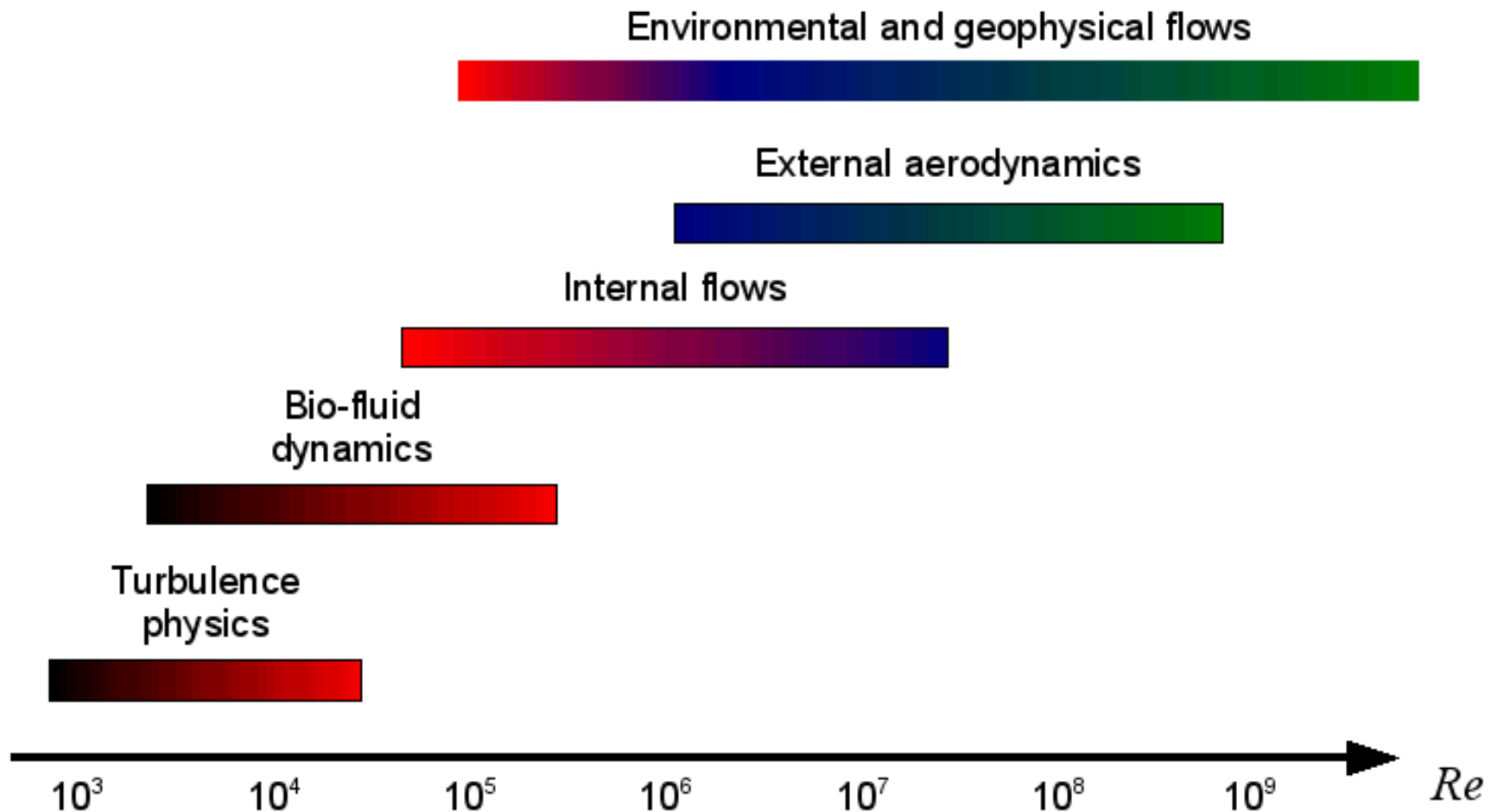
MAY 18-22, 2015



UNIVERSITÀ
DEGLI STUDI
DI UDINE

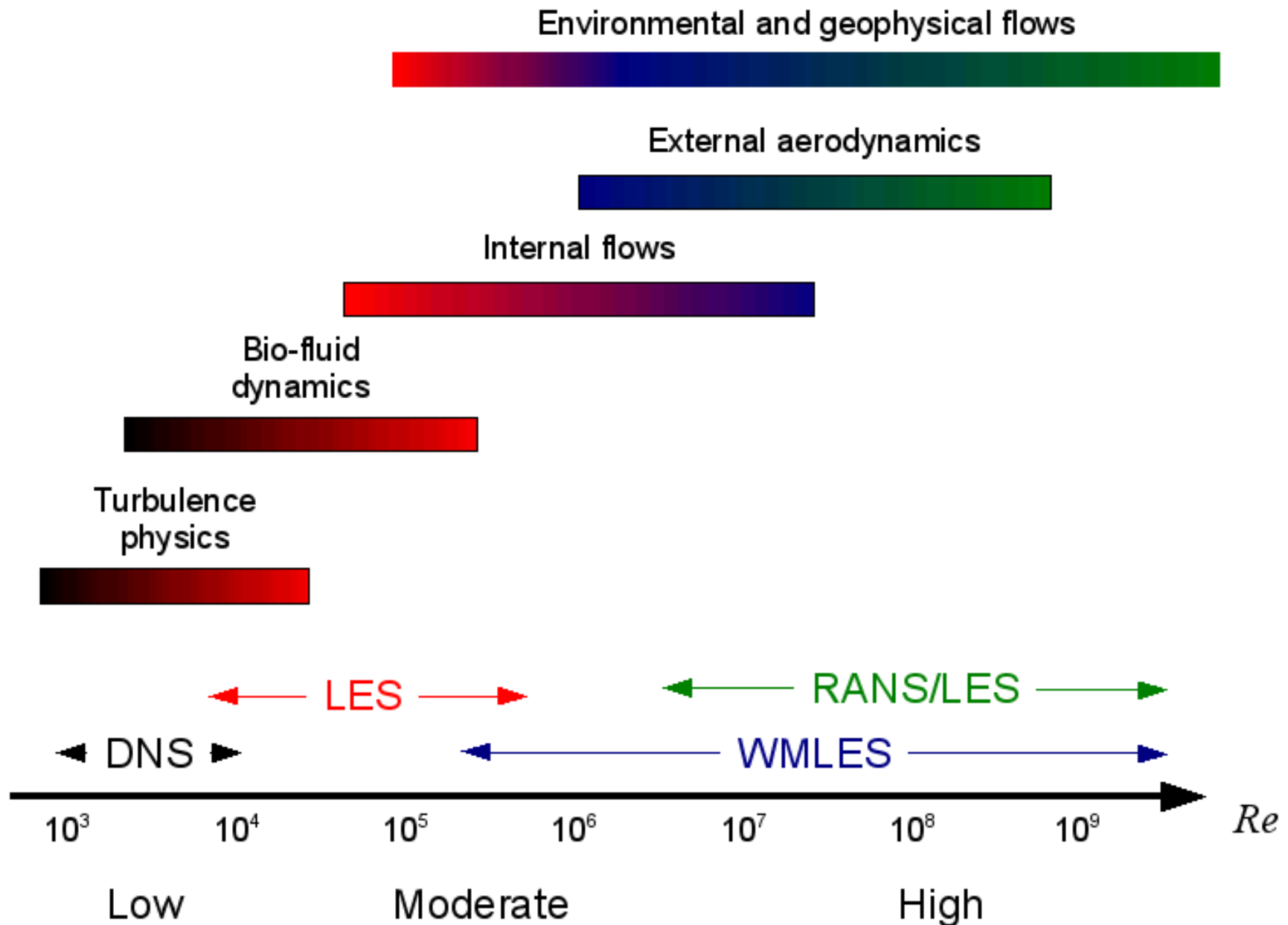


- Turbulence plays a critical role in engineering and in the physical sciences.





- Turbulence plays a critical role in engineering and in the physical sciences.
- Eddy-resolving techniques are becoming the most complete, general methods to solve turbulent flow problems.
 - *Include more physics*
 - *Are more accurate*
 - *Are becoming more affordable*





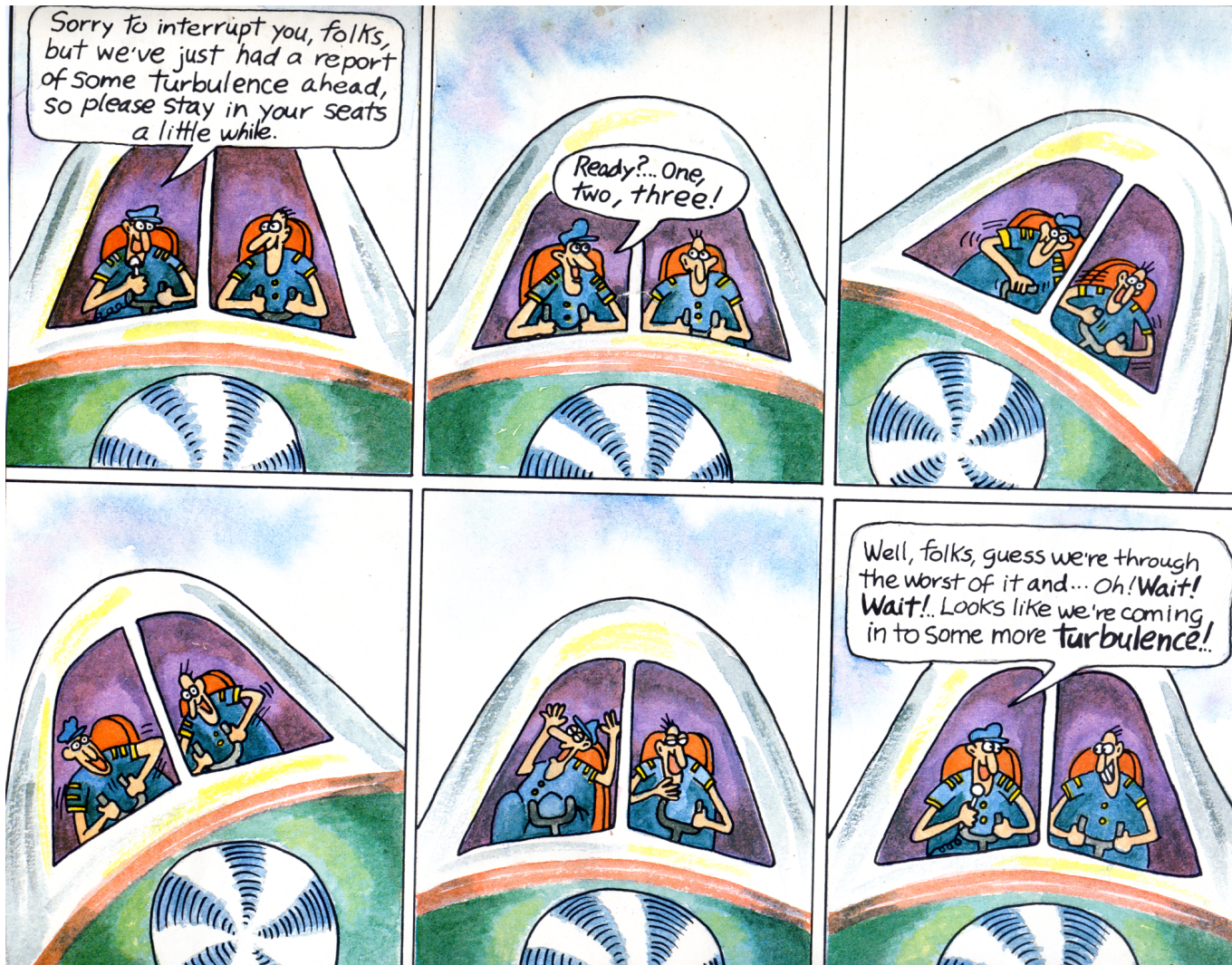
- **Motivation**
- Governing equations
- Boundary conditions
- Subfilter-scale modelling
- Validation of an LES
- Applications
- Challenges
- Conclusions and course roadmap



- **Motivation:**
 - *What is turbulence?*
 - *Review of turbulence physics*
 - *Why simulations?*
 - *Methodologies*
 - *Resolution requirements*
- Governing equations for LES
- Boundary conditions
- Subfilter-scale modelling
- Validation of an LES
- Applications
- Challenges
- Conclusions



- What is turbulence?





- Turbulence is the chaotic state of fluid motion that occurs when convective effects are much larger than viscous effects.
 - *Ship:* $Re = o(10^9)$
 - *Ocean:* $Re = o(10^9)$
 - *Aircraft:* $Re = o(10^8)$
 - *Car:* $Re = o(10^6)$
 - *Golf ball:* $Re = o(10^5)$
 - *Artery:* $Re = o(10^3)$

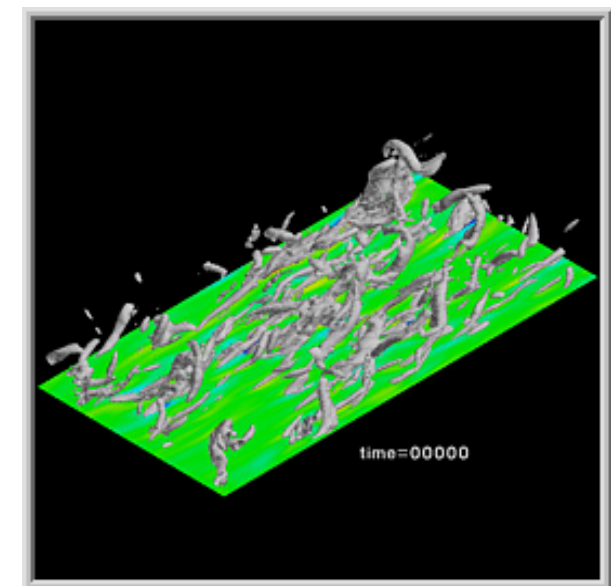


- Turbulence is the chaotic state of fluid motion that occurs when convective effects are much larger than viscous effects.
- Turbulence contains vorticity: $\omega = \nabla \times \mathbf{V}$
 - *Coherent vortical motions \Rightarrow Eddies*



- Turbulence is the chaotic state of fluid motion that occurs when convective effects are much larger than viscous effects.
- Turbulence contains vorticity: $\omega = \nabla \times \mathbf{V}$
 - *Coherent vortical motions \Rightarrow Eddies*
 - *Deterministic but occur at random locations*
 - *Turbulence is not a completely stochastic phenomenon*
 - *Statistical descriptions are inadequate.*

Coherent eddies in turbulent channel flow
(Bewley, Temam and Moin 2000).





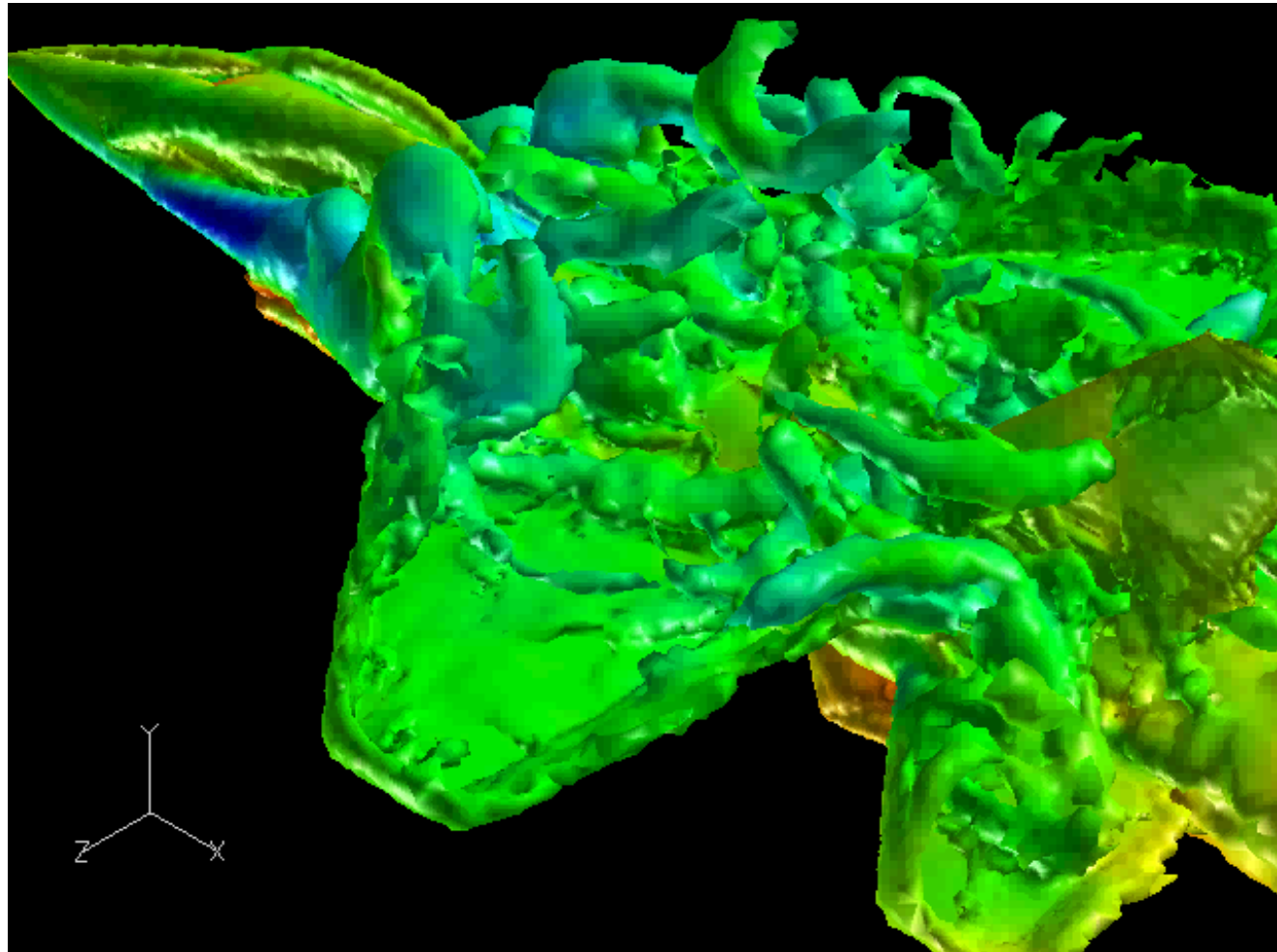
- Turbulence is the chaotic state of fluid motion that occurs when convective effects are much larger than viscous effects.
- Turbulence contains vorticity: $\omega = \nabla \times \mathbf{V}$
- Turbulence is not a completely random phenomenon.
- Turbulence enhances mixing.
- Turbulence is the natural state of fluid motion.



- Turbulence is all around us
 - *Engineering devices*
 - Aerospace applications

Flow over an F-16
at 45° angle of attack

Calculation by
J. Forsythe
(Cobalt)

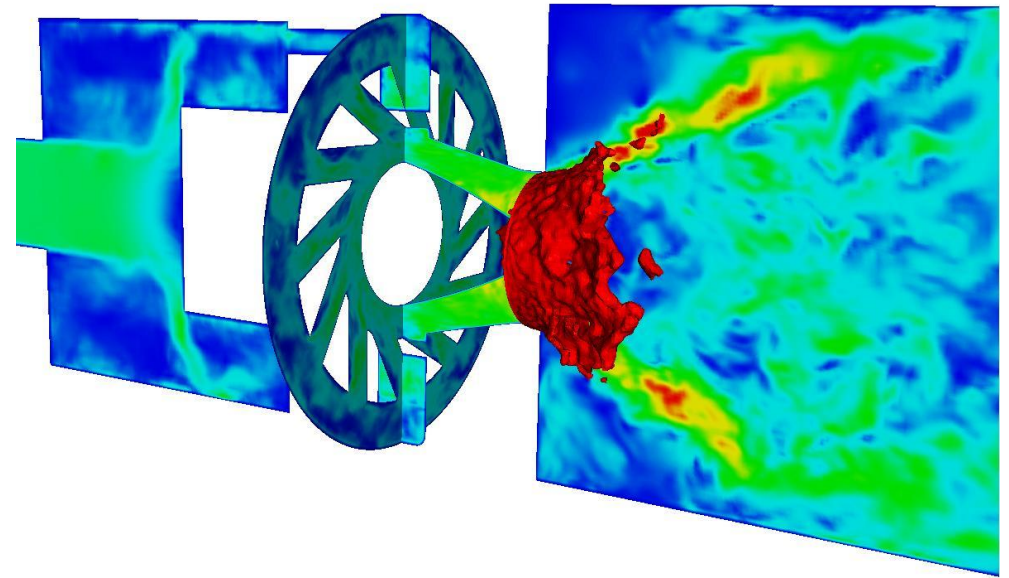




- Turbulence is all around us

- *Engineering devices*

- Aerospace applications
 - Naval applications
 - Vehicle aerodynamics
 - Combustion systems



Flow in a combustor
Calculation by H. Pitsch
(Stanford University)



- Turbulence is all around us

- *Engineering devices*

- Aerospace applications
 - Naval applications
 - Vehicle aerodynamics
 - Combustion systems

- *Geophysical applications*

- Oceanography



Frontal Features and Vortices
in the Tidal Potomac



- Turbulence is all around us
 - *Engineering devices*
 - Aerospace applications
 - Naval applications
 - Vehicle aerodynamics
 - Combustion systems
 - *Geophysical applications*
 - Oceanography
 - Meteorology and weather prediction
 - Environmental engineering
 - *Biological flows*



- Turbulence is all around us

- *Engineering devices*

- Aerospace applications
 - Naval applications
 - Vehicle aerodynamics
 - Combustion systems

- *Geophysical applications*

- Oceanography
 - Meteorology and weather prediction
 - Environmental engineering

- *Biological flows*

⇒ It is critical to predict and analyse the effects of turbulence on mass, momentum and energy transport



- **Motivation:**
 - *What is turbulence?*
 - *Review of turbulence physics*
 - *Why simulations?*
 - *Methodologies*
 - *Resolution requirements*
- Governing equations for LES
- Boundary conditions
- Subfilter-scale modelling
- Validation of an LES
- Applications
- Challenges
- Conclusions



- Consider a high-Reynolds-number turbulent flow.
 - *Length scale* L
 - *Velocity scale* U
 - *Reynolds number* UL/ν
- Turbulence made up of eddies of different sizes:
 - *Length-scale* ℓ
 - *Velocity scale* $u(\ell)$
 - *Time scale* $\tau(\ell)$

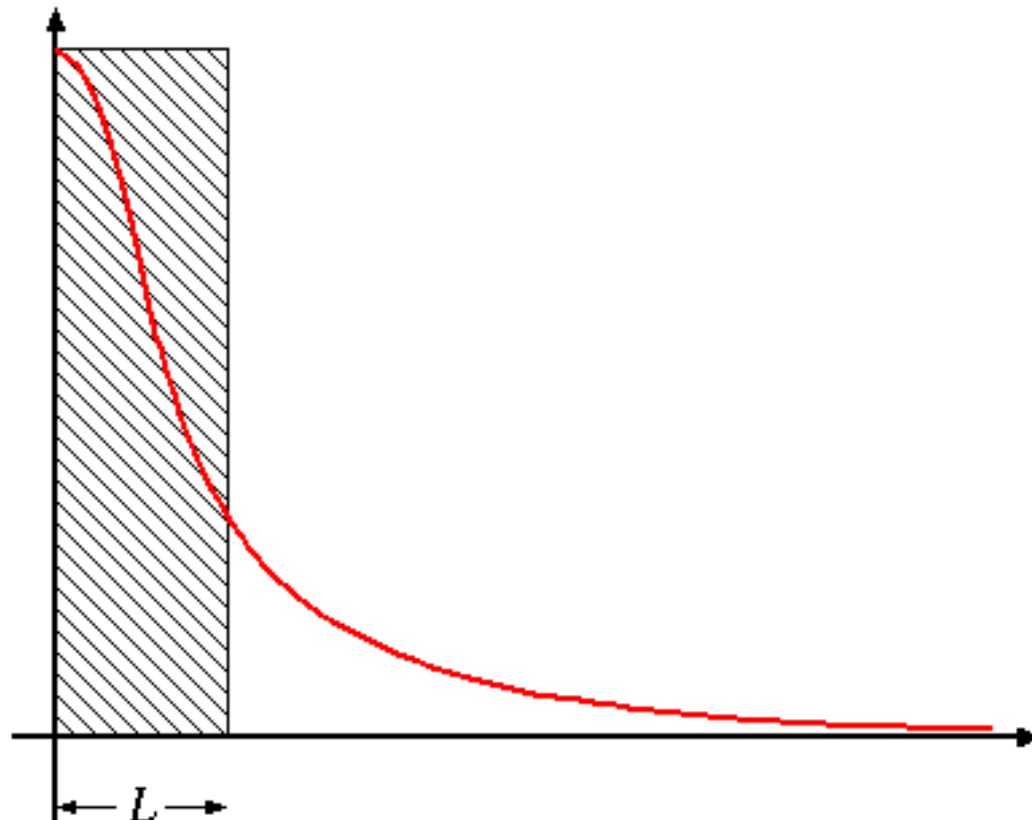


- Largest eddies: $\ell_o \sim L; \quad u_o \sim U$
 $\Rightarrow Re_o = \frac{u_o \ell_o}{\nu} = o(Re) \gg 1$
 \Rightarrow Dissipation is unimportant
- Large eddies break up and transfer their energy to smaller eddies.
- The process continues until the smallest eddies, whose Re is so small that dissipation becomes important.



- Typical length scale of the largest eddies is the integral scale L

$$L_{ij} = \int_0^{\infty} R_{ij}(r) dr$$





- Energy

- *Enters at the large scales (production likes anisotropy)*
- *Is progressively transferred to smaller scales*
- *Is dissipated by the smallest scales*

- Dissipation

- *Takes place at the smallest scales.*
- *Is determined by the production (largest scales).*
- *Energy $\propto u_o^2$ Time-scale $\propto \ell_o/u_o$*

$$\Rightarrow \varepsilon \propto \frac{u_o^3}{\ell_o}$$

(Independent of ν)



- The large eddies are affected by the b.c.
⇒ are anisotropic
- As they break up they lose memory of the b.c.s
⇒ become more isotropic.
- Kolmogorov's hypothesis of local isotropy:
 - *At sufficiently high Reynolds number, the small-scale motions $l \ll \ell_o$ are statistically isotropic*



THE KOLMOGOROV SCALES

- At small scales, ν is important, ε determines the rate of energy transfer.
- Kolmogorov's 1st similarity hypothesis:
- At sufficiently high Reynolds number, the statistics of the small-scale motions are uniquely determined by ε and ν .
- We can build length-, time- and velocity scales using ε and ν :

Kolmogorov scales

$$\begin{aligned}\eta &= (\nu^3 / \varepsilon)^{1/4} \\ u_\eta &= (\nu \varepsilon)^{1/4} \\ \tau_\eta &= (\nu / \varepsilon)^{1/2}\end{aligned}$$



THE KOLMOGOROV SCALES

- At small scales, ν is important, ε determines the rate of energy transfer.
- Kolmogorov's 1st similarity hypothesis:
- At sufficiently high Reynolds number, the statistics of the small-scale motions are uniquely determined by ε and ν .
- We can build length-, time- and velocity scales using ε and ν :

Kolmogorov scales

$$\eta/\ell_o \sim Re^{-3/4}$$

$$u_\eta/u_o \sim Re^{-1/4}$$

$$\tau_\eta/\tau_o \sim Re^{-1/2}$$



THE INERTIAL SUBRANGE

- Intermediate range: $\eta \ll \ell \ll \ell_o$
- $Re \gg 1$
 - \Rightarrow Viscous forces \ll inertia forces
 - $\Rightarrow \nu$ is unimportant.
- Kolmogorov's 2nd similarity hypothesis:
- At sufficiently high Reynolds number, the statistics of the motions of scale $\eta \ll \ell \ll \ell_o$ are uniquely determined by ε independent of ν .



Inertial (sub)range of turbulence



THE INERTIAL SUBRANGE

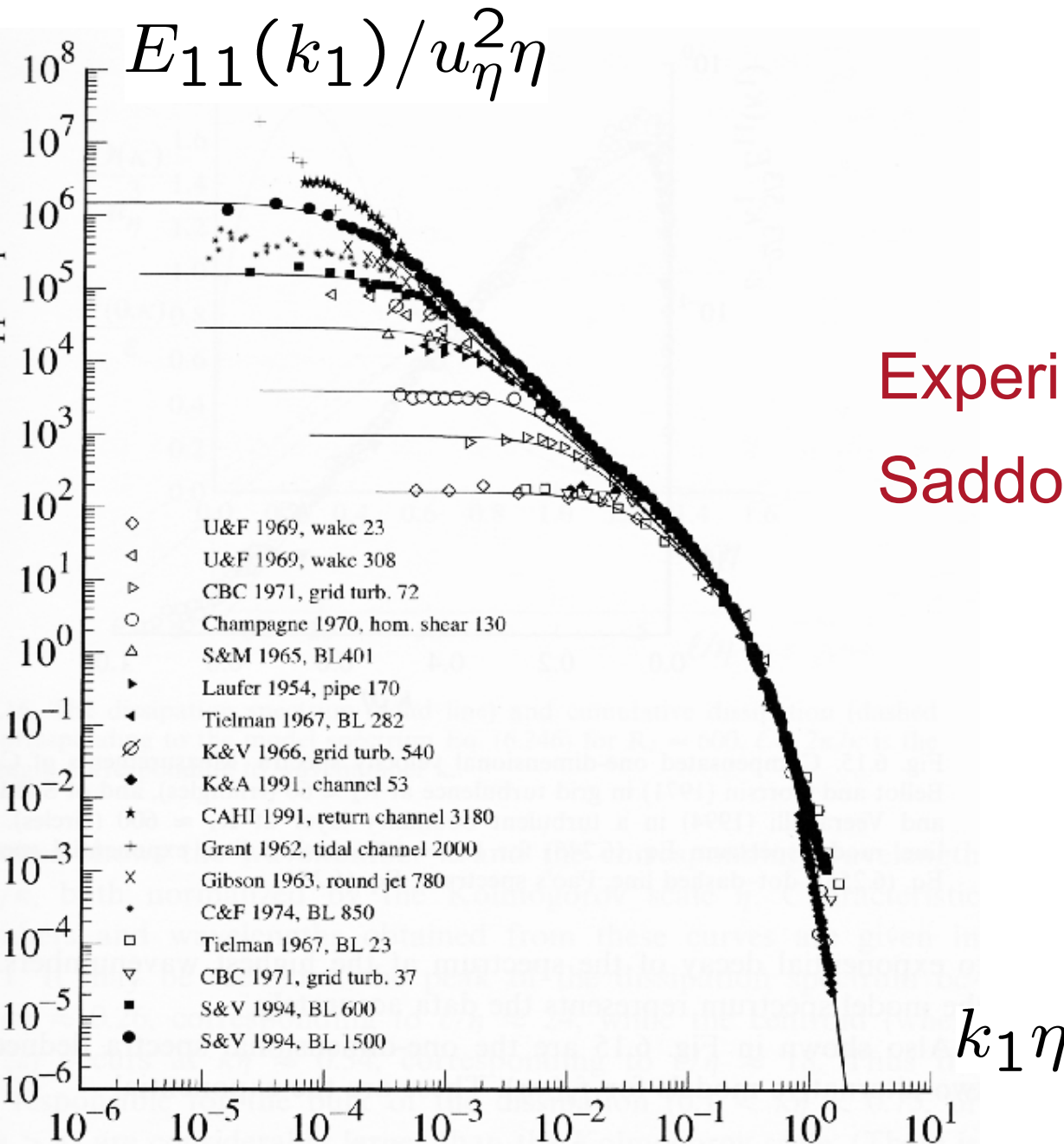
- In the inertial subrange:

$$2 \times \text{TKE} = \int_0^{\infty} E(\kappa) d\kappa; \quad E(\kappa) = f(\kappa, \varepsilon)$$

$$\Rightarrow E(\kappa) = C_K \varepsilon^{2/3} \kappa^{-5/3}$$

- C_K is the Kolmogorov constant

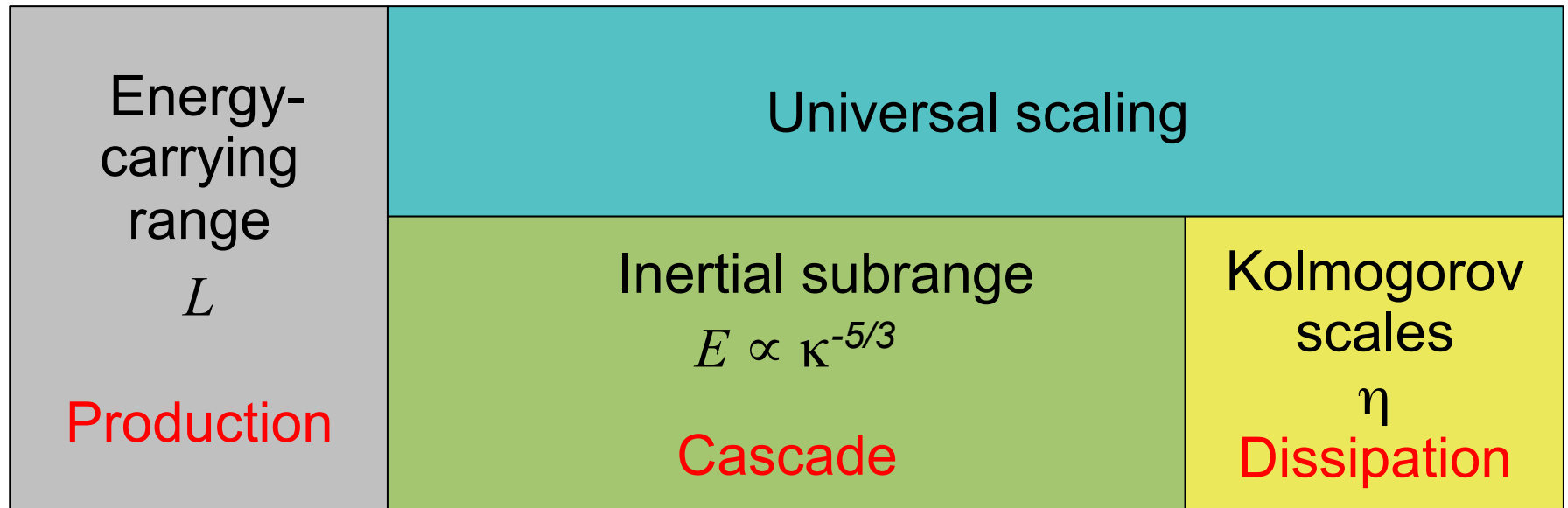
Kolmogorov
spectrum



Experimental verification:
Saddougui & Veeravalli (1994)



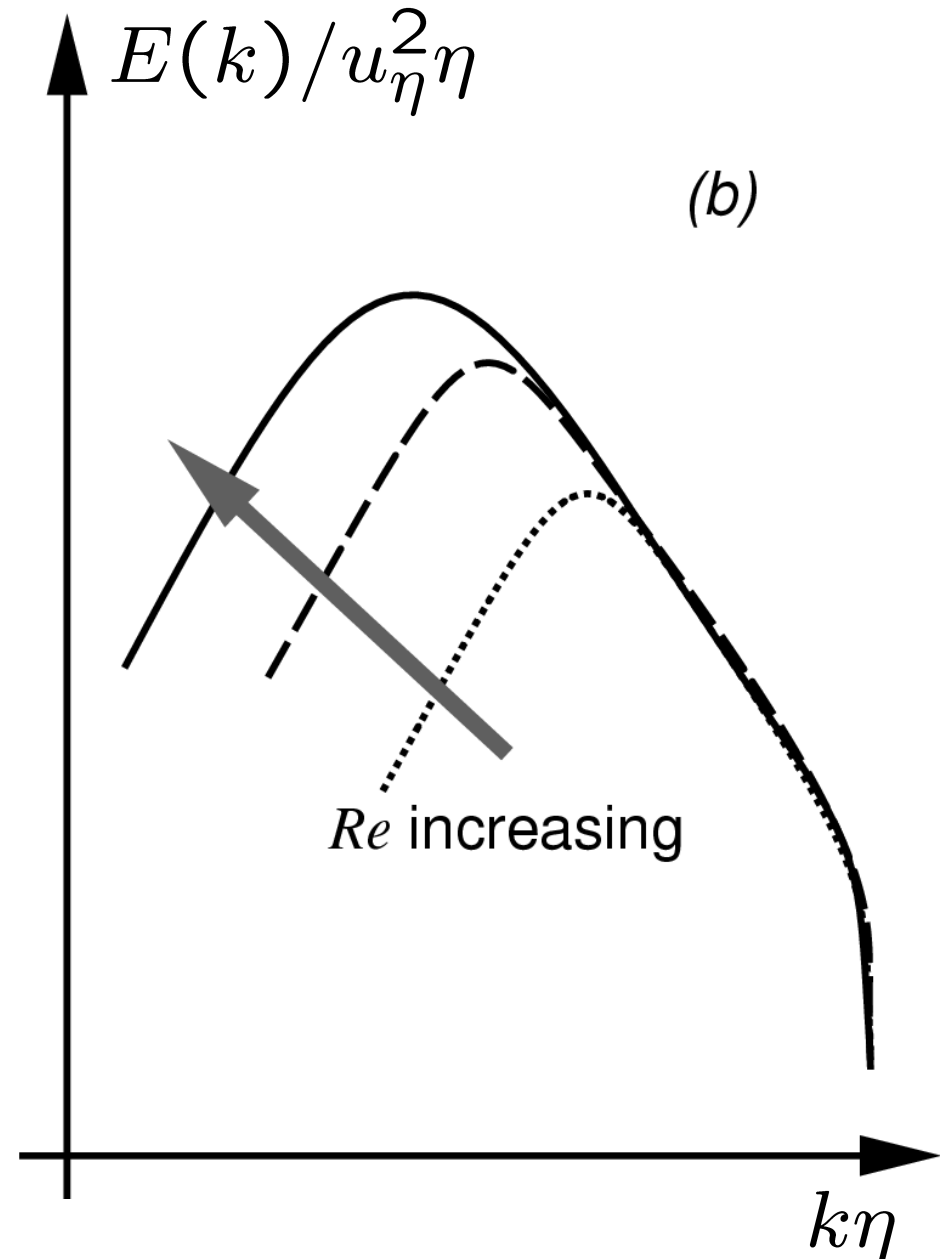
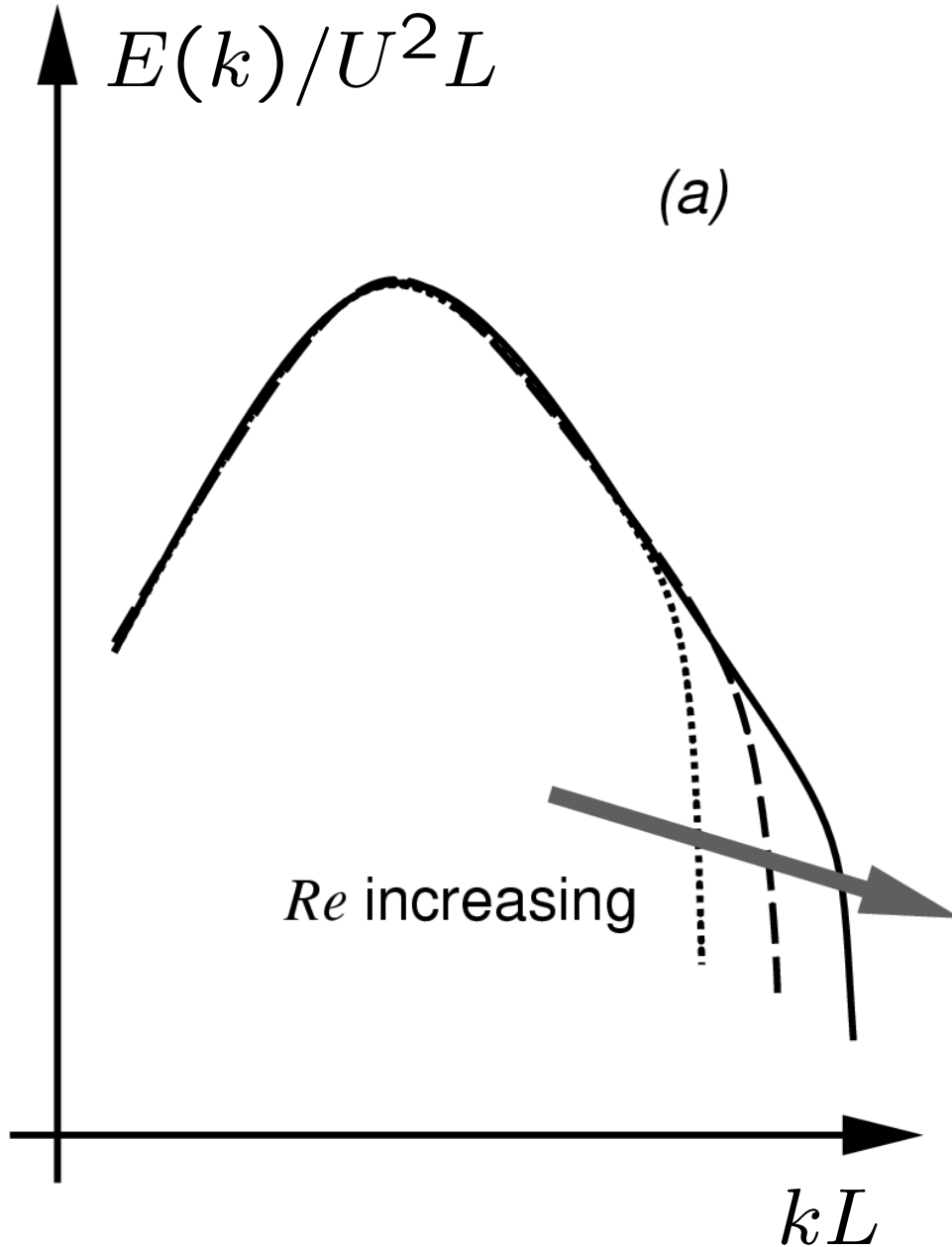
THE ENERGY CASCADE



← Increasing ℓ ————— Increasing κ →



REYNOLDS NUMBER EFFECTS





- Flows with non-zero mean-velocity gradient, and far from solid boundaries.
- Slow streamwise development:

$$\frac{\partial}{\partial x} \ll \frac{\partial}{\partial y}, \frac{\partial}{\partial r}$$

- Self-similarity occurs: a length-scale ℓ_r and a velocity-scale U_r can be found such that the normalized statistics depend on $\eta = y/\ell_r$ and not on x and y .

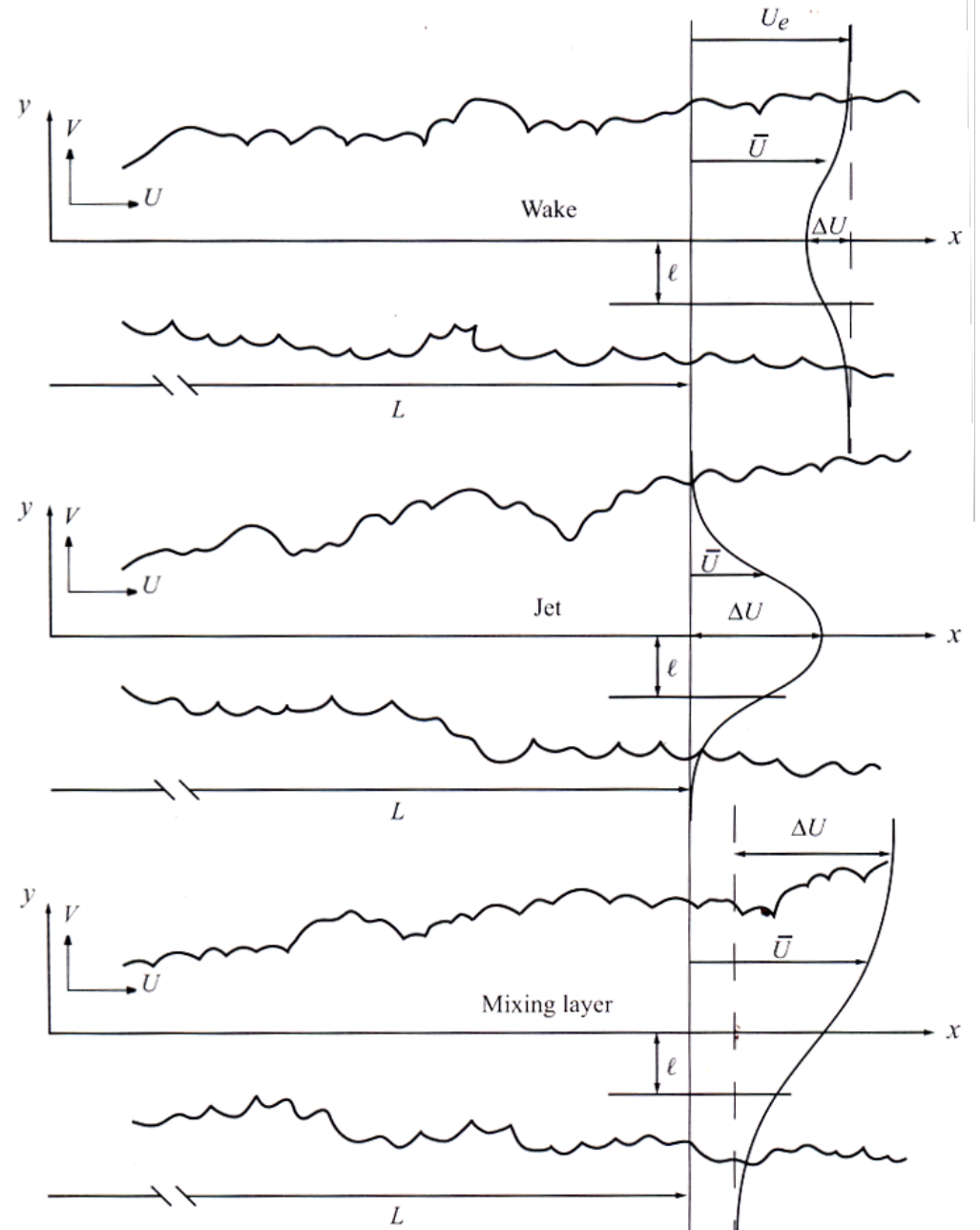


- Three typical cases:

- *Wake*

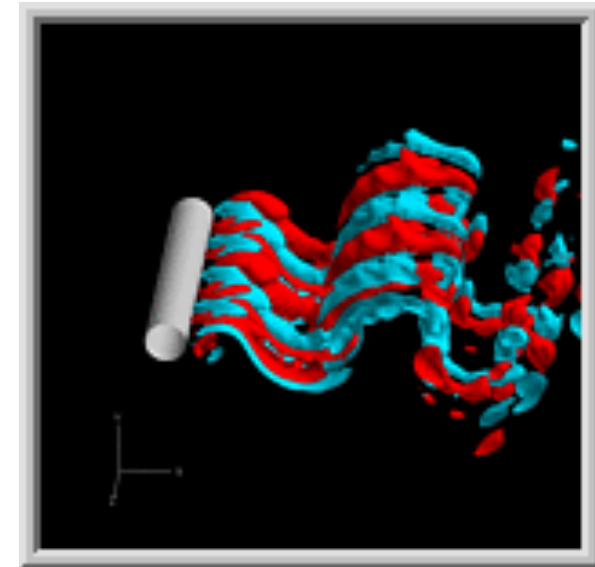
- *Jet*

- *Mixing layer*



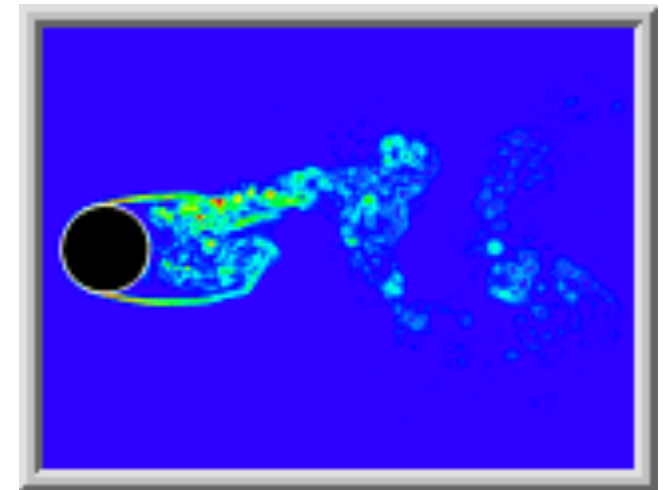


- Most studied cases:
 - *Cylinder*
 - *Sphere*
- Low Reynolds number:
 - *Vortex shedding (Kármán street)*
 - *Three-dimensionality (rib vortices)*
- Laminar boundary layer:
 - *Subcritical*
 - *Early separation*
 - *Transition takes place in the shear layer*
 - *Well-defined vortex shedding*
- Turbulent boundary layer
 - *Supercritical*





- Most studied cases:
 - *Cylinder*
 - *Sphere*
- Low Reynolds number:
- Laminar boundary layer:
 - *Subcritical*
 - *Early separation*
 - *Transition takes place in the shear layer*
 - *Fairly well-defined vortex shedding*
- Turbulent boundary layer





- Most studied cases:
 - *Cylinder*
 - *Sphere*
- Low Reynolds number
- Laminar boundary layer
- Turbulent boundary layer
 - *Coherent eddies are less evident but still exist*
 - *Delayed separation on body gives lower drag.*



- Self-similarity exists far downstream (80-150 D) with

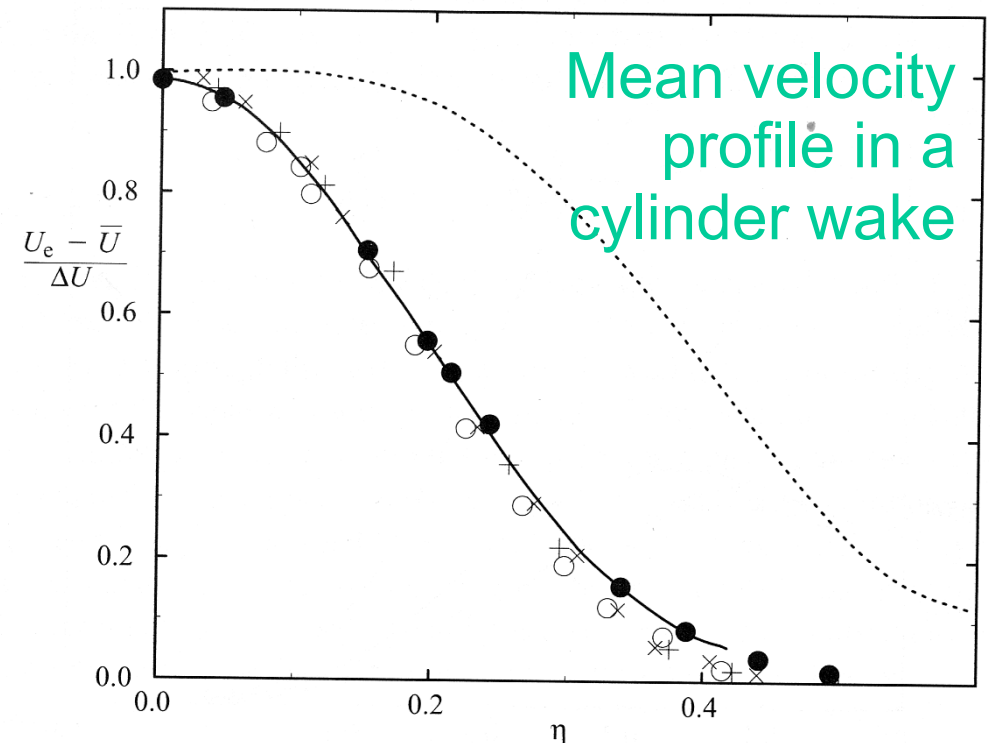
$$U_r = \Delta U = U_\infty - U_c$$
$$l_r = \text{Distance where } U = U_c + \Delta U/2$$

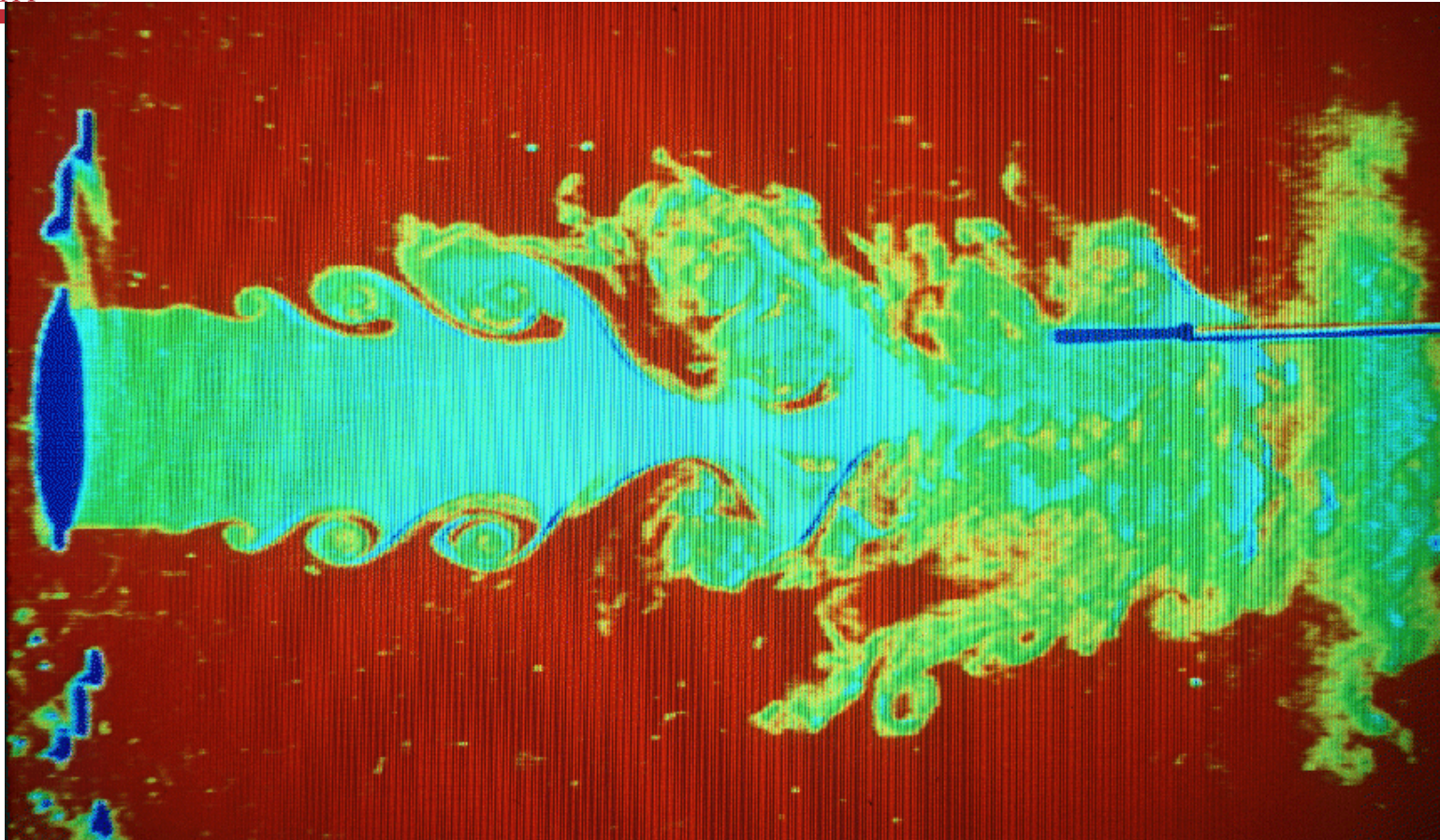
- Spreading rate for similarity:

$$l \sim x^{1/2}$$

- Velocity deficit decay rate for similarity:

$$\Delta U \sim x^{-1/2}$$





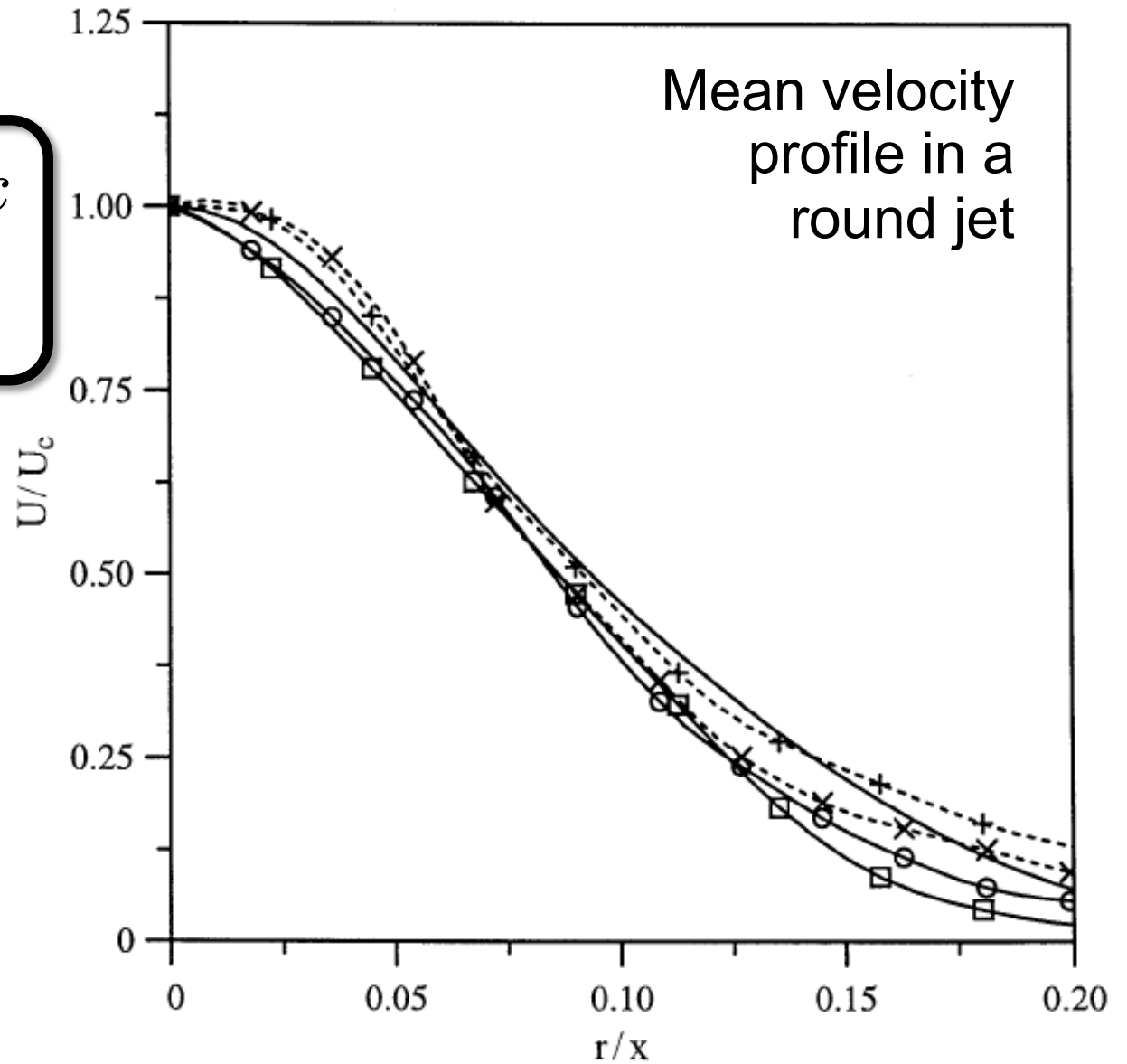
- The jet has excess momentum and spreads outwards.
- Instabilities in the near-jet result in vortex rings.
- At high Re the rings rapidly break down.



- Similarity with

$$l_r = \text{halfwidth} \sim x$$

$$U_r = U_c \sim x^{-1}$$

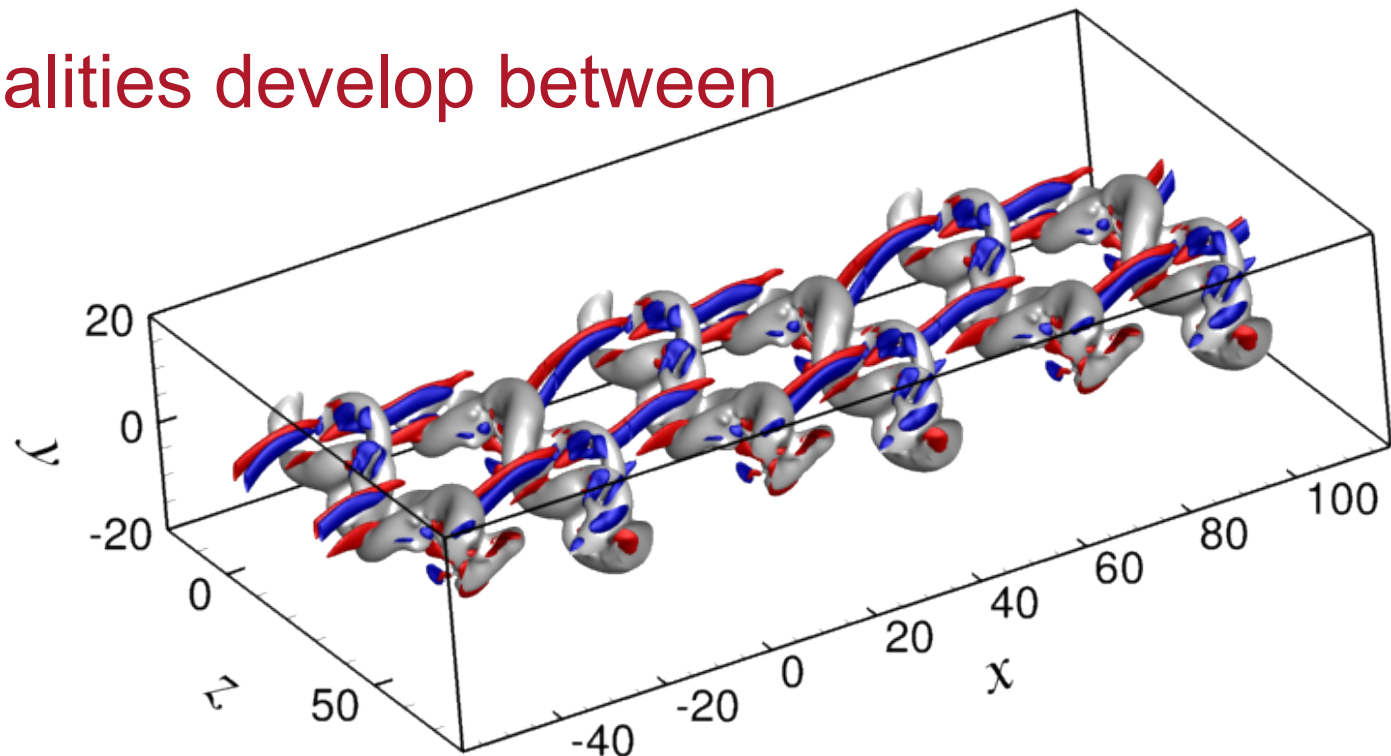




- The inflectional instability results in the formation of spanwise rollers that pair and grow.



- Three-dimensionalities develop between the rollers

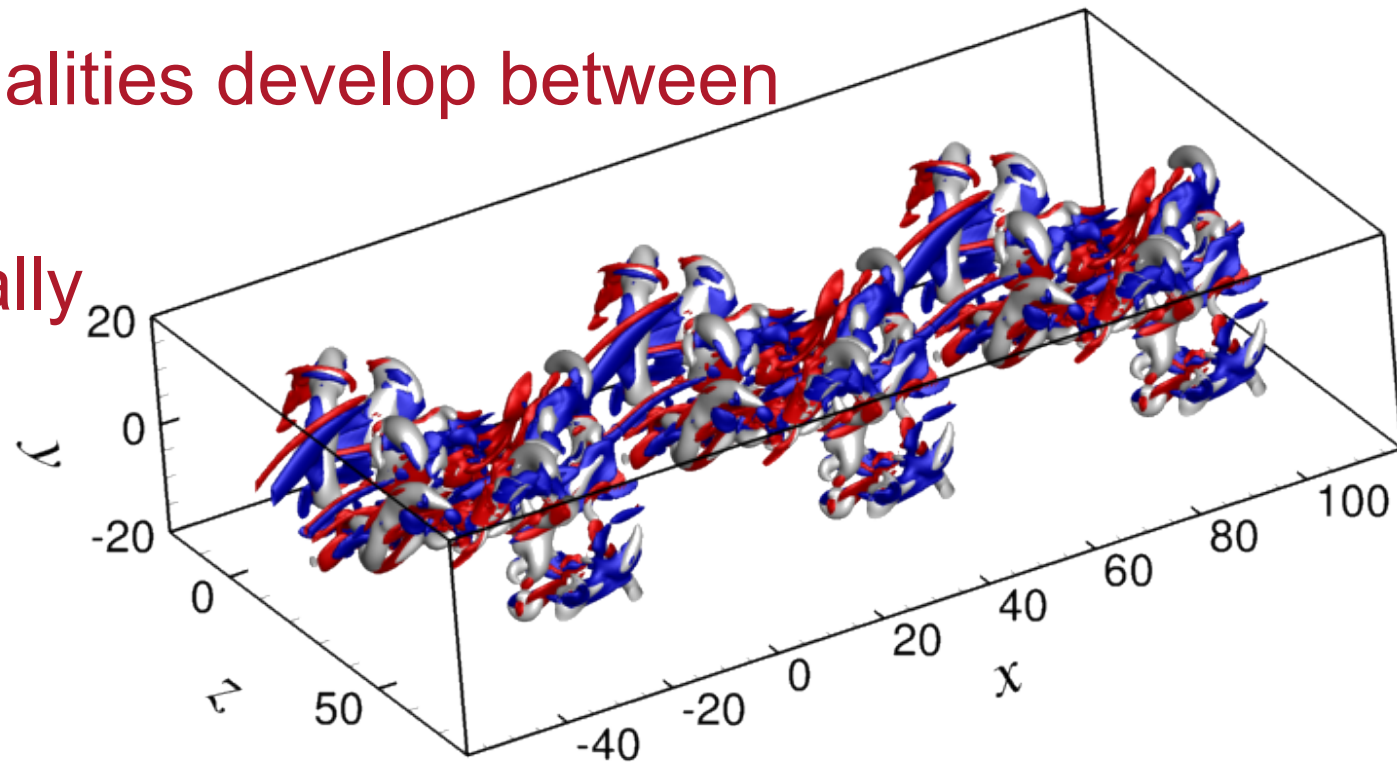




- The inflectional instability results in the formation of spanwise rollers that pair and grow.

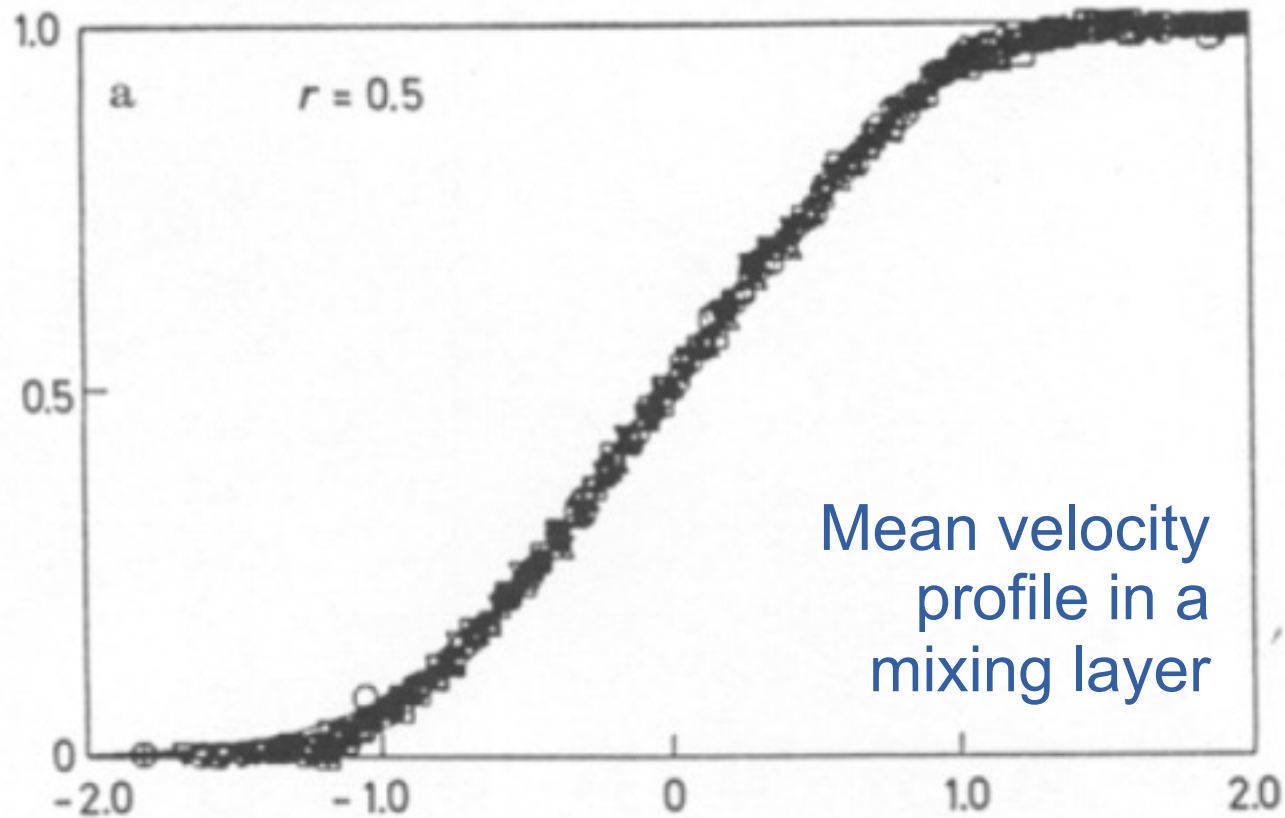


- Three-dimensionalities develop between the rollers
- The flow eventually transition to turbulence





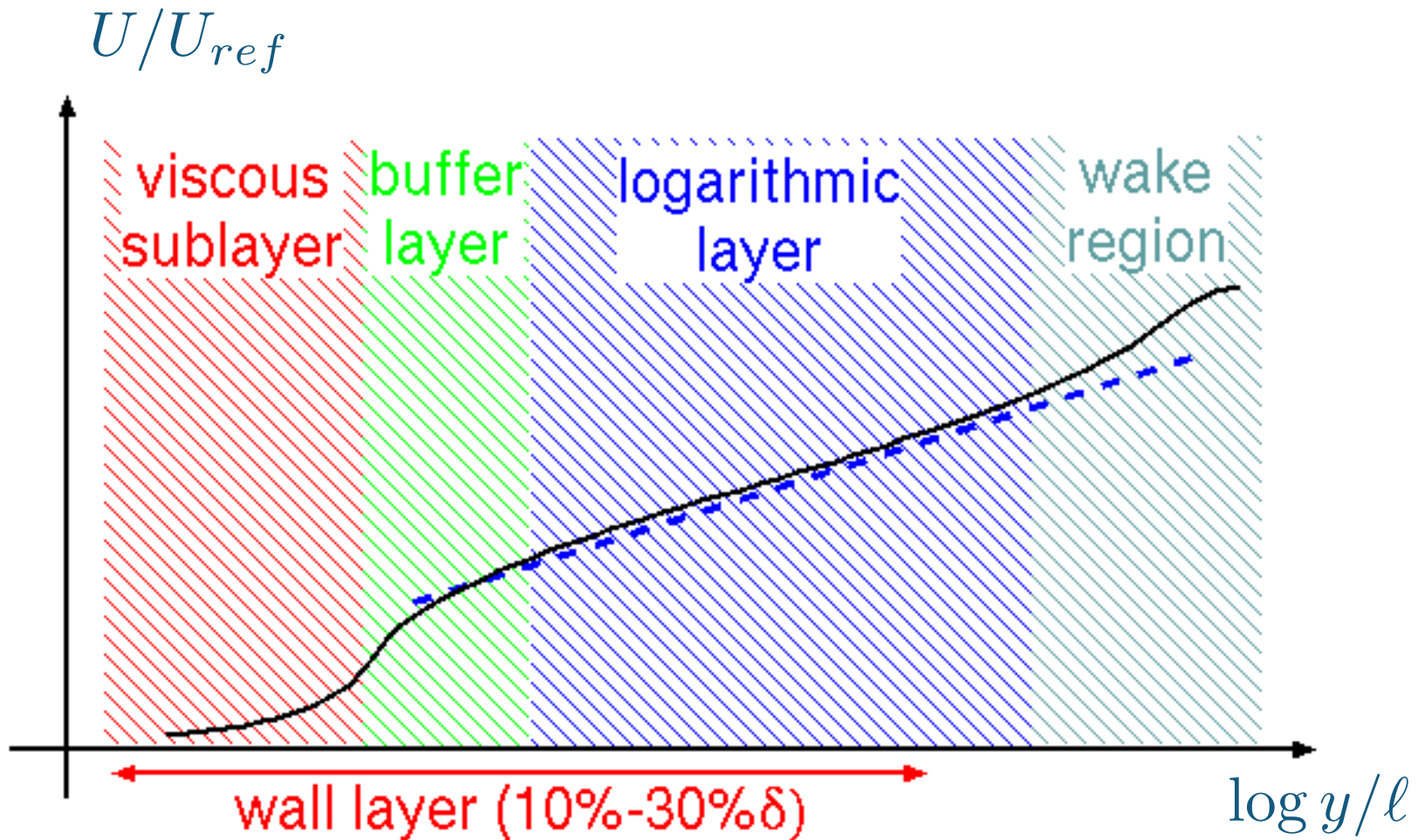
- Self-similarity is observed with $l_r \sim x$





- Very common in engineering applications.
 - *Pipes and ducts.*
 - *External aerodynamics.*
 - *Meteorology.*
 - *Internal combustion engines.*

- Four regions of the flow, different physics



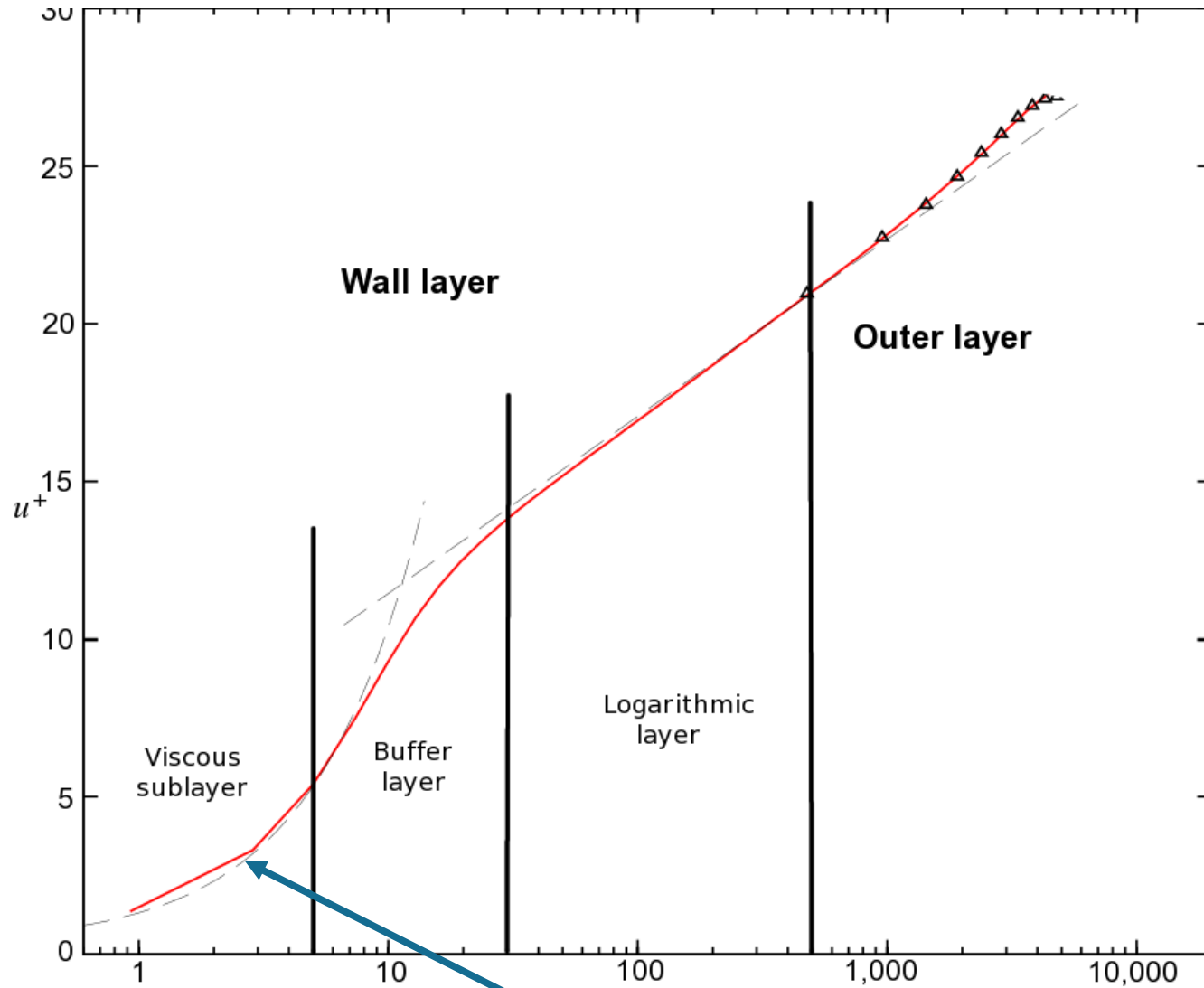


- Reynolds shear stress $-\langle u'v' \rangle \propto y^3$ is small.
- τ is due to viscous contribution only.
- Length and velocity scales should be obtained using only the wall stress τ_w , ρ and ν .

$$u_\tau = \left(\frac{\tau_w}{\rho} \right)^{1/2} \quad \text{and} \quad y_f = \frac{\mu}{(\rho\tau_w)^{1/2}} = \frac{\nu}{u_\tau}$$

$$u^+ = y^+$$

VISCOUS SUBLAYER



Linear behaviour
for $y^+ < 5$



- Reynolds stresses \gg viscous stresses.
- The flow is unaffected by the outer flow.
- All turbulence should depend on τ_w and distance from the wall:

$$\frac{dU}{dy} = \frac{u_\tau}{\kappa y}$$
$$-\langle u'v' \rangle = \tau_w$$

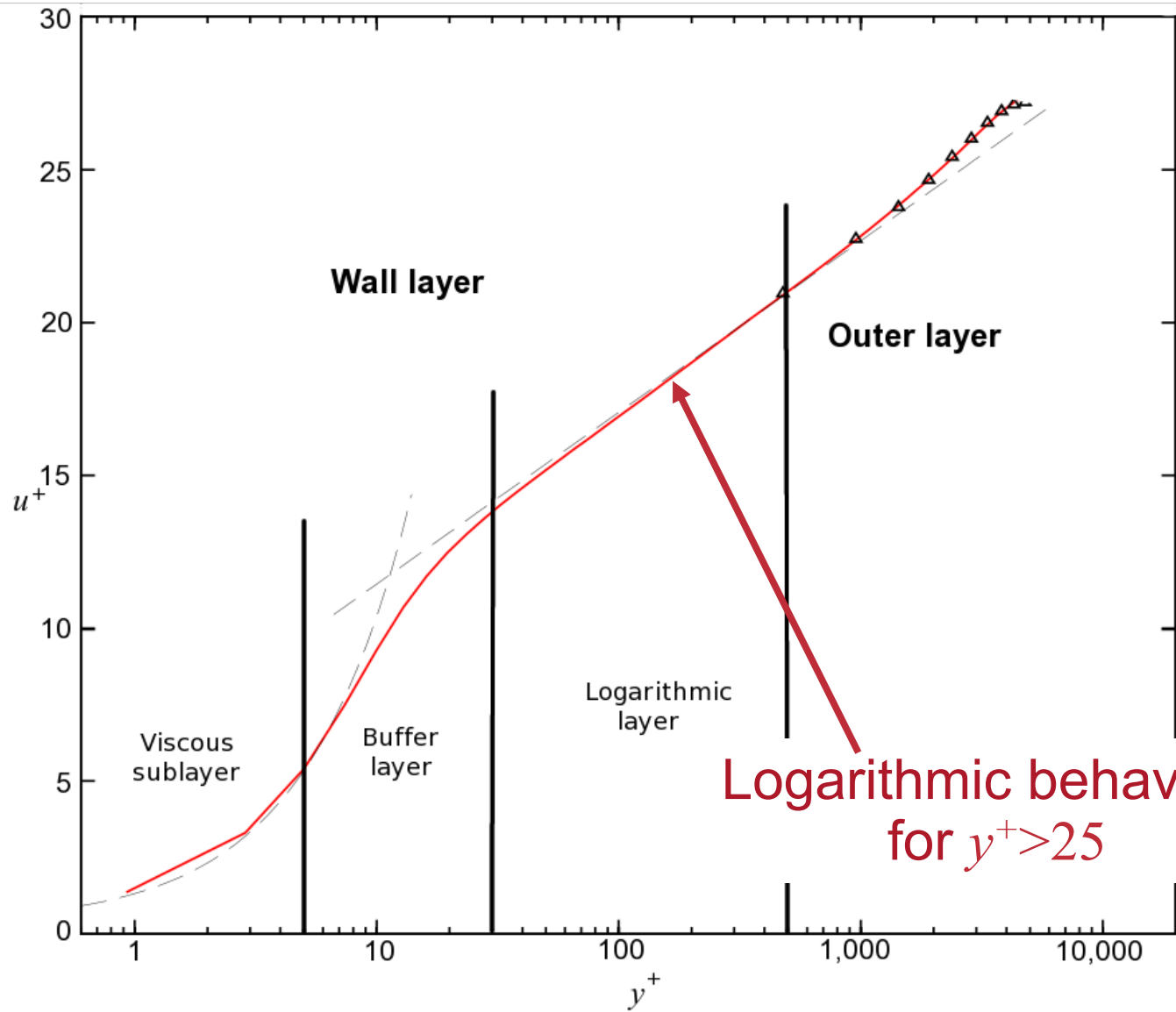
κ is a constant (von Kármán constant).

- Integrating...

$$u^+ = \frac{1}{\kappa} \log y^+ + B$$



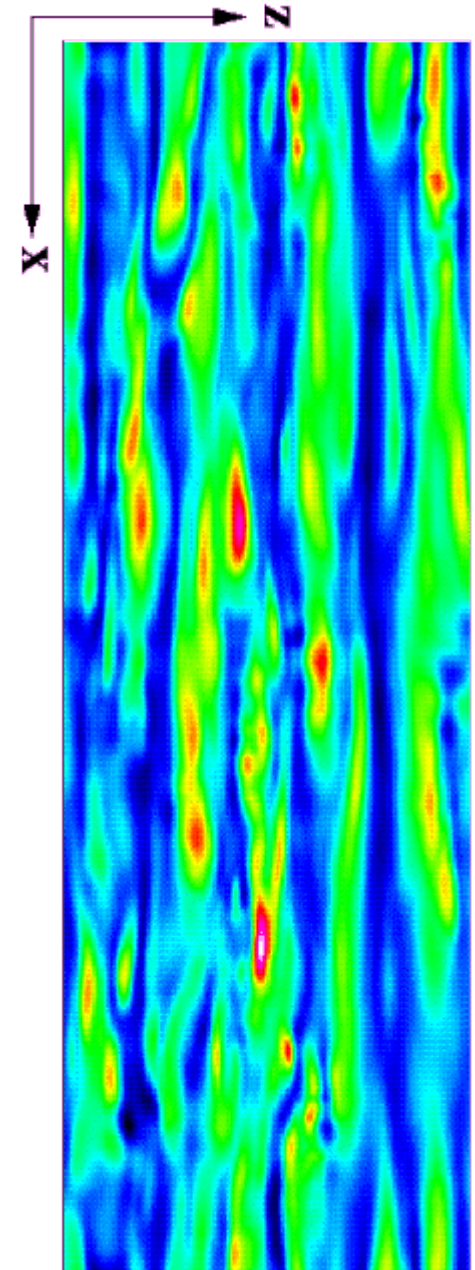
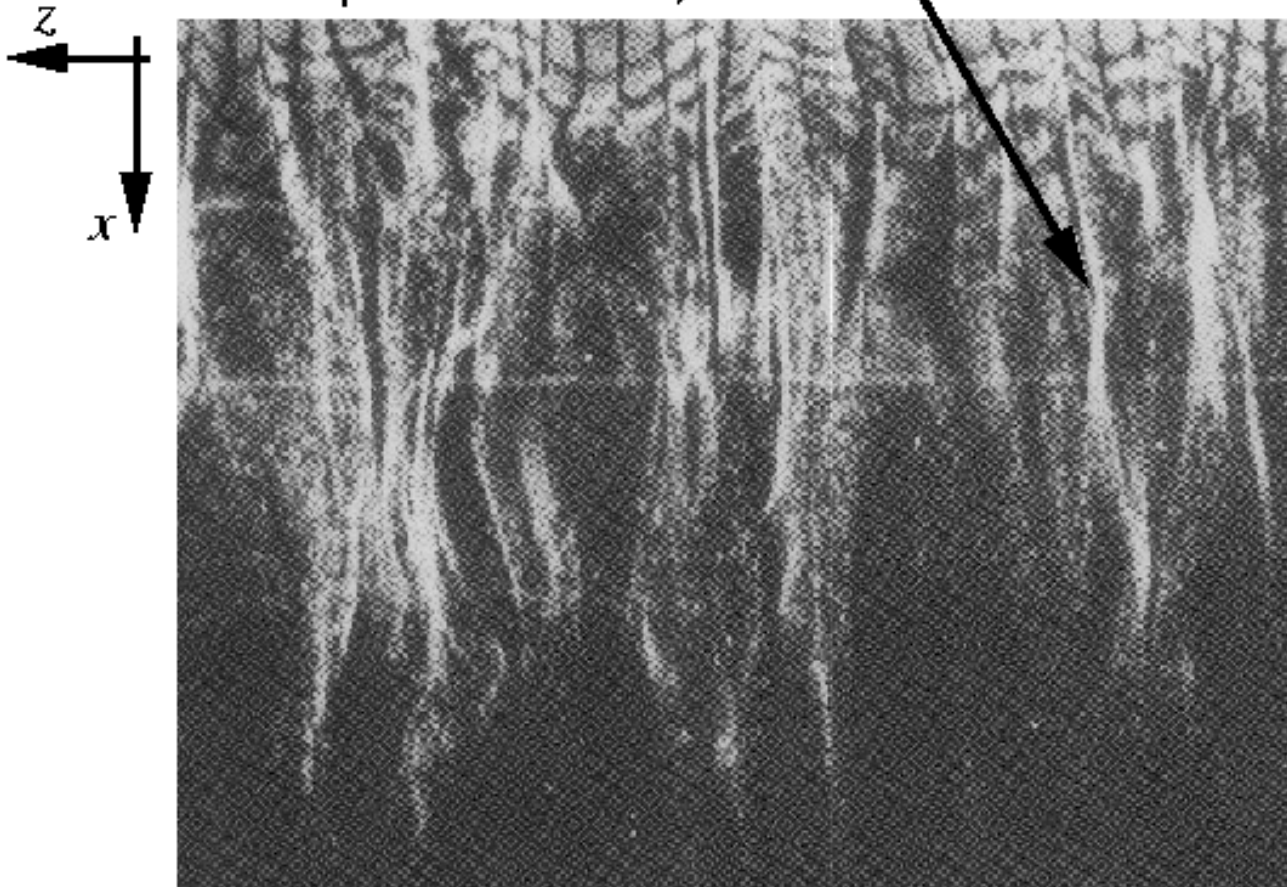
LOGARITHMIC LAYER





- Viscous sublayer: Alternating regions of high and low velocity (streaks).

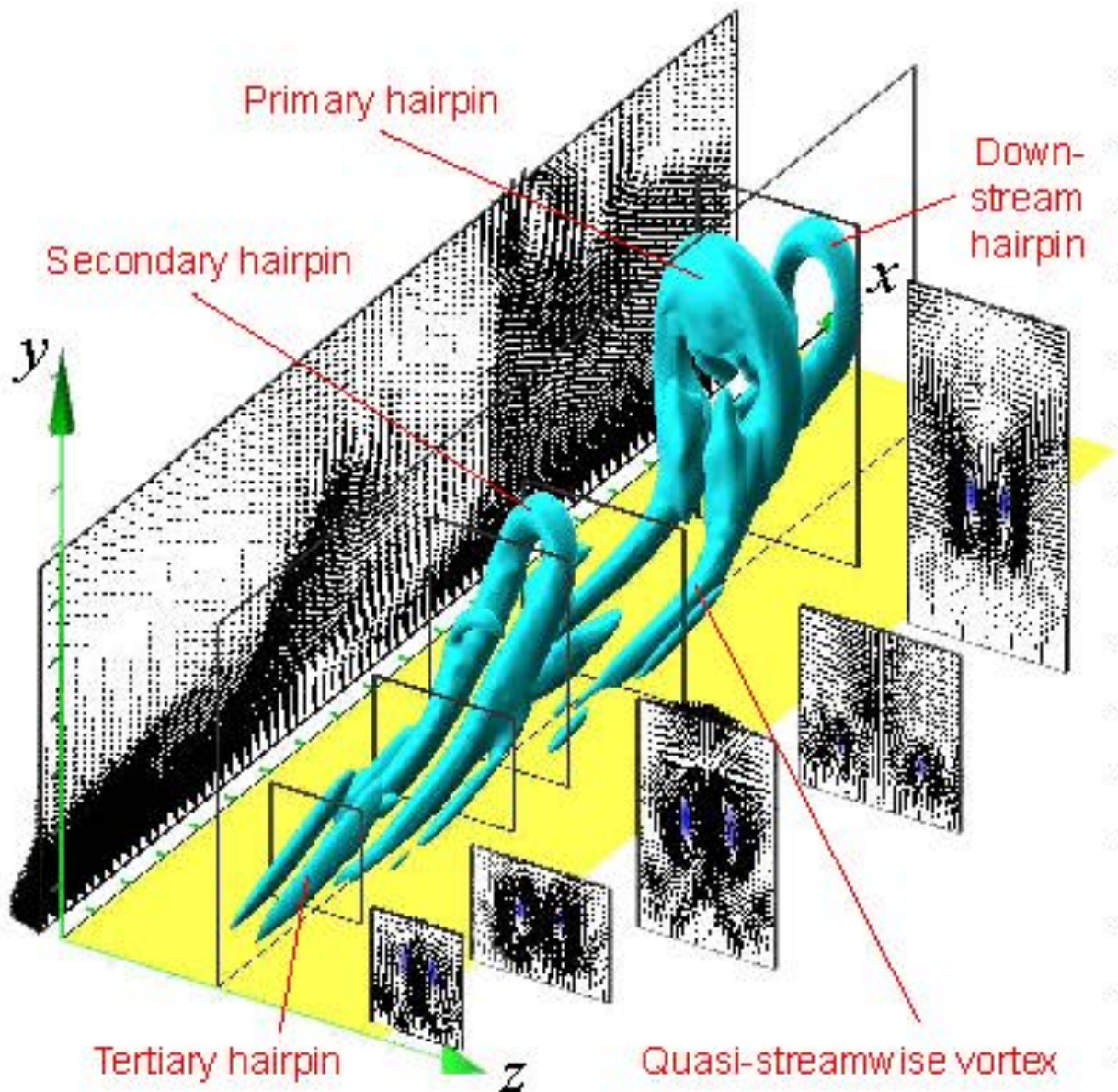
Low-speed streaks, scale λ^+



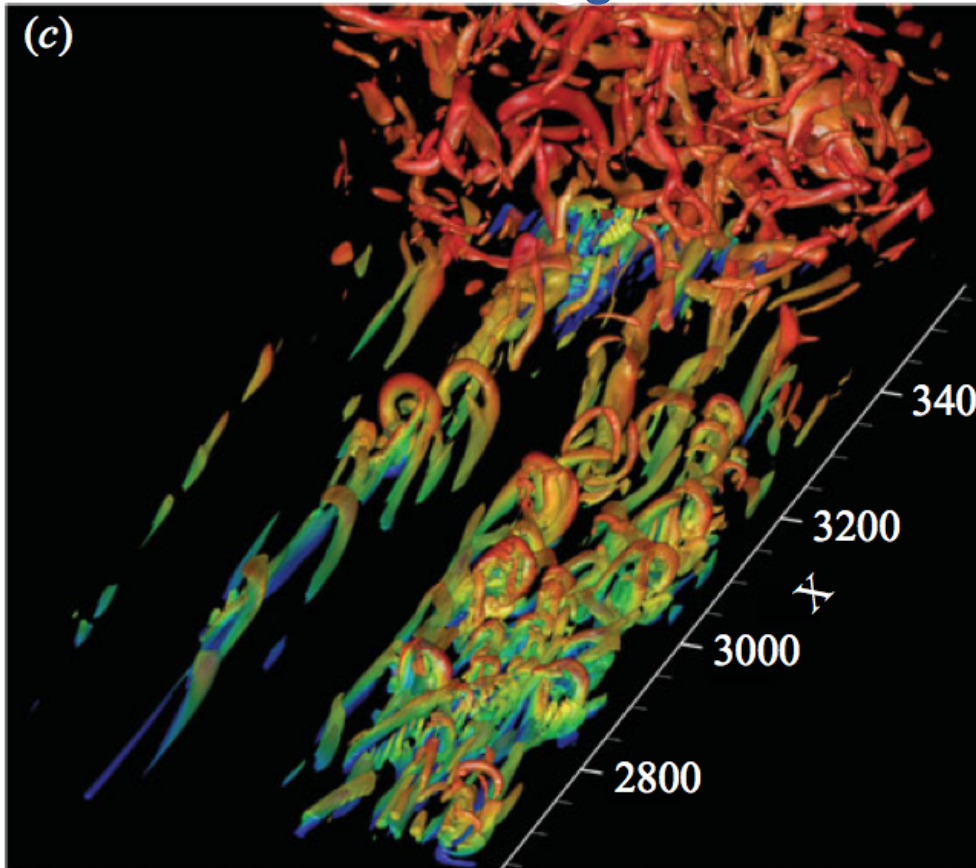


WALL-LAYER EDDIES

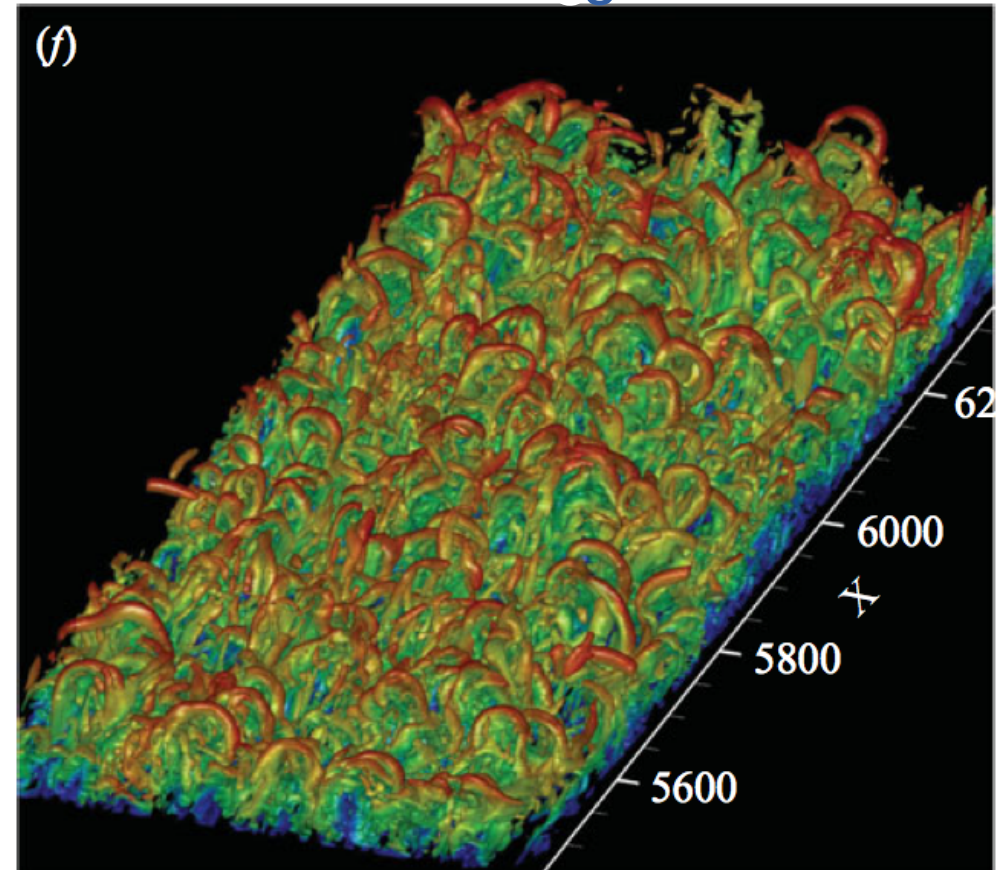
- Regions of the flow in which some variable is correlated with itself.
- Quasi-streamwise vortices (viscous sublayer).
- Hairpins (logarithmic layer).



transitional region



turbulent region



DNS of transition in a flat-plate boundary layer (Wu & Moin, 2009)

Turbulent eddies are visualized by the second invariant of the velocity gradient tensor and coloured base on the local value of the streamwise velocity.



- **Motivation:**
 - *What is turbulence?*
 - *Review of turbulence physics*
 - *Why simulations?*
 - *Methodologies*
 - *Resolution requirements*
- Governing equations for LES
- Boundary conditions
- Subgrid-scale modelling
- Validation of an LES
- Applications
- Challenges
- Conclusions



- Prediction and design tools are required
- Theory: **infeasible**
 - *Governing equations (Navier-Stokes equations) are highly non-linear.*

$$\frac{\partial \rho}{\partial t} + \frac{\partial}{\partial x_j} (\rho u_j) = 0$$

Conservation of mass

Conservation of momentum

$$\frac{\partial \rho u_i}{\partial t} + \frac{\partial}{\partial x_j} (\rho u_i u_j) = -\frac{\partial p}{\partial x_i} + \frac{\partial}{\partial x_j} \left[\mu \left(2S_{ij} - \frac{2}{3} \delta_{ij} S_{kk} \right) \right]$$

$$\frac{\partial e}{\partial t} + \frac{\partial}{\partial x_j} (u_j e) = \frac{\partial Q}{\partial t} - \frac{\partial}{\partial x_j} \left(\kappa \frac{\partial T}{\partial x_j} \right)$$

Conservation of energy

⇒ *Exact solutions cannot be found*



- Prediction and design tools are required
- Theory: **infeasible**
- Experiments
 - *Empirical design rules*
 - *Building and testing of prototypes*
 - *Iterative improvement of design*
 - *Issues: time, cost, accessibility of conditions, limited exploration*



- Prediction tools are required
- Theory: **infeasible**
- Experiments: **costly, incomplete**
- Numerical methods:
 - *Predict performance of proposed designs*
 - *Advantages: speed, novel designs, optimization*
 - *Issues: accuracy, reliability, level of description, cost*

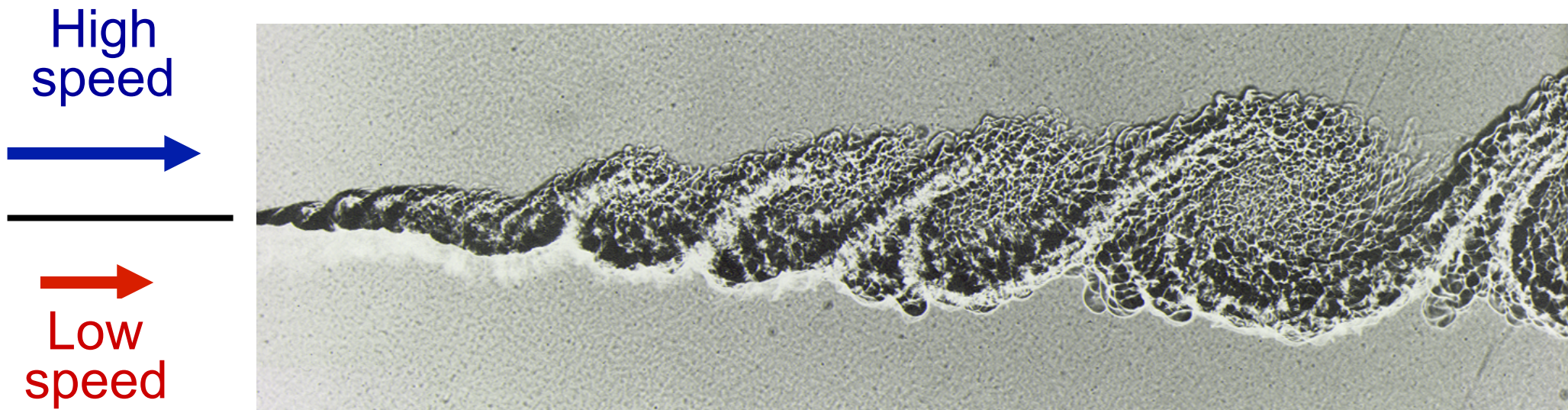


- Prediction tools are required
- Theory: **infeasible**
- Experiments: **costly, incomplete**
- Numerical methods: **possible but difficult**
- Trend: more use of computation, less testing.



- **Motivation:**
 - *What is turbulence?*
 - *Review of turbulence physics*
 - *Why simulations?*
 - *Methodologies*
 - *Resolution requirements*
- Governing equations for LES
- Boundary conditions
- Subfilter-scale modelling
- Validation of an LES
- Applications
- Challenges
- Conclusions

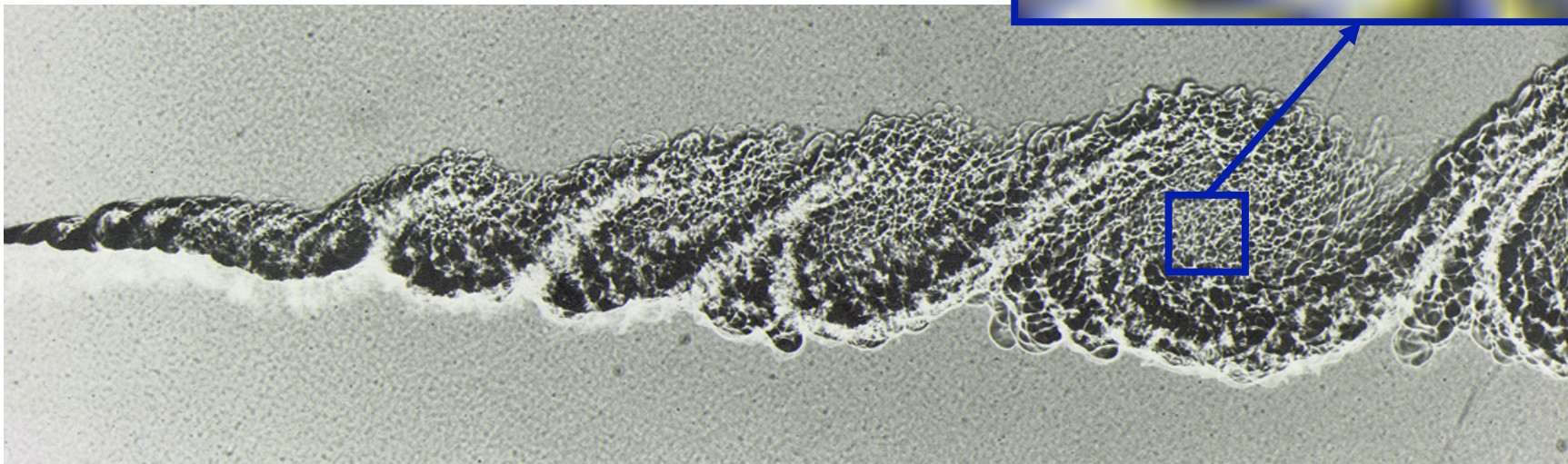
- Turbulent transport is due to the vortical motions (eddies).



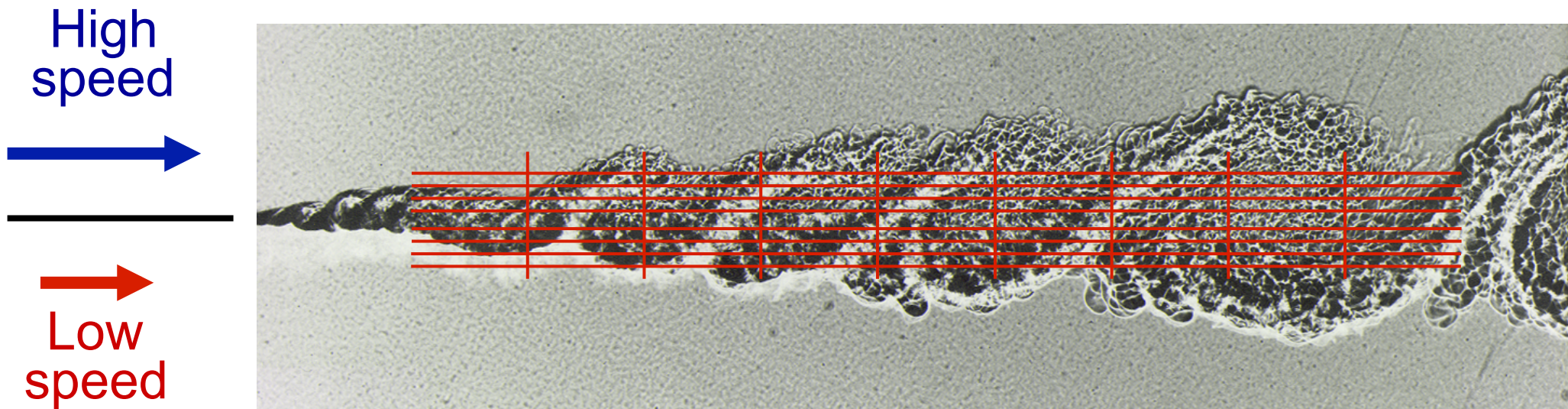
- Turbulent transport is due to the vortical motions (eddies).
- Solution methodologies:
 - *Full description of all eddies*
⇒ *Direct Numerical Simulation (DNS)*

High
speed
→

→
Low
speed



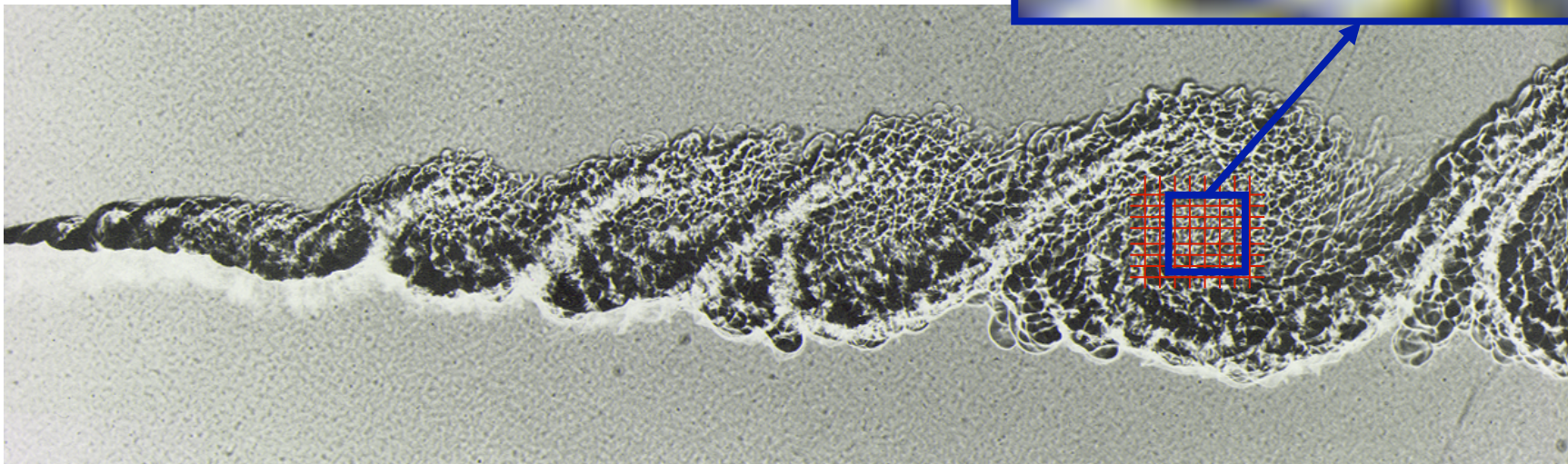
- Turbulent transport is due to the vortical motions (eddies).
- Solution methodologies:
 - *Full description of all eddies (DNS)*
 - *Statistical description of all eddies*
⇒ *Solution of the Reynolds-Averaged Navier-Stokes (RANS) equations*



- Turbulent transport is due to the vortical motions (eddies).
- Solution methodologies:
 - *Full description of all eddies (DNS)*
 - *Statistical description of all eddies (RANS)*
 - *Partial description of the eddies*
⇒ ***Large-Eddy Simulation (LES)***

High
speed
→

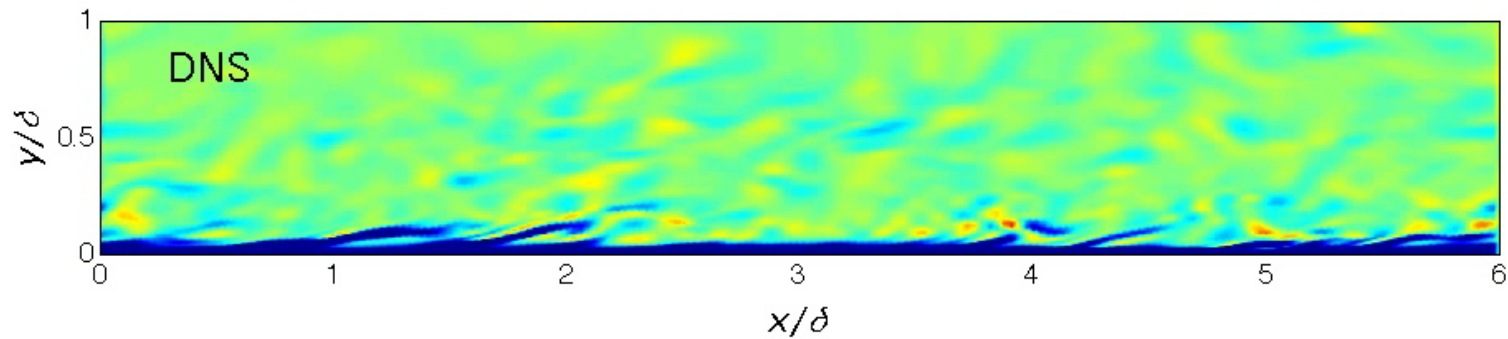
Low
speed
→



DIRECT NUMERICAL SIMULATION

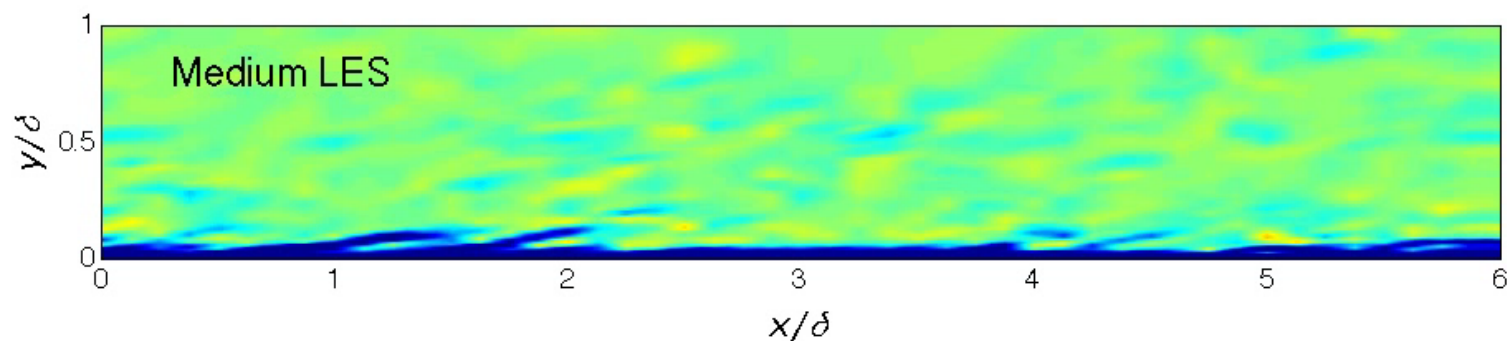
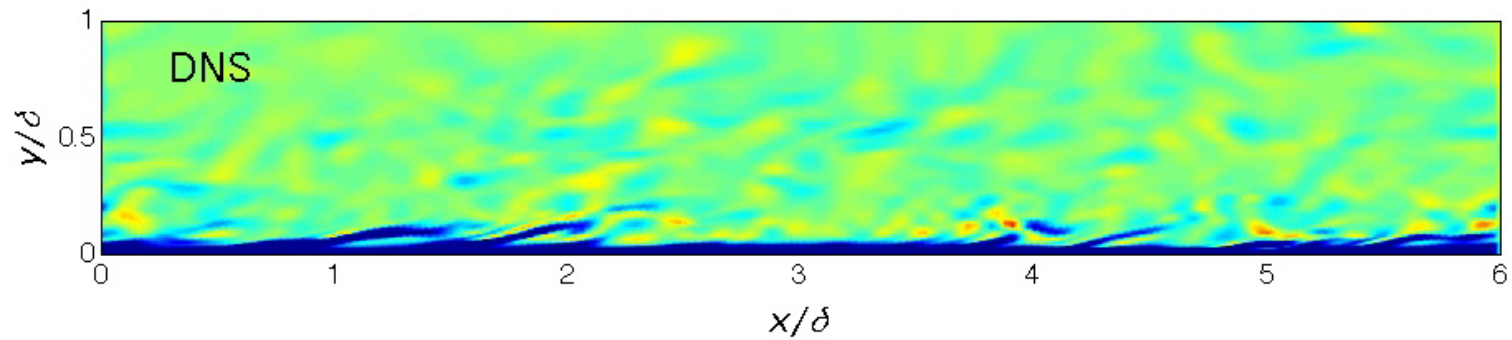
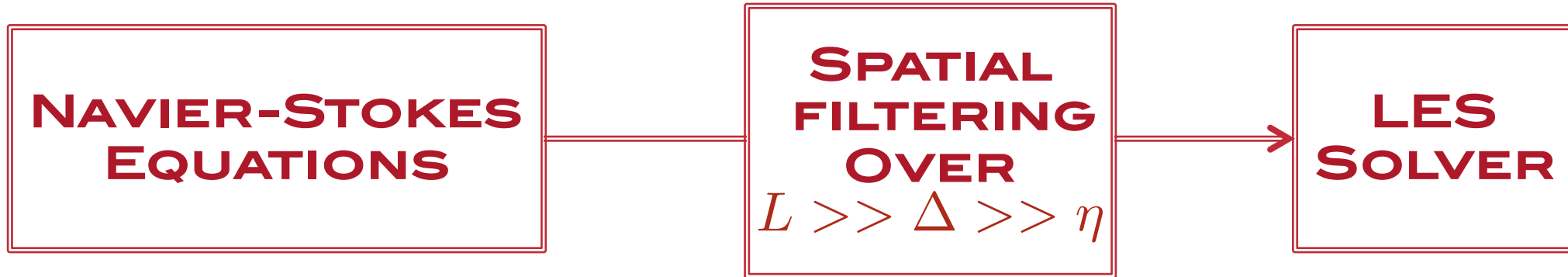
NAVIER-STOKES
EQUATIONS

DNS
SOLVER



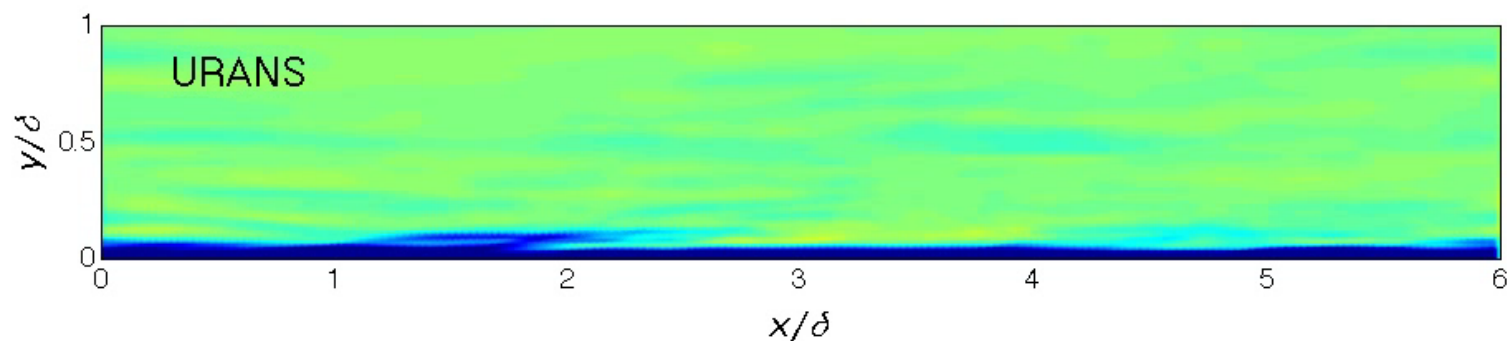
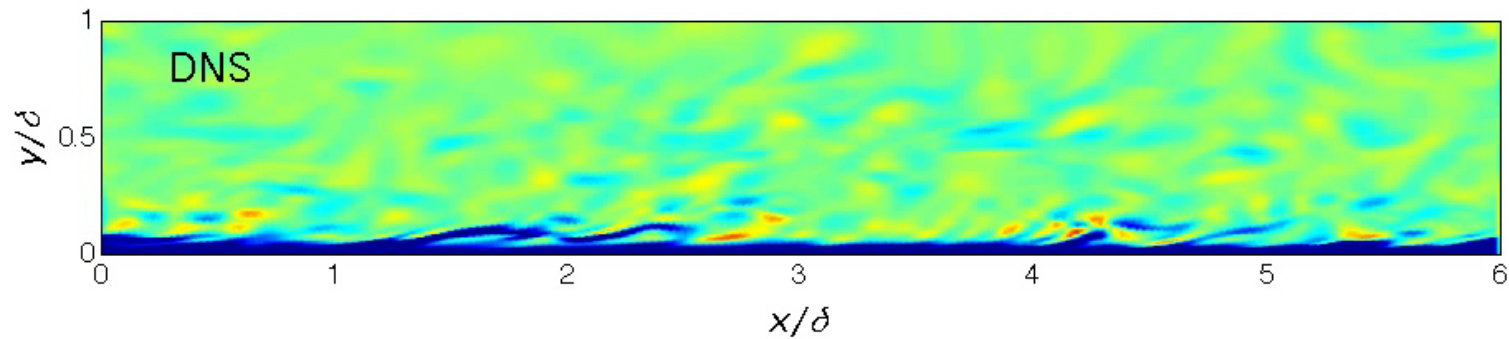
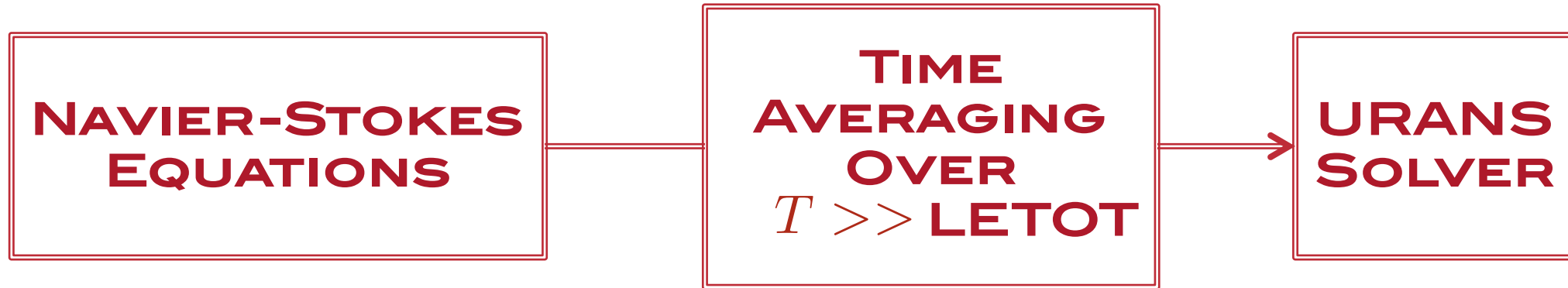
Vorticity contours, Channel flow, $Re=7000$

LARGE-EDDY SIMULATION



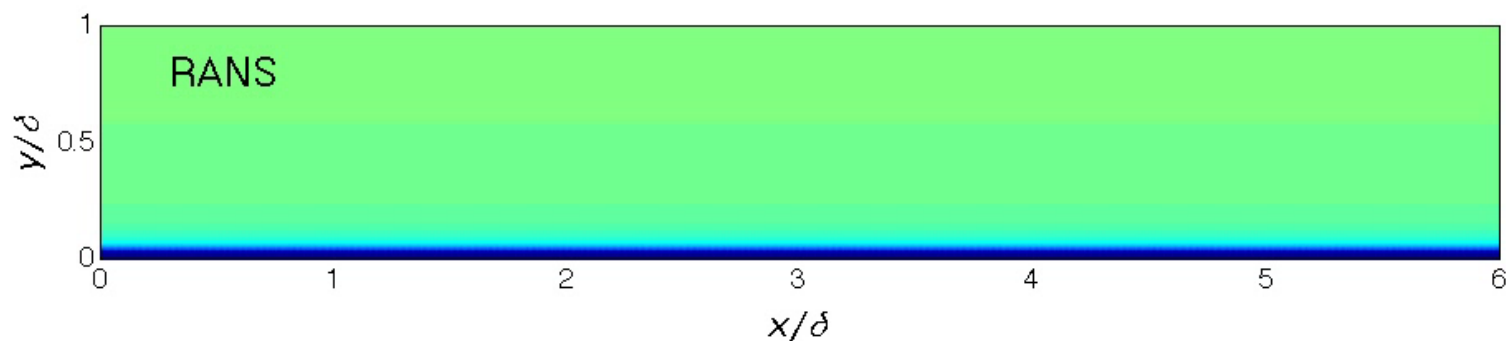
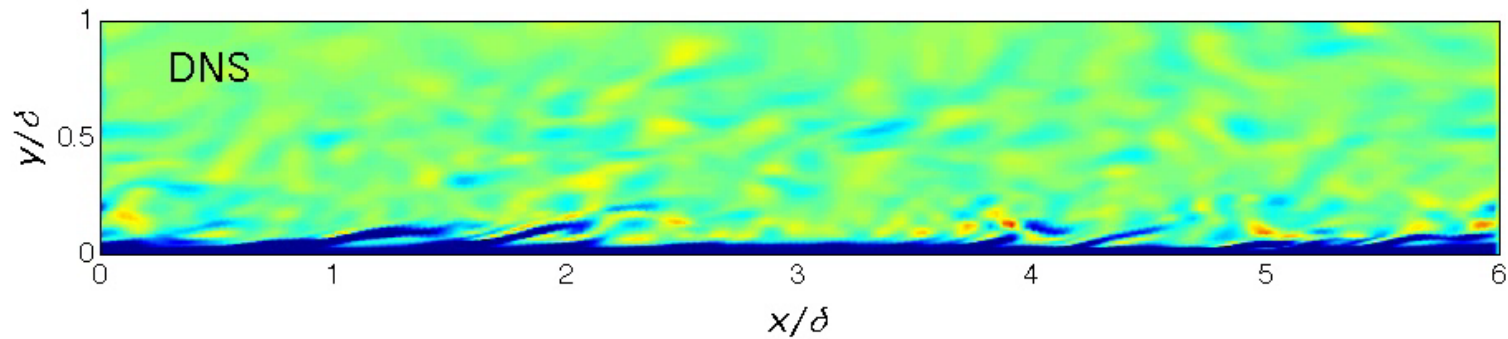
Vorticity contours, Channel flow, $Re=7000$

UNSTEADY REYNOLDS-AVERAGED NS SIMULATION



Vorticity contours, Channel flow, $Re=7000$

UNSTEADY REYNOLDS-AVERAGED NS SIMULATION

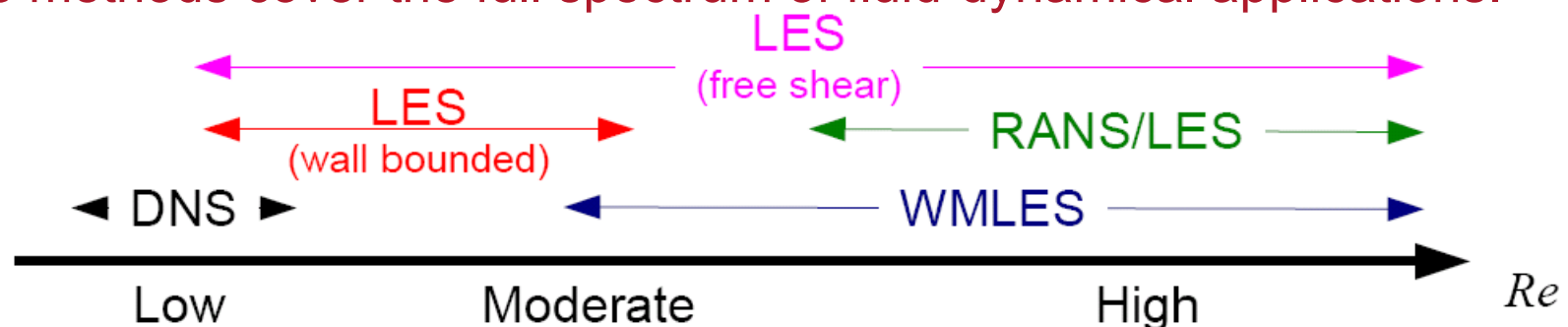


Vorticity contours, Channel flow, $Re=7000$

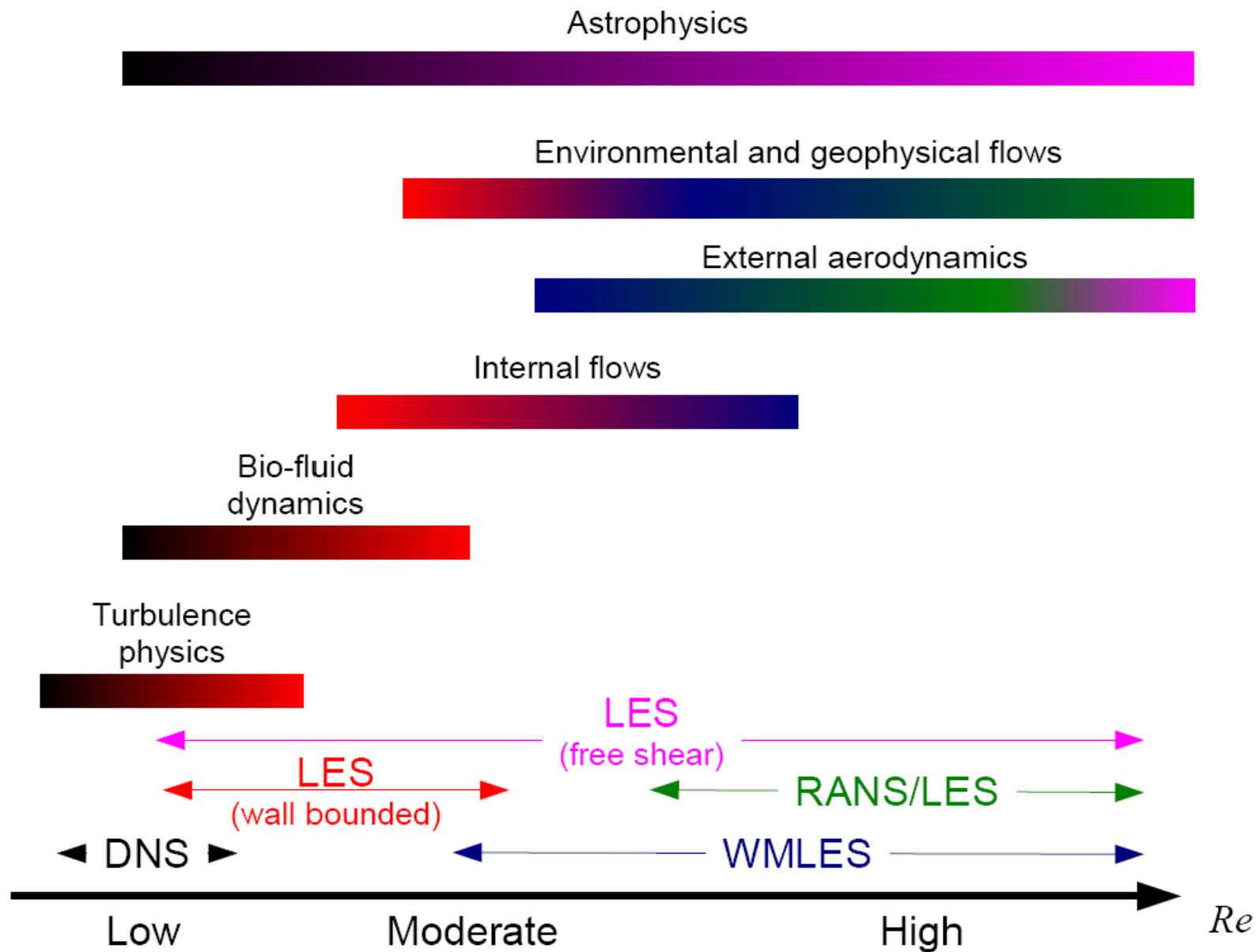


- Multi-point, non invasive information.
- Frequency and wave-number information.
- Little modelling \Rightarrow increased accuracy
- More faithful reproduction of the flow physics.

- Eddy-resolving methods for the numerical simulation of turbulent flows have resulted in
 - *Improved understanding of the flow physics*
 - *Novel flow-control ideas*
- Direct Numerical Simulations (DNS):
 - *No empiricism, Low Re , physics, simple geometry.*
- Large-Eddy Simulations (LES):
 - *Little empiricism, medium Re , physics.*
- Hybrid RANS/LES:
 - *Stronger empiricism, high Re , physics and design, complex geometry.*
- These methods cover the full spectrum of fluid-dynamical applications.



BENEFITS OF EDDY-RESOLVING METHODS

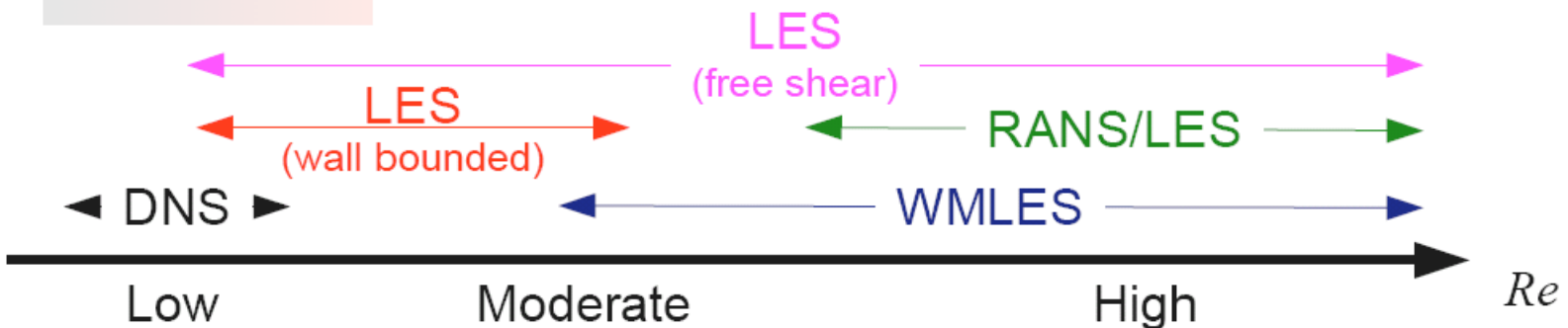


BENEFITS OF EDDY-RESOLVING METHODS

Astrophysics

- Range of scales: $10^{10} \Rightarrow$ DNS is infeasible.
- Wall effects are negligible \Rightarrow LES can be cost-effective at high Re
- Interaction of large-scale rotation with turbulent eddies.
- Lorentz forces are an additional mechanism that affects the turbulence dynamics.

□ *Stiffness*



- Eddy-resolving methods can account directly for

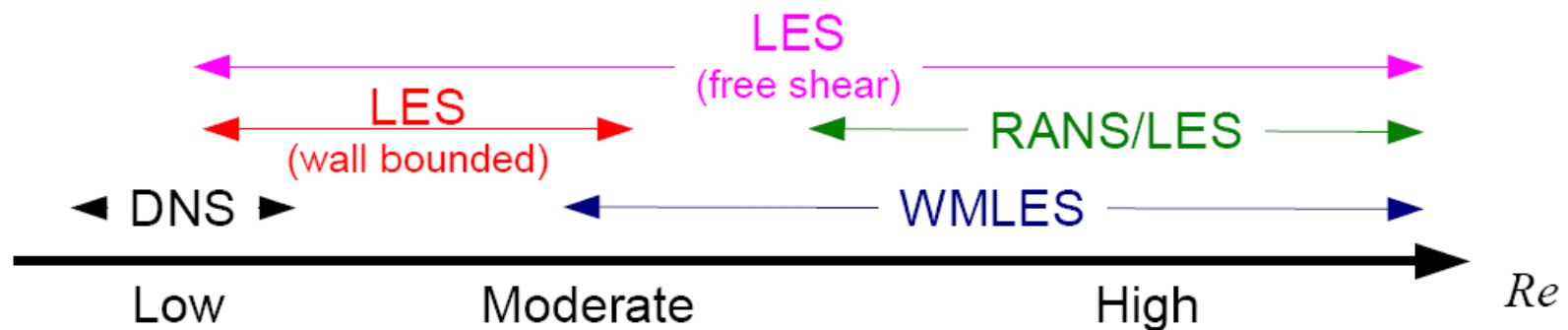
- *Large-scale mixing*
- *Stratification*
- *Rotation effects*

Environmental and geophysical flows

External aerodynamics

- Possible applications:

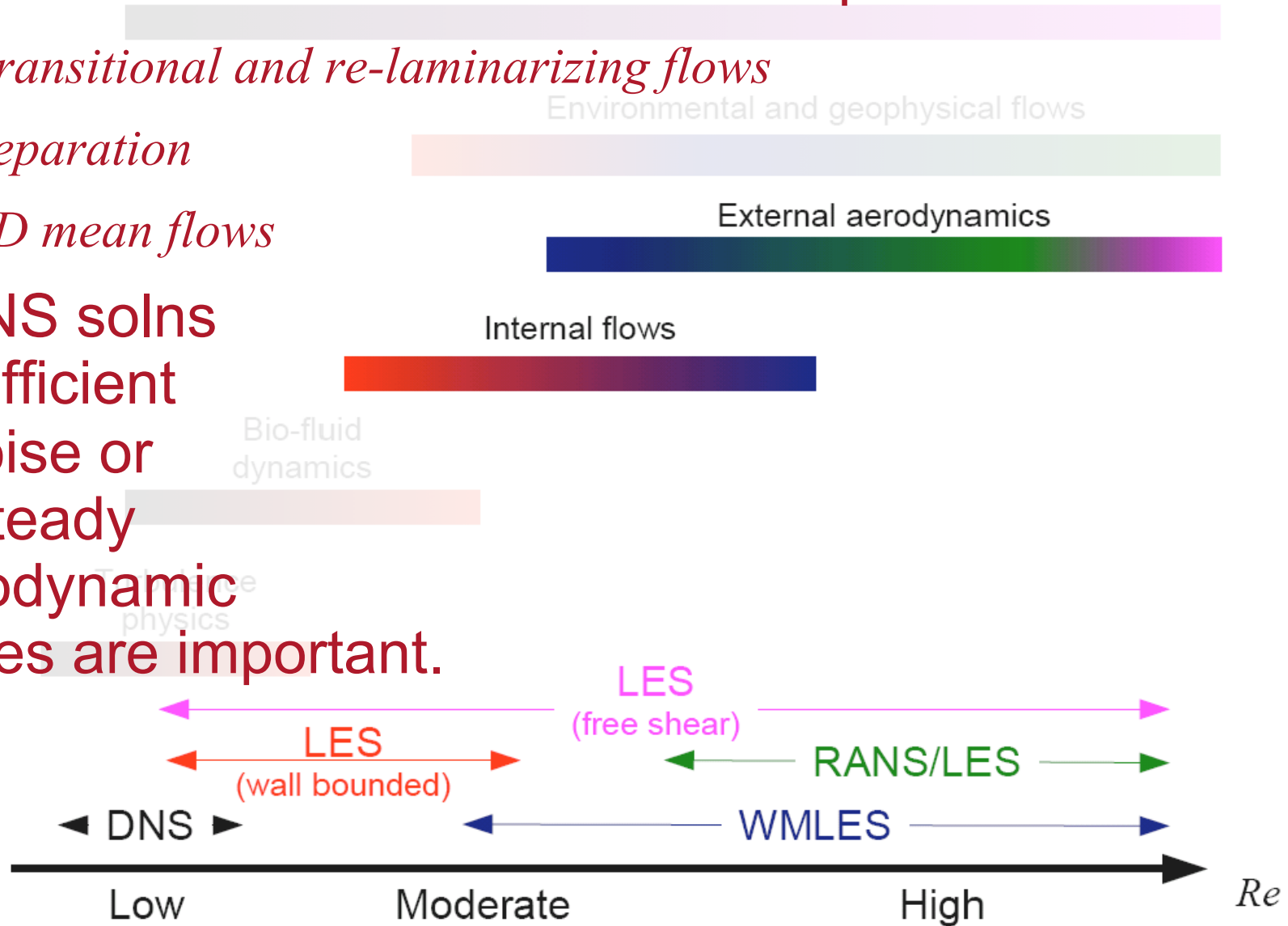
- *Sediment transport*
- *Coastal oceanography*
- *Pollutant dispersion*



- More accurate calculation of non-equilibrium turbulence:

- *Transitional and re-laminarizing flows*
- *Separation*
- *3D mean flows*

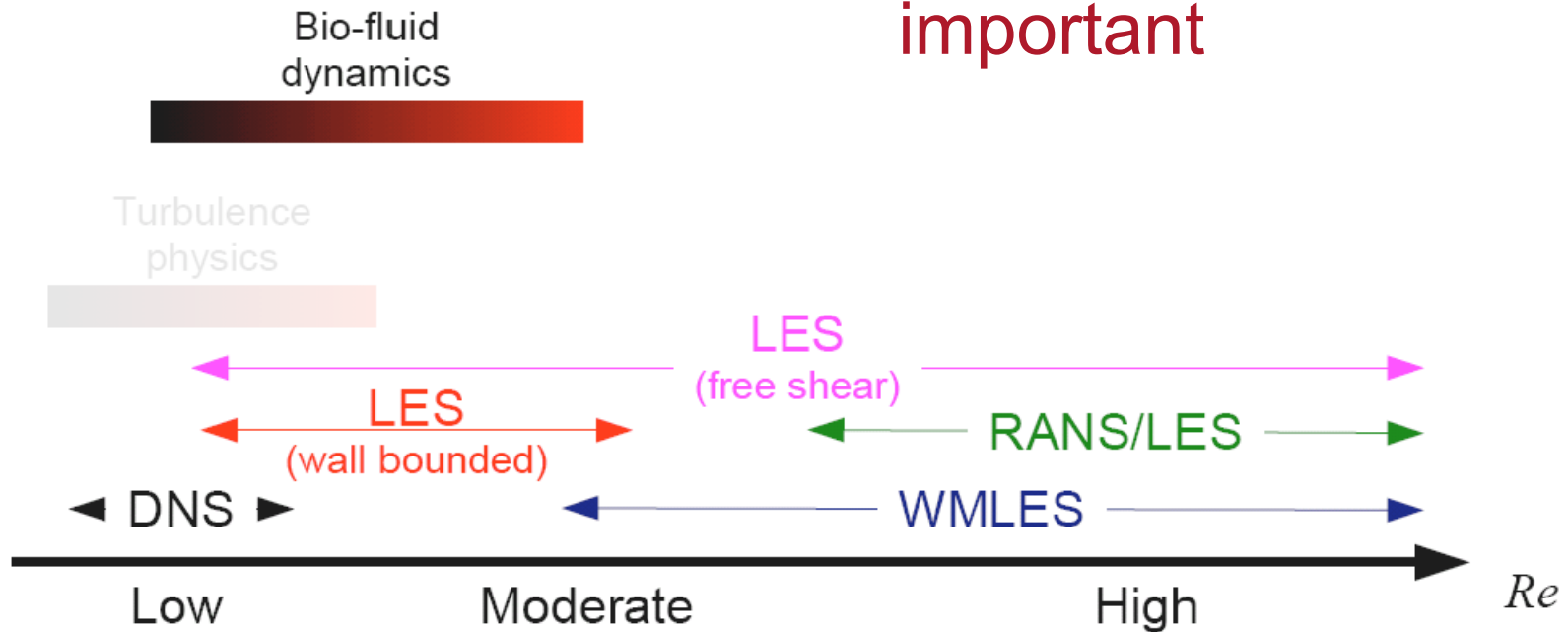
- RANS solns insufficient if noise or unsteady aerodynamic forces are important.



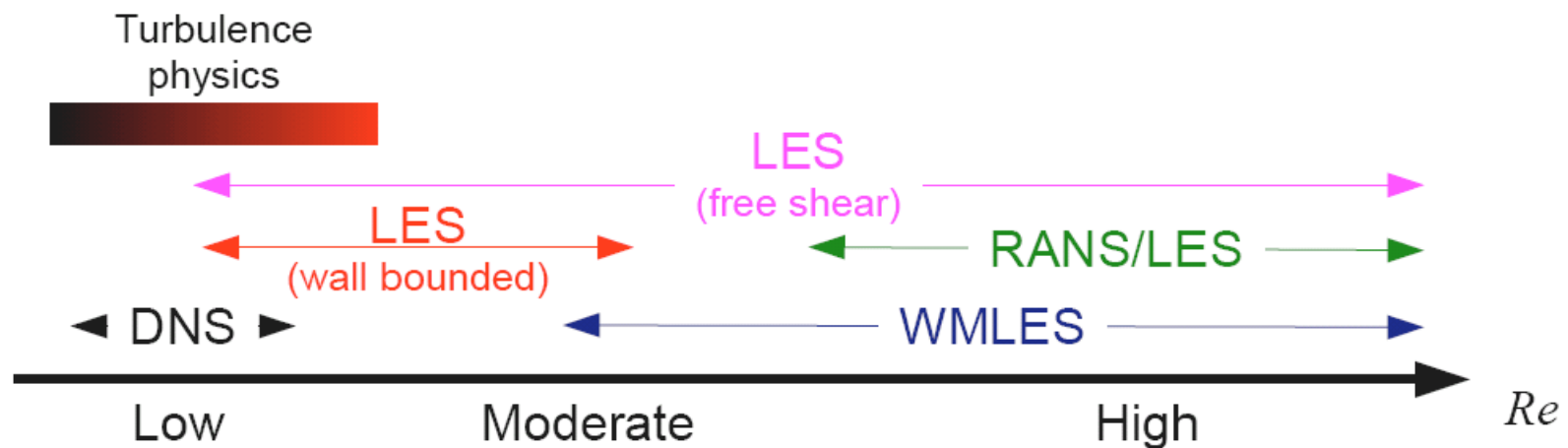


BENEFITS OF EDDY-RESOLVING METHODS

- Moderate Re , complex geometries
- Highly 3D, unsteady, transitional flows.
 - *RANS methods inaccurate*
- Oscillating mean flows
 - *Alternating favorable and adverse ∇P*
 - ⇒ *Relaminarization and transition*
- Fluid-structure interactions may be important



- Ideal method to test theories
 - *Controlled boundary and initial conditions.*
 - *Possibility to conduct innovative thought experiments*
- Data can be used to test lower-level models
 - *Understanding of the physical phenomena involved*
 - *Determination of constants*
 - *Term-by-term model evaluation*



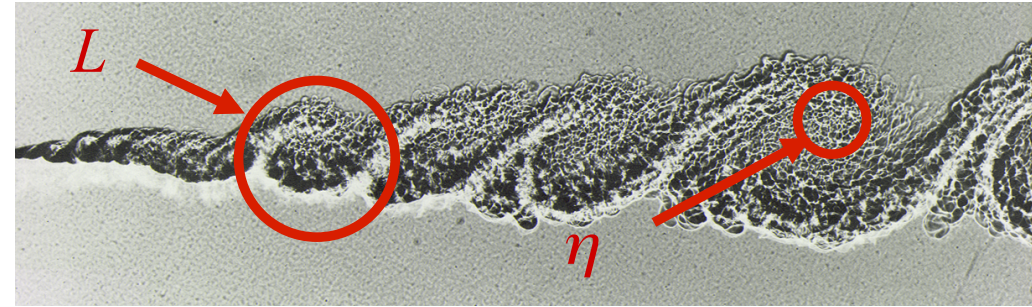


- **Motivation:**
 - *What is turbulence?*
 - *Review of turbulence physics*
 - *Why simulations?*
 - *Methodologies*
 - *Resolution requirements*
- Governing equations for LES
- Boundary conditions
- Subfilter-scale modelling
- Validation of an LES
- Applications
- Challenges
- Conclusions



RESOLUTION REQUIREMENTS (DNS)

- The computational domain size is of the order of the integral scale L .



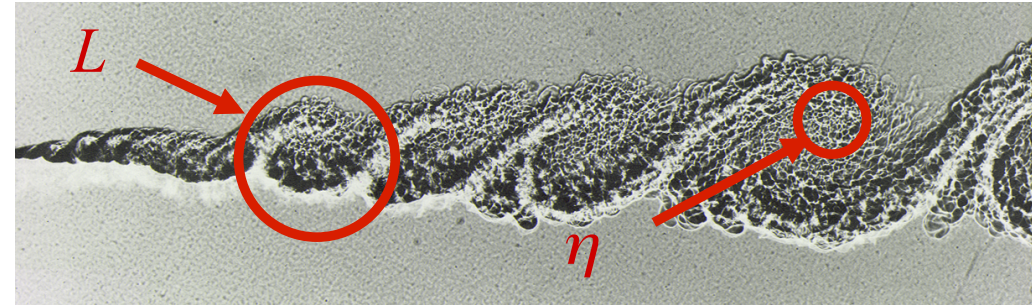
- The grid size must be of the order of the Kolmogorov length η

$$N_x N_y N_z \sim (L/\eta)^3 \sim Re^{9/4}$$



RESOLUTION REQUIREMENTS (DNS)

- The computational domain size is of the order of the integral scale L .



- The grid size must be of the order of the Kolmogorov length η

$$N_x N_y N_z \sim (L/\eta)^3 \sim Re^{9/4}$$

- The equations must be integrated for a time of the order of the integral time scale T .

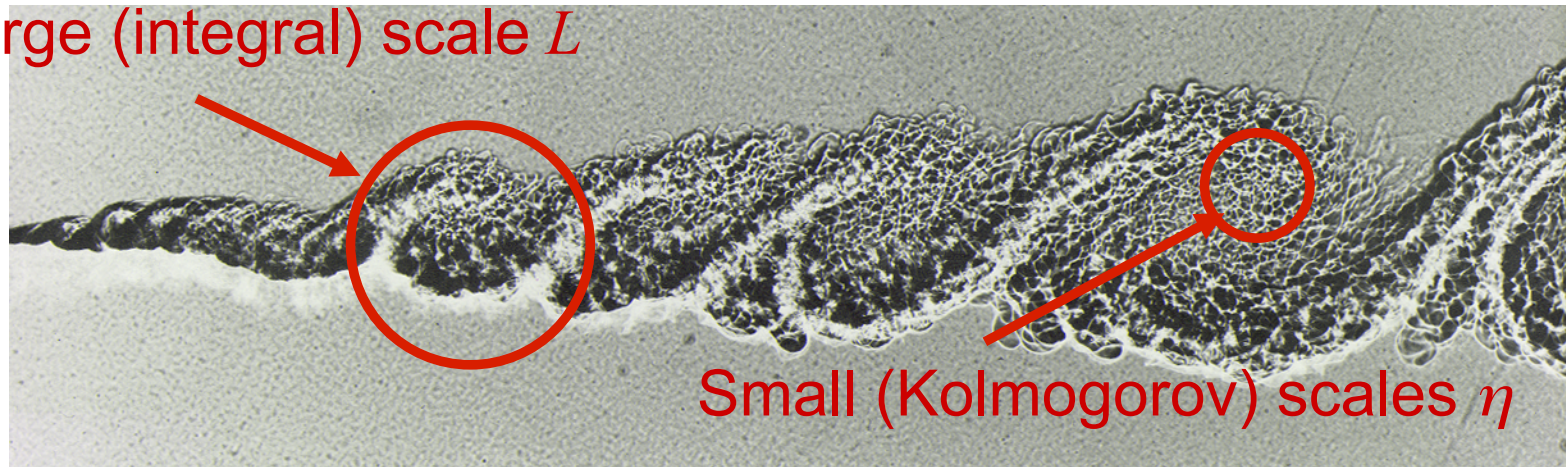
- The time-step $\Delta T \propto$ the grid size (CFL condition) or \propto the Kolmogorov time scale τ_η

$$N_t \sim T/\Delta t \sim L/\eta \sim Re^{3/4}$$

$$N_t \sim T/\tau \sim Re^{1/2}$$

$$\Rightarrow N_x N_y N_z \times N_t \sim Re^{11/4} \rightarrow Re^3$$

Large (integral) scale L



Small (Kolmogorov) scales η

- Cost $N_x N_y N_z \times N_t \sim Re^{11/4} \rightarrow Re^3$
- $Re = 10^4 \Rightarrow o(10^3)$ CPU hours, Gflop machine
- $Re = 10^9 \Rightarrow o(10^{11})$ CPU hours, Tflop machine



- Velocity decomposed into large-scale (resolved) and subfilter-scale (unresolved) parts.
 - *The large scales (\sim Integral scale, L), which depend on the boundary conditions (i.e., are flow dependent) are computed.*
 - *The small scales, which are more universal (less dependent on boundary conditions) are modelled.*
 - *Large scales contribute most of the Reynolds stresses.*



RESOLUTION REQUIREMENTS (LES)

- Only resolve the integral length scale (energy-containing eddies)
 - The integral scale varies with Re .
 - Cost scales with the Reynolds number
- ⇒ High Re calculations are possible.



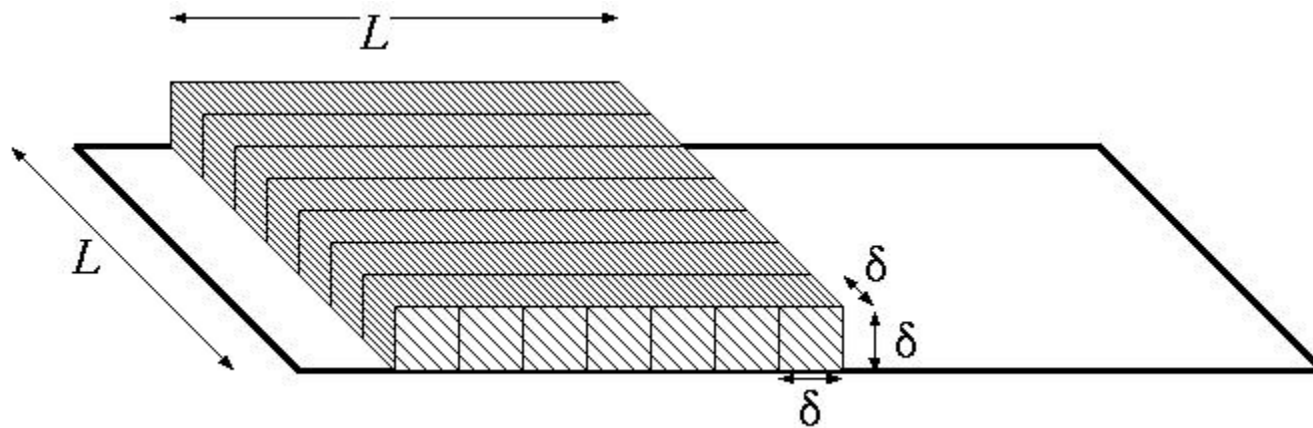
RESOLUTION REQUIREMENTS (WALL-BOUNDED FLOWS)

- Outer layer:
 - *Need 20-30 points in each direction to resolve an integral scale δ .*
 - *Cover the body of dimensions L^2 with a layer of cubes of side δ .*

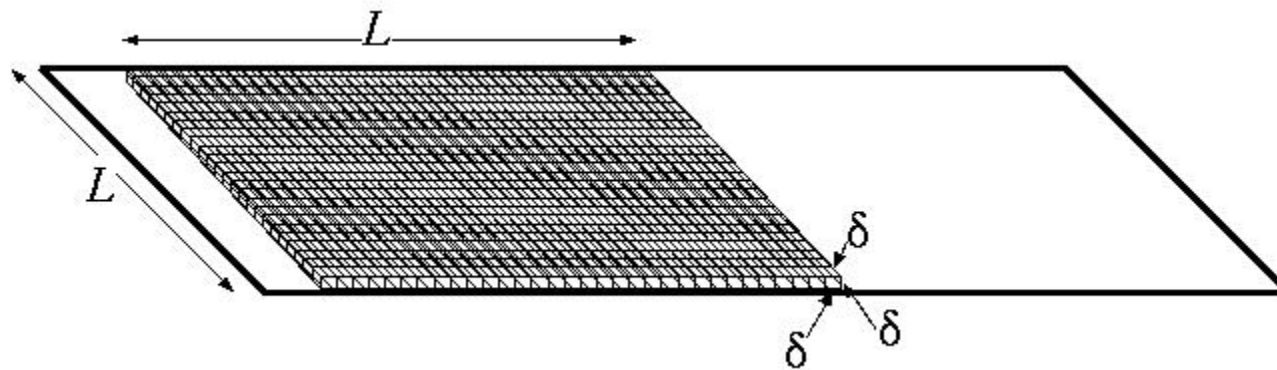
$$N_x N_y N_z \sim N_{cubes} \sim (L/\delta)^2$$

RESOLUTION REQUIREMENTS (WALL-BOUNDED FLOWS)

- Outer layer:
 - L/δ varies with Re



Low Re



High Re

RESOLUTION REQUIREMENTS (WALL-BOUNDED FLOWS)

- Outer layer:

- *L/δ varies with Re slowly (generally $\sim Re^{0.2}$)*

$$N_x N_y N_z \sim N_{cubes} \sim (L/\delta)^2 \sim Re^{0.4}$$

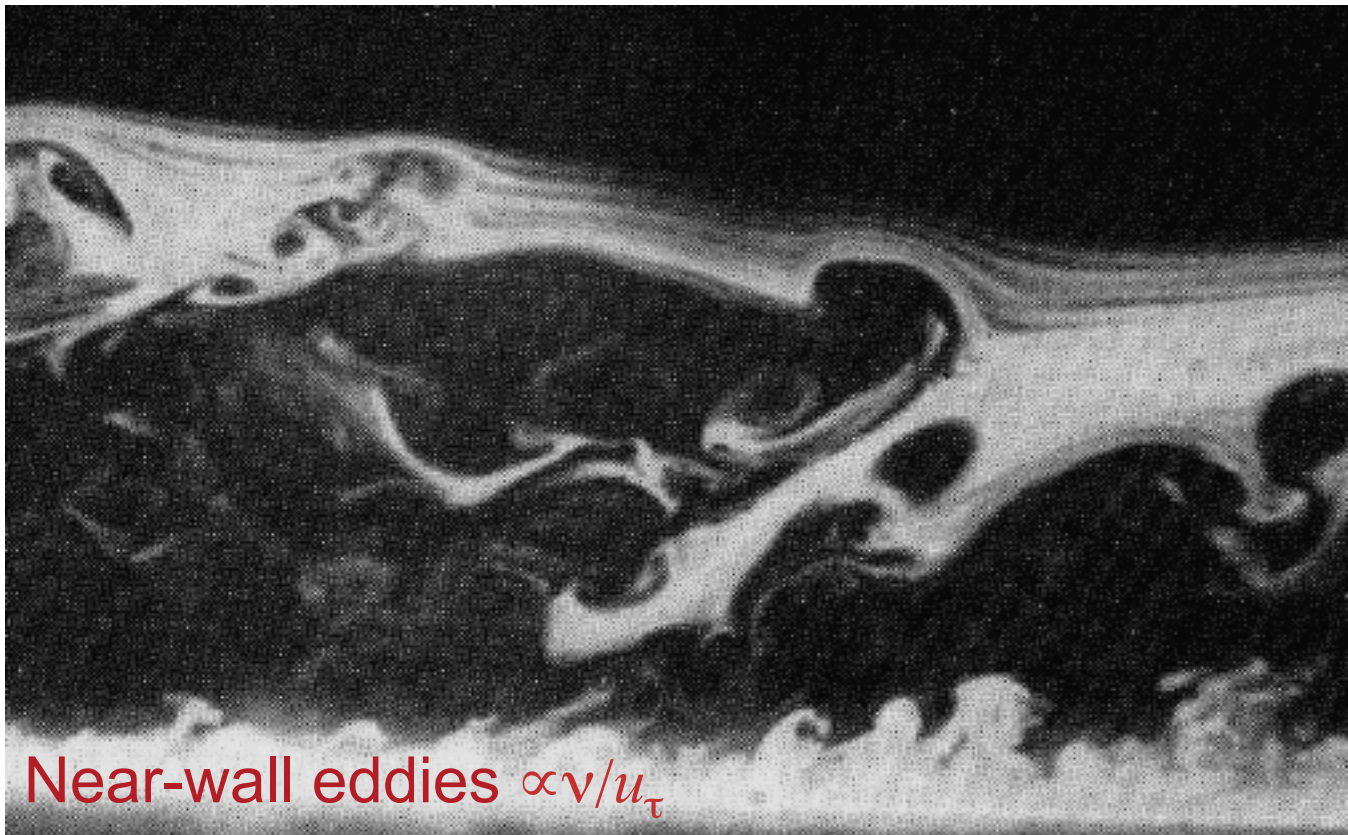
- *Total cost of the calculation:*

$$N_x N_y N_z \times N_t \sim Re^{0.6}$$



RESOLUTION REQUIREMENTS (WALL-BOUNDED FLOWS)

- Outer layer: only scales of order δ must be resolved.
 $\Rightarrow \text{Cost} \sim Re^{0.6}$
- Inner layer: near-wall eddies must be resolved.



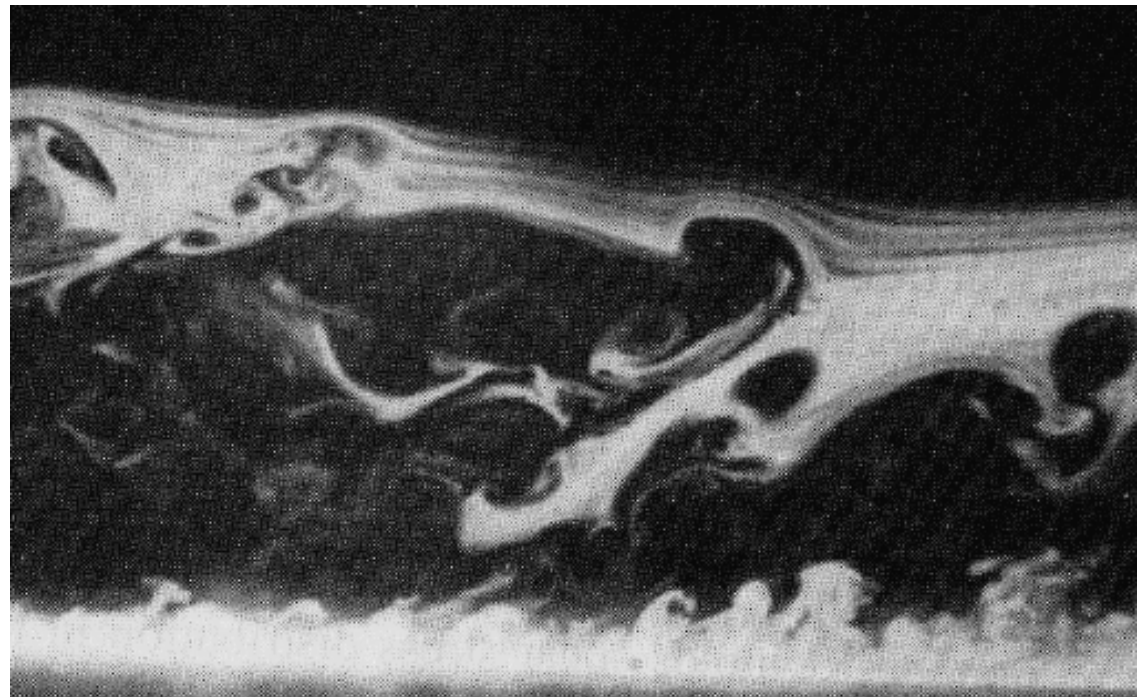
Near-wall eddies $\propto \nu/u_\tau$



RESOLUTION REQUIREMENTS (WALL-BOUNDED FLOWS)

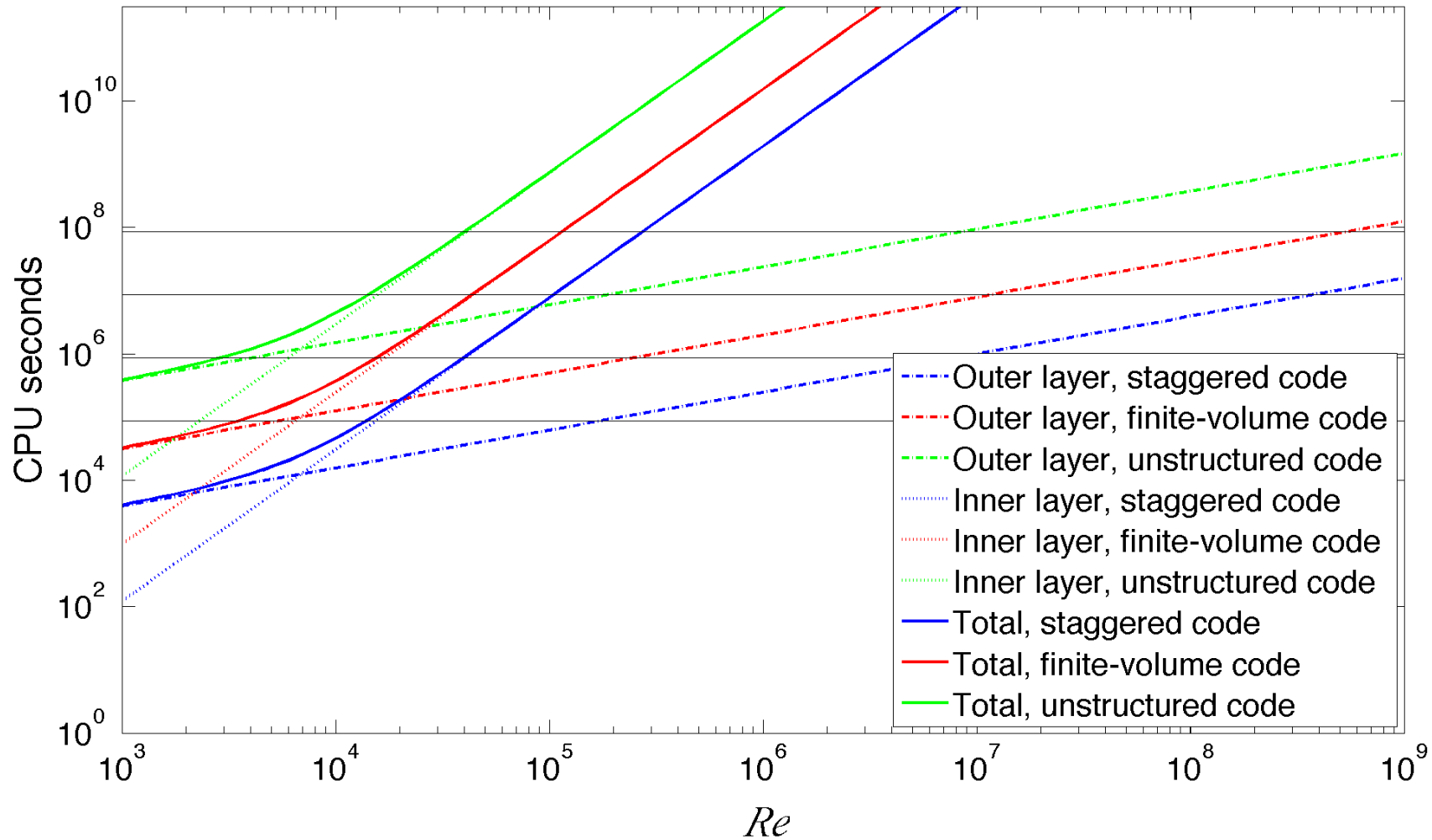
- Outer layer: only scales of order δ must be resolved.
 $\Rightarrow \text{Cost} \sim Re^{0.6}$
- Inner layer: near-wall eddies must be resolved.
 - *Grid must scale in wall units*
 - $N_x N_y N_z \propto Re^{1.8}$

$$\Rightarrow \text{Cost} \propto Re^{2.4}$$



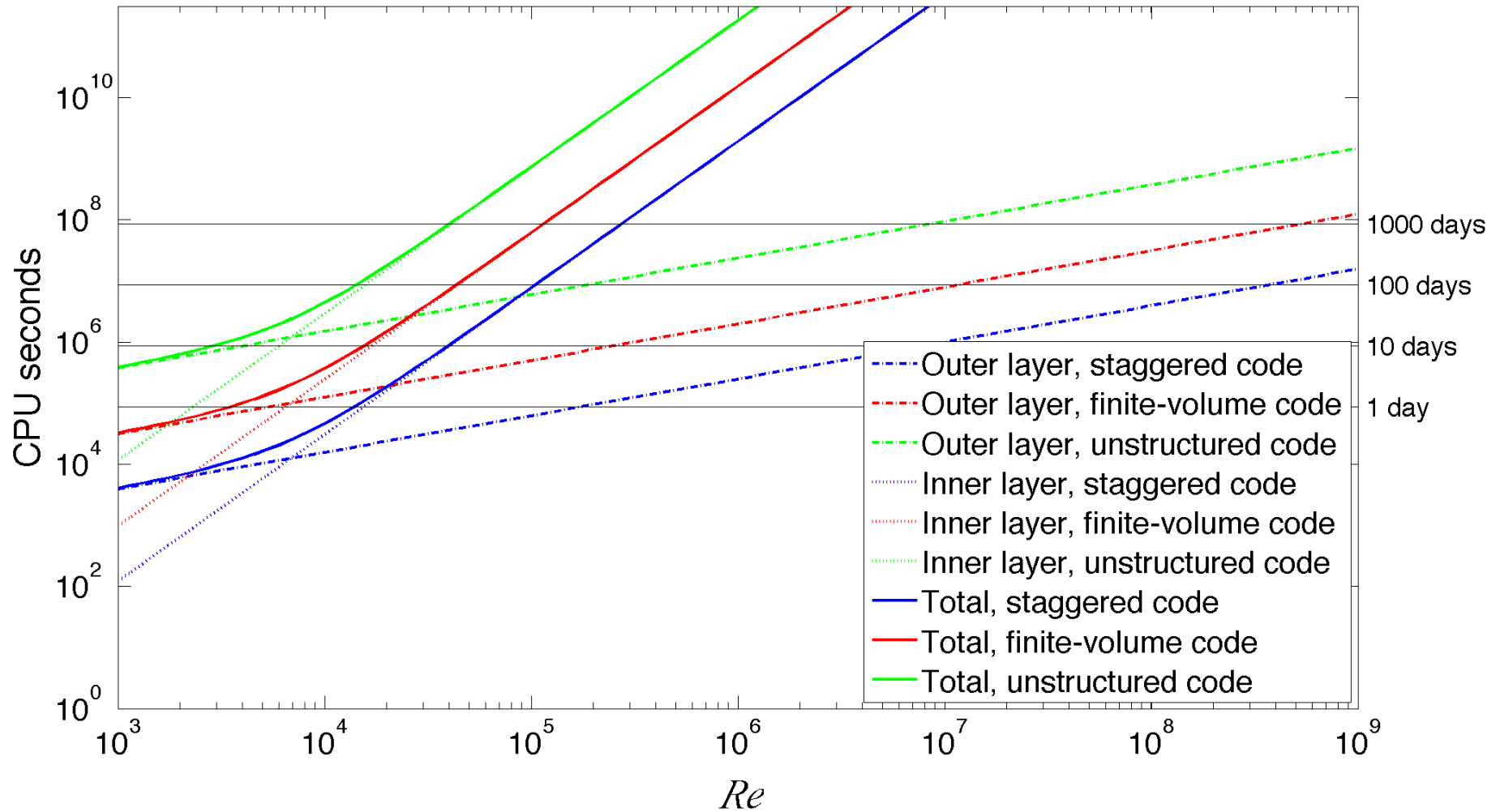


RESOLUTION REQUIREMENTS (WALL-BOUNDED FLOWS)



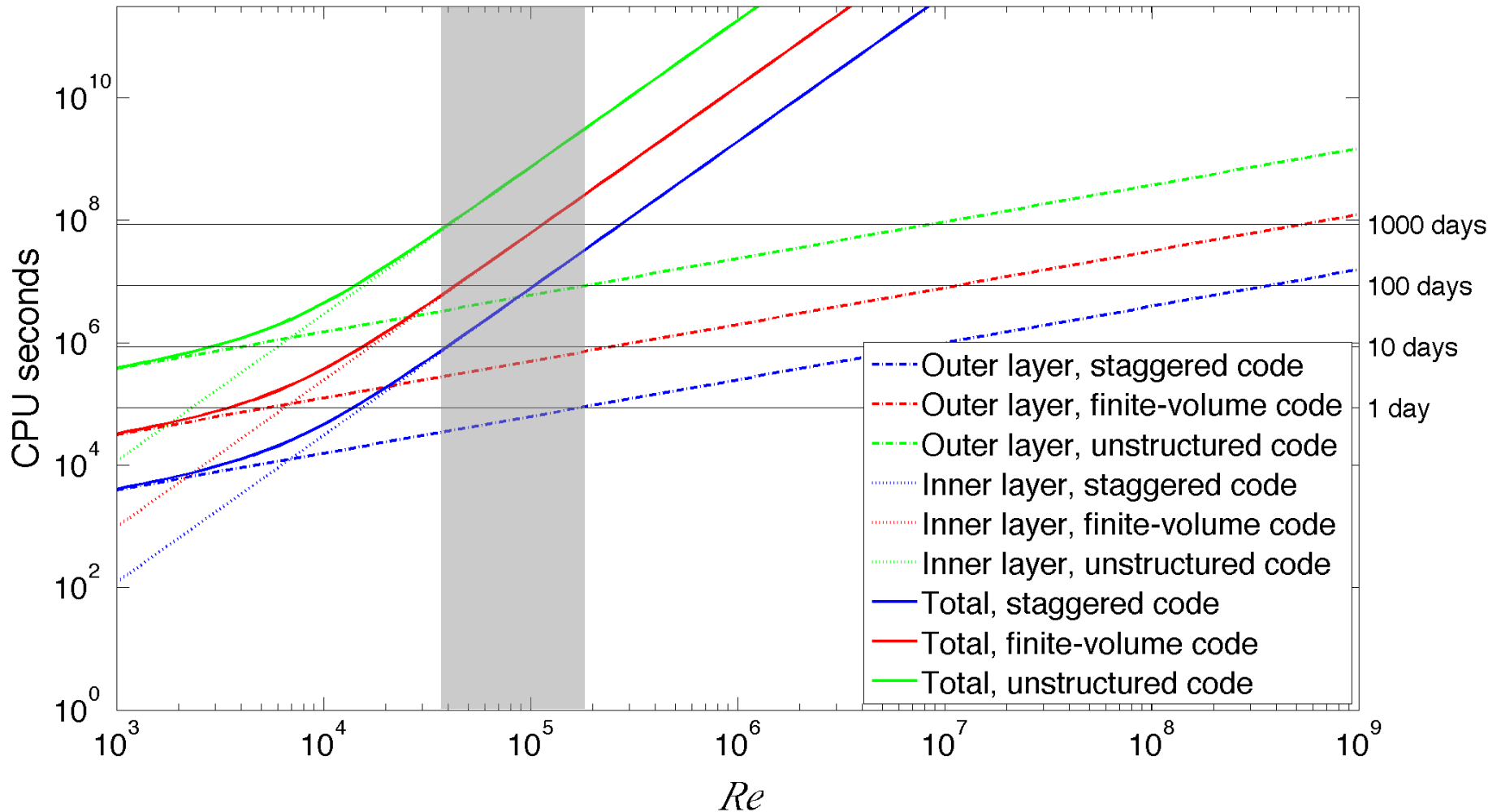


RESOLUTION REQUIREMENTS (WALL-BOUNDED FLOWS)





RESOLUTION REQUIREMENTS (WALL-BOUNDED FLOWS)



⇒ At high Re , throughput times are too long for design applications



WHY NOT LES AND DNS?

- Small scales need to be resolved
⇒ fine grid.
- Averaging of the results, not of the equations
⇒ long integration times.
- Vortex dynamics are important, vortex stretching must be accounted for
⇒ always 3D.

⇒ Require large computational resources



- Motivation:
- Simulation methodologies
- **Governing equations**
- Numerical methods
- Subfilter-scale modelling
- Validation of an LES
- Applications
- Challenges
- Conclusions



- Motivation:
- Simulation methodologies
- **Governing equations**
 - *Filtering*
 - *Filtered Navier-Stokes equations*
- Numerical methods
- Subfilter-scale modelling
- Validation of an LES
- Applications
- Challenges
- Conclusions



- Conservation of mass, momentum and energy

$$\frac{\partial \rho}{\partial t} + \frac{\partial}{\partial x_j} (\rho u_j) = 0$$

$$\frac{\partial \rho u_i}{\partial t} + \frac{\partial}{\partial x_j} (\rho u_i u_j) = -\frac{\partial p}{\partial x_i} + \frac{\partial}{\partial x_j} \left[\mu \left(2S_{ij} - \frac{2}{3} \delta_{ij} S_{kk} \right) \right]$$

$$\frac{\partial e}{\partial t} + \frac{\partial}{\partial x_j} (u_j e) = \frac{\partial Q}{\partial t} - \frac{\partial}{\partial x_j} \left(\kappa \frac{\partial T}{\partial x_j} \right),$$

- Where:

$$S_{ij} = \frac{1}{2} \left(\frac{\partial u_i}{\partial x_j} + \frac{\partial u_j}{\partial x_i} \right) \quad \text{Strain-rate tensor}$$

$$e = c_v T + \rho u_i u_i \quad \text{Total energy}$$

$$\kappa = \alpha / \rho c_p \quad \text{Diffusivity}$$

CONSERVATION EQUATIONS (INCOMPRESSIBLE FLOW)

- If $\rho = \text{constant}$

$$\frac{\partial u_j}{\partial x_j} = 0$$

$$\frac{\partial u_i}{\partial t} + \frac{\partial}{\partial x_j} u_i u_j = -\frac{1}{\rho} \frac{\partial p}{\partial x_i} + \nu \nabla^2 u_i$$

$$\frac{\partial T}{\partial t} + \frac{\partial}{\partial x_j} u_j T = \kappa \nabla^2 T$$



- Velocity decomposed into large-scale (resolved) and subfilter-scale (unresolved) parts.
 - *The large scales (\sim **Integral scale, L**), which depend on the boundary conditions (i.e., are flow dependent) are computed.*
 - *The small scales, which are more universal (less dependent on boundary conditions) are modelled.*
 - *Large scales contribute most of the Reynolds stresses.*



- The large and small scales are separated by the filtering operation

$$\bar{f}(\mathbf{x}) = \int_D f(\mathbf{x}') G(\mathbf{x}, \mathbf{x}', \bar{\Delta}) d\mathbf{x}'$$

- Filtering is a local spatial averaging over the filter width Δ that smooths out fluctuations whose scale is $>\Delta$.
- Filtering in general is **not** a Reynolds operator:

$$\overline{f'} \neq 0; \quad \overline{\bar{f}} \neq \bar{f}$$



- The filtering operation can be applied to the NS equations

$$\int_D [\text{Navier-Stokes Eqns}] G(\mathbf{x}, \mathbf{x}', \bar{\Delta}) d\mathbf{x}'$$

to yield the filtered NS equations

$$\frac{\partial \bar{u}_i}{\partial x_i} = 0$$

$$\frac{\partial \bar{u}_i}{\partial t} + \frac{\partial}{\partial x_j} (\bar{u}_i \bar{u}_j) = -\frac{1}{\rho} \frac{\partial \bar{p}}{\partial x_i} - \frac{\partial \tau_{ij}}{\partial x_j} + \nu \frac{\partial^2 \bar{u}_i}{\partial x_j \partial x_j}$$

$$\frac{\partial \bar{T}}{\partial t} + \frac{\partial}{\partial x_j} \bar{u}_j \bar{T} = -\frac{\partial Q_j}{\partial x_j} + \kappa \nabla^2 \bar{T}$$

- Unresolved stresses and heat flux appear:

$$\tau_{ij} = \overline{u_i u_j} - \bar{u}_i \bar{u}_j$$

$$Q_j = \overline{u_j T} - \bar{u}_j \bar{T}$$



- The unresolved stresses are known as **Subgrid-Scale (SGS) stresses** or (better) as **SubFilter-Scale (SFS) stresses**.
- They must be expressed in terms of filtered variables (SFS model)

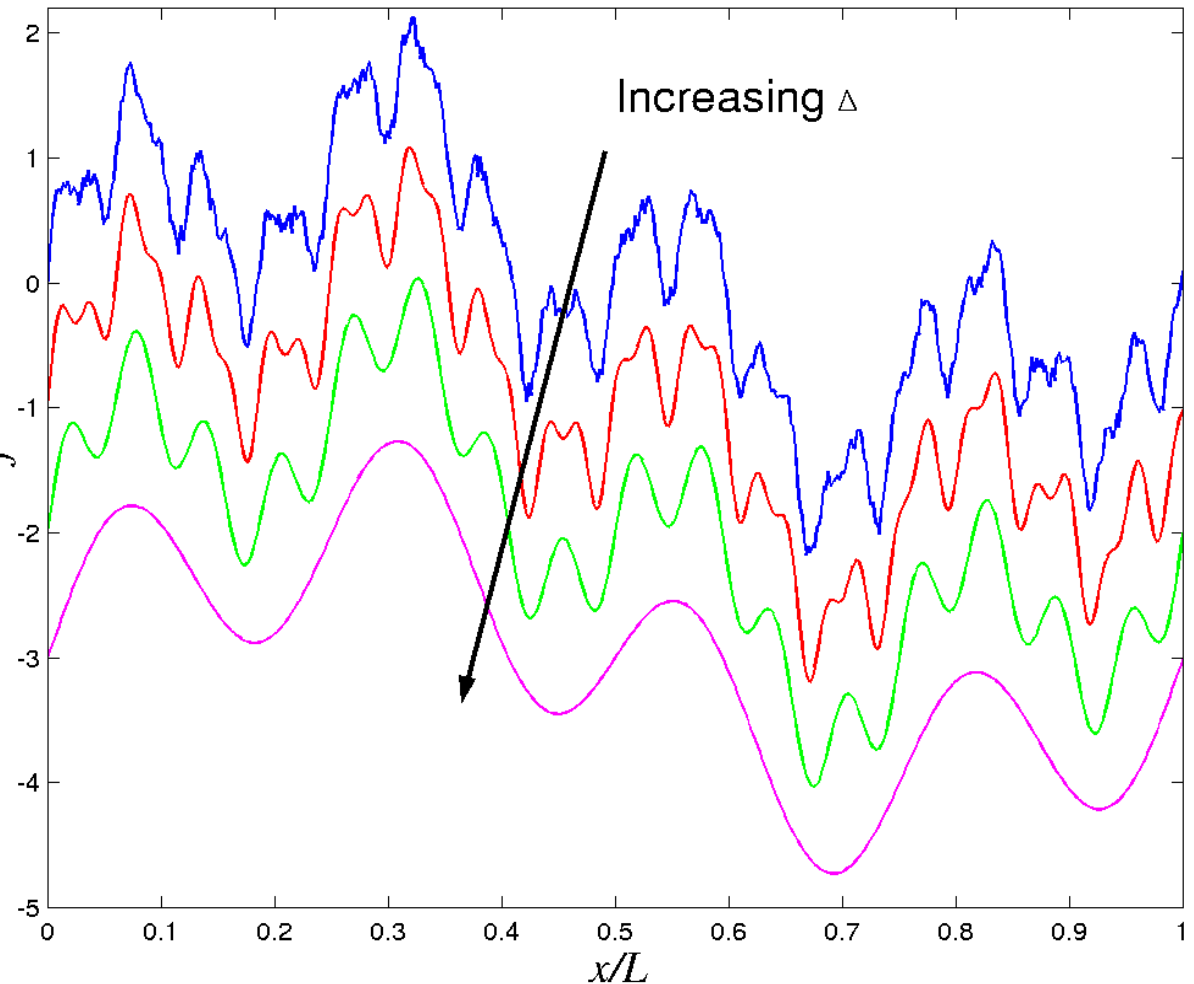
$$\tau_{ij} = f(\bar{u}_i, \bar{S}_{ij}, \dots)$$

- *The small scales, which are more universal (less dependent on boundary conditions) are modelled.*
- The large scales, which depend on the boundary conditions(i.e., are flow dependent) are computed.



THE FILTERING OPERATION

- Filtering is a local spatial averaging over the filter width Δ .
- Increasing Δ removes more and more scales from the velocity field \Rightarrow the contribution of τ_{ij} increases.





- Fundamental assumption in LES:

The energy-carrying eddies are resolved,
only the small scales are modelled

- Implication:

The filter-width must be smaller
than the local integral scale, L

- Practice:

The filter width Δ is proportional
to the grid size h



- If Δ is proportional to h :
 - *The ratio between filter-width and integral scale varies*
 - ⇒ Suboptimal grids (that are not refined when the integral scale decreases) may have unexpectedly large errors
 - *Rapid variations of the grid size are reflected in the eddy viscosity*
 - ⇒ Unphysical discontinuities in the SFS contribution to the transport can occur.
 - ⇒ Numerical and commutation errors may be significant.
 - *Grid convergence studies are difficult.*



- Motivation:
- Simulation methodologies
- Governing equations
- **Boundary conditions**
- Subfilter-scale modelling
- Validation of an LES
- Applications
- Challenges
- Conclusions